

Amplifier Piezoelectric Actuator APA300ML - Test Bench

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The goal of this test bench is to extract all the important parameters of the Amplified Piezoelectric Actuator APA300ML.

This include:

- Stroke
- Stiffness
- Hysteresis
- Gain from the applied voltage V_a to the generated Force F_a
- Gain from the sensor stack strain δL to the generated voltage V_s

- Dynamical behavior

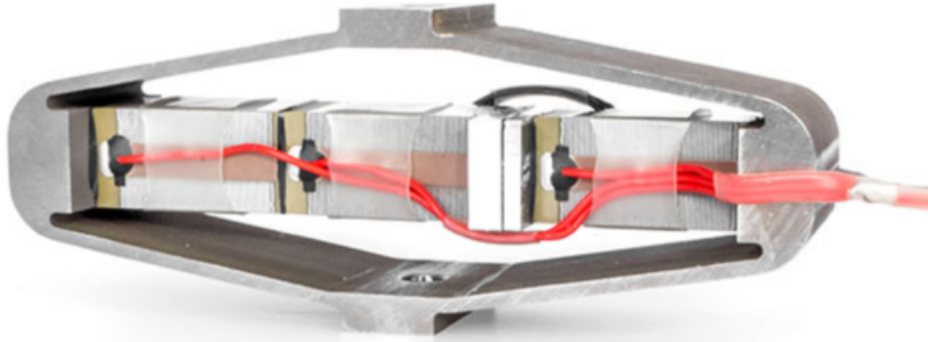


Figure 0.1: Picture of the APA300ML

1 Model of an Amplified Piezoelectric Actuator and Sensor

Consider a schematic of the Amplified Piezoelectric Actuator in Figure 1.1.

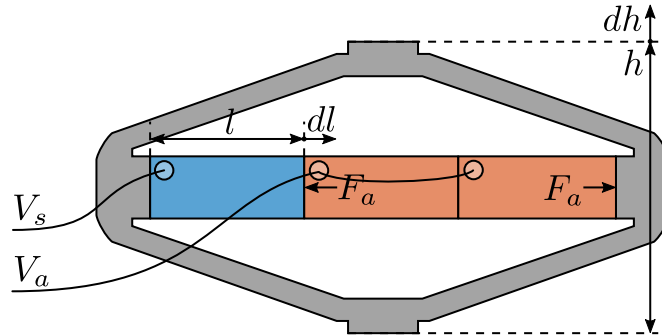


Figure 1.1: Amplified Piezoelectric Actuator Schematic

A voltage V_a applied to the actuator stacks will induce an actuator force F_a :

$$F_a = g_a \cdot V_a \quad (1.1)$$

A change of length dl of the sensor stack will induce a voltage V_s :

$$V_s = g_s \cdot dl \quad (1.2)$$

We wish here to experimental measure g_a and g_s .

The block-diagram model of the piezoelectric actuator is then as shown in Figure 1.2.

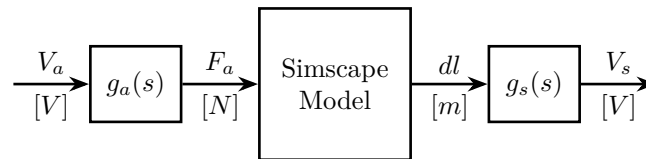


Figure 1.2: Model of the APA with Simscape/Simulink

2 First Basic Measurements

- Section 2.1:
- Section 2.2:
- Section 2.3:
- Section 2.4:

2.1 Geometrical Measurements

The received APA are shown in Figure 2.1.

2.1.1 Measurement Setup

The flatness corresponding to the two interface planes are measured as shown in Figure 2.2.

2.1.2 Measurement Results

The height (Z) measurements at the 8 locations (4 points by plane) are defined below.

```
Matlab
apa1 = 1e-6*[0, -0.5, 3.5, 3.5, 42, 45.5, 52.5, 46];
apa2 = 1e-6*[0, -2.5, -3, 0, -1.5, 1, -2, -4];
apa3 = 1e-6*[0, -1.5, 15, 17.5, 6.5, 6.5, 21, 23];
apa4 = 1e-6*[0, 6.5, 14.5, 9, 16, 22, 29.5, 21];
apa5 = 1e-6*[0, -12.5, 16.5, 28.5, -43, -52, -22.5, -13.5];
apa6 = 1e-6*[0, -8, -2, 5, -57.5, -62, -55.5, -52.5];
apa7 = 1e-6*[0, 19.5, -8, -29.5, 75, 97.5, 70, 48];
apa7b = 1e-6*[0, 9, -18.5, -30, 31, 46.5, 16.5, 7.5];
apa = {apa1, apa2, apa3, apa4, apa5, apa6, apa7b};
```

The X/Y Positions of the 8 measurement points are defined below.

```
Matlab
W = 20e-3; % Width [m]
L = 61e-3; % Length [m]
d = 1e-3; % Distance from border [m]
l = 15.5e-3; % [m]

pos = [[-L/2 + d; W/2 - d], [-L/2 + l - d; W/2 - d], [-L/2 + l - d; -W/2 + d], [-L/2 + d; -W/2 + d], [L/2 - l + d; W/2 - d],
↪ [L/2 - d; W/2 - d], [L/2 - d; -W/2 + d], [L/2 - l + d; -W/2 + d]];
```



Figure 2.1: Received APA

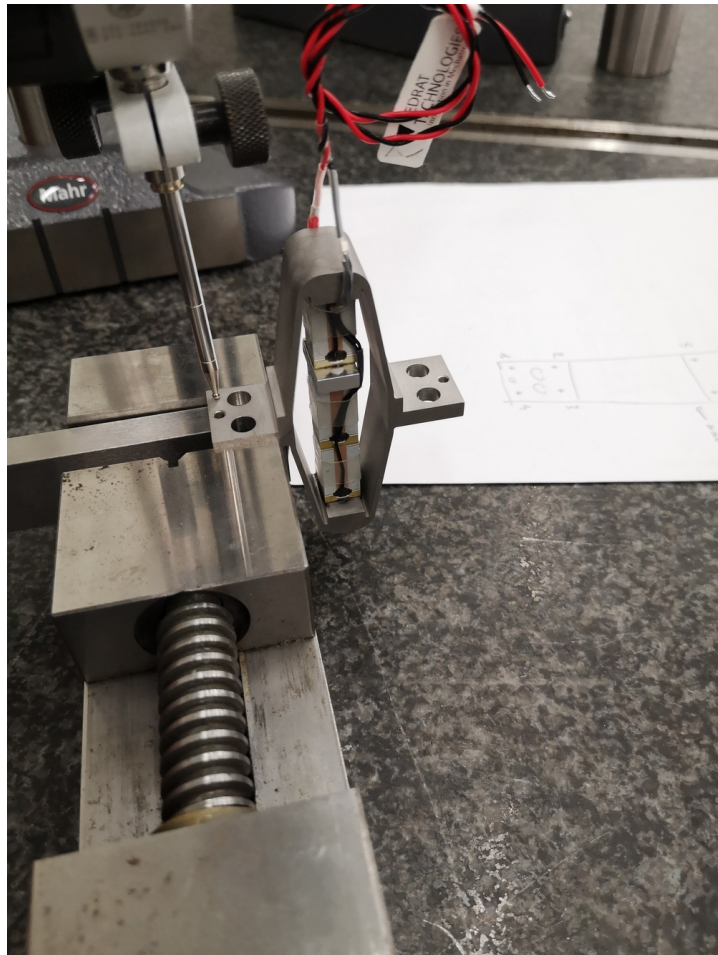


Figure 2.2: Measurement Setup

Finally, the flatness is estimated by fitting a plane through the 8 points using the `fminsearch` command.

```

Matlab
apa_d = zeros(1, 7);
for i = 1:7
    fun = @(x)max(abs((pos; apa{i})-[0;0;x(1)]'*([x(2:3);1])/norm([x(2:3);1])));
    x0 = [0;0;0];
    [x, min_d] = fminsearch(fun,x0);
    apa_d(i) = min_d;
end

```

The obtained flatness are shown in Table 2.1.

Table 2.1: Estimated flatness

	Flatness [μm]
APA 1	8.9
APA 2	3.1
APA 3	9.1
APA 4	3.0
APA 5	1.9
APA 6	7.1
APA 7	18.7

2.2 Electrical Measurements

Note

The capacitance of the stacks is measure with the [LCR-800 Meter \(doc\)](#)

The excitation frequency is set to be 1kHz.

Table 2.2: Capacitance measured with the LCR meter. The excitation signal is a sinus at 1kHz

	Sensor Stack	Actuator Stacks
APA 1	5.10	10.03
APA 2	4.99	9.85
APA 3	1.72	5.18
APA 4	4.94	9.82
APA 5	4.90	9.66
APA 6	4.99	9.91
APA 7	4.85	9.85

Warning

There is clearly a problem with APA300ML number 3

The APA number 3 has ben sent back to Cedrat, and a new APA300ML has been shipped back.



Figure 2.3: LCR Meter used for the measurements

2.3 Stroke measurement

We here wish to estimate the stroke of the APA.

To do so, one side of the APA is fixed, and a displacement probe is located on the other side as shown in Figure 2.4.

Then, a voltage is applied on either one or two stacks using a DAC and a voltage amplifier.

Note

Here are the documentation of the equipment used for this test bench:

- **Voltage Amplifier:** [PD200](#) with a gain of 20
- **16bits DAC:** [IO313 Speedgoat card](#)
- **Displacement Probe:** [Millimar C1216 electronics](#) and [Millimar 1318 probe](#)

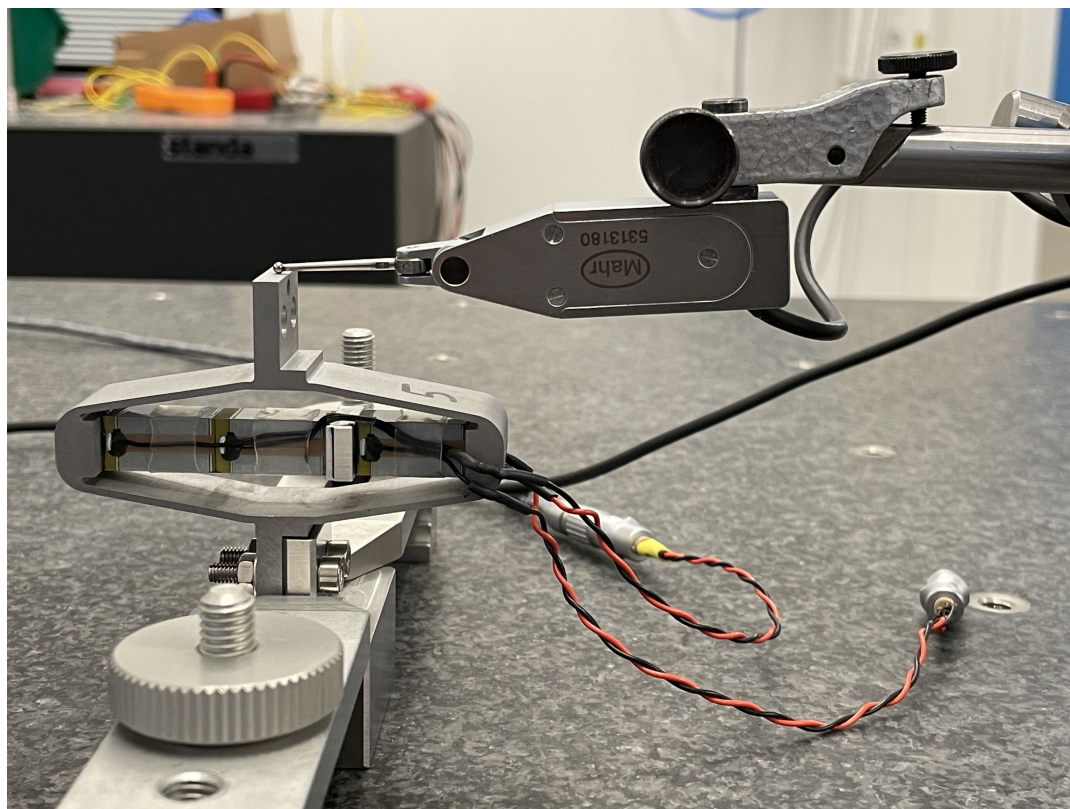


Figure 2.4: Bench to measured the APA stroke

2.3.1 Voltage applied on one stack

Let's first look at the relation between the voltage applied to **one** stack to the displacement of the APA as measured by the displacement probe.

The applied voltage is shown in Figure 2.5.

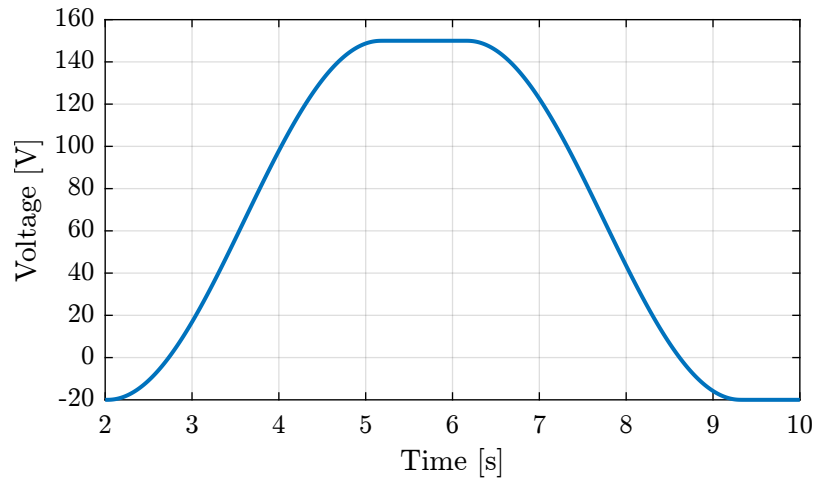


Figure 2.5: Applied voltage as a function of time

The obtained displacement is shown in Figure 2.6. The displacement is set to zero at initial time when the voltage applied is -20V.

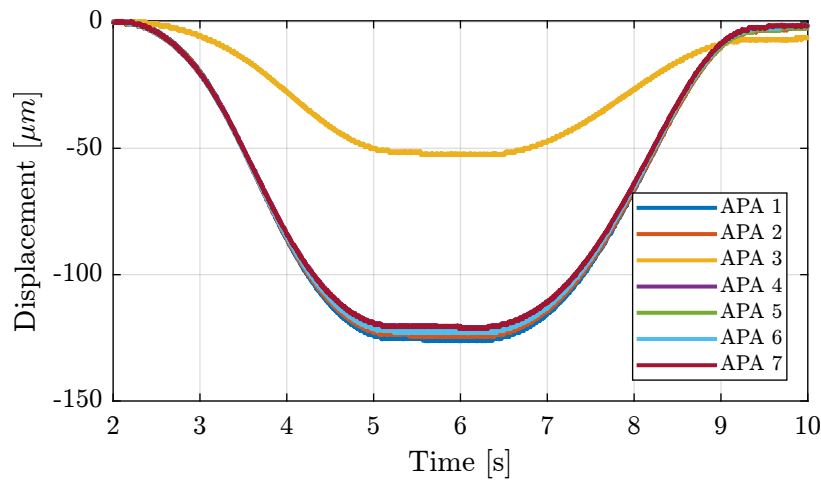


Figure 2.6: Displacement as a function of time for all the APA300ML

Finally, the displacement is shown as a function of the applied voltage in Figure 2.7. We can clearly see that there is a problem with the APA 3. Also, there is a large hysteresis.

Important

We can clearly see from Figure 2.7 that there is a problem with the APA number 3.

2.3.2 Voltage applied on two stacks

Now look at the relation between the voltage applied to the **two** other stacks to the displacement of the APA as measured by the displacement probe.

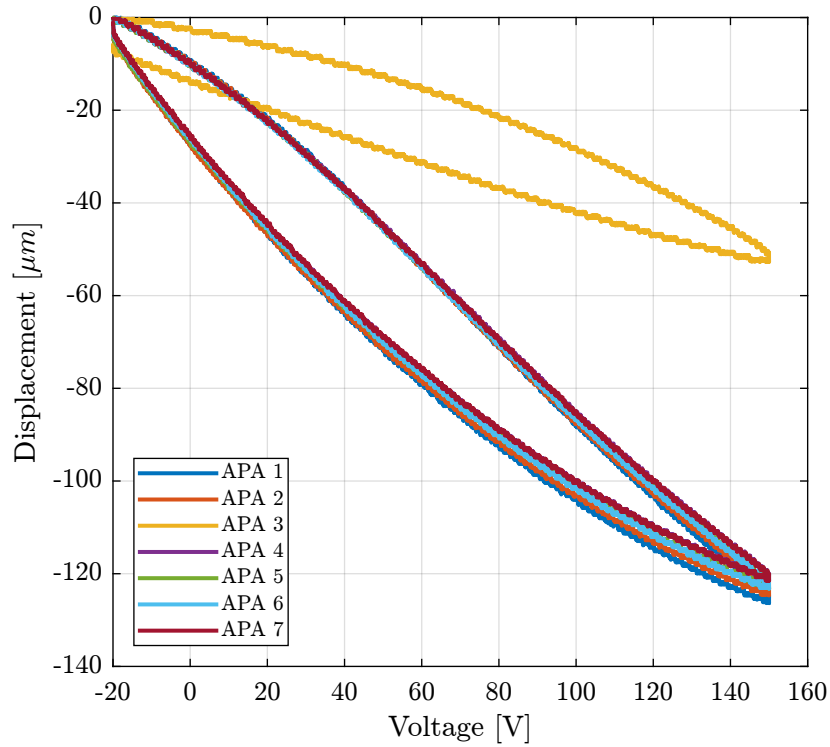


Figure 2.7: Displacement as a function of the applied voltage

The obtained displacement is shown in Figure 2.8. The displacement is set to zero at initial time when the voltage applied is -20V.

Finally, the displacement is shown as a function of the applied voltage in Figure 2.9. We can clearly see that there is a problem with the APA 3. Also, there is a large hysteresis.

2.3.3 Voltage applied on all three stacks

Finally, we can combine the two measurements to estimate the relation between the displacement and the voltage applied to the **three** stacks (Figure 2.10).

The obtained maximum stroke for all the APA are summarized in Table 2.3.

Table 2.3: Measured maximum stroke

	Stroke [μm]
APA 1	373.2
APA 2	365.5
APA 3	181.7
APA 4	359.7
APA 5	361.5
APA 6	363.9
APA 7	358.4

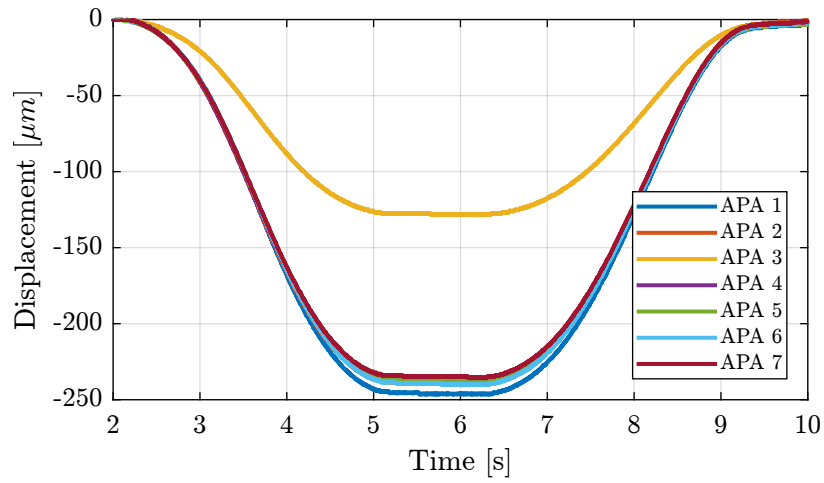


Figure 2.8: Displacement as a function of time for all the APA300ML

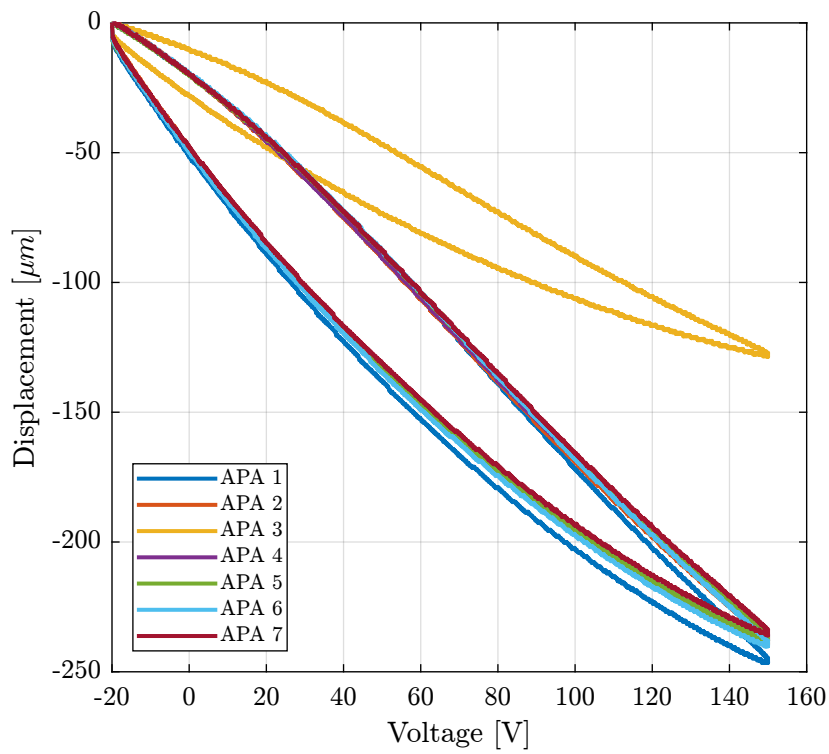


Figure 2.9: Displacement as a function of the applied voltage

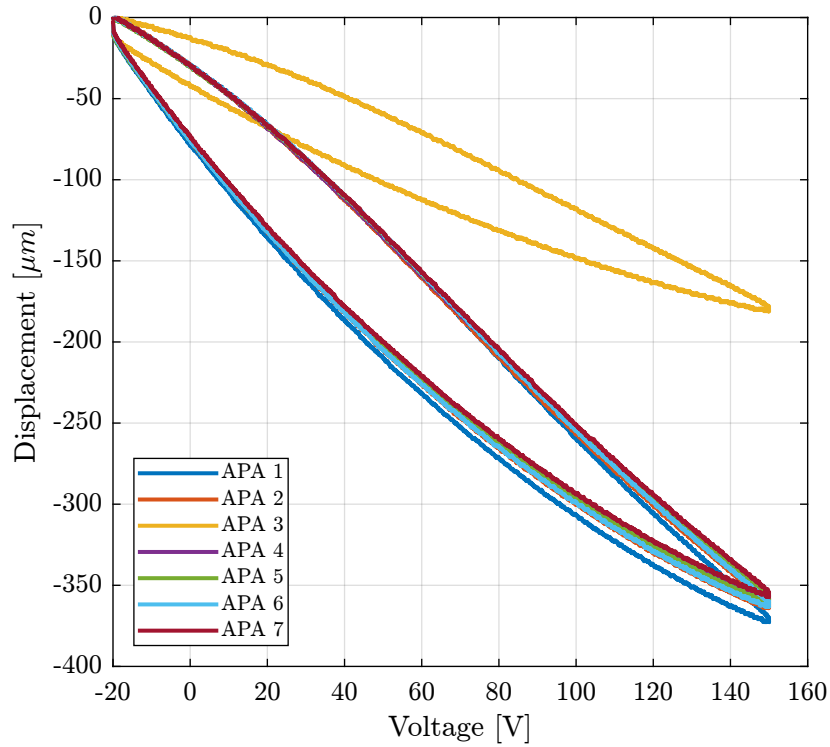


Figure 2.10: Displacement as a function of the applied voltage

2.4 Spurious resonances

2.4.1 Introduction

Three main resonances are foreseen to be problematic for the control of the APA300ML:

- Mode in X-bending at 189Hz (Figure 2.11)
- Mode in Y-bending at 285Hz (Figure 2.12)
- Mode in Z-torsion at 400Hz (Figure 2.13)

These modes are present when flexible joints are fixed to the ends of the APA300ML.

In this section, we try to find the resonance frequency of these modes when one end of the APA is fixed and the other is free.

2.4.2 Setup

The measurement setup is shown in Figure 2.14. A Laser vibrometer is measuring the difference of motion of two points. The APA is excited with an instrumented hammer and the transfer function from the hammer to the measured rotation is computed.

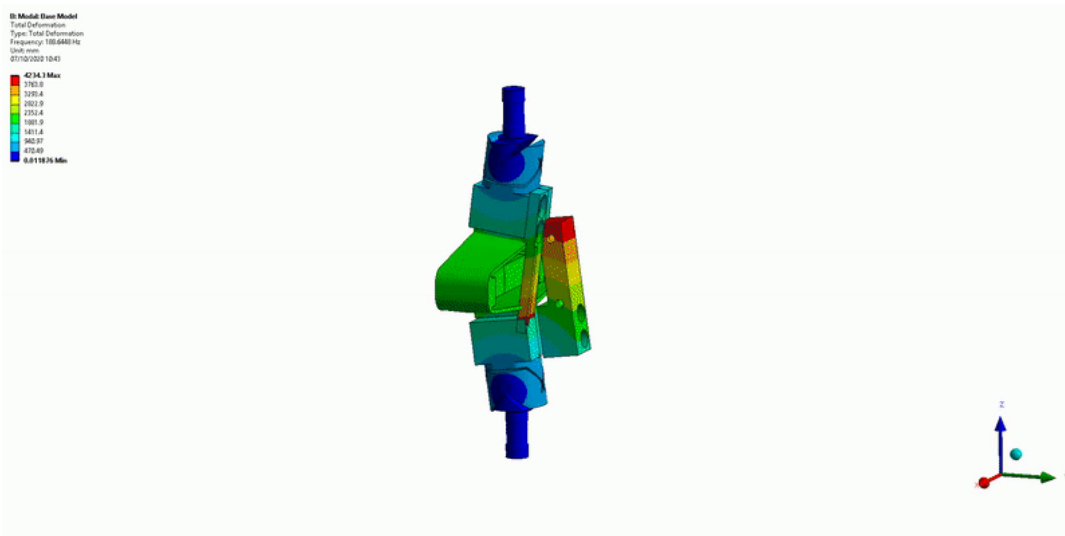


Figure 2.11: X-bending mode (189Hz)

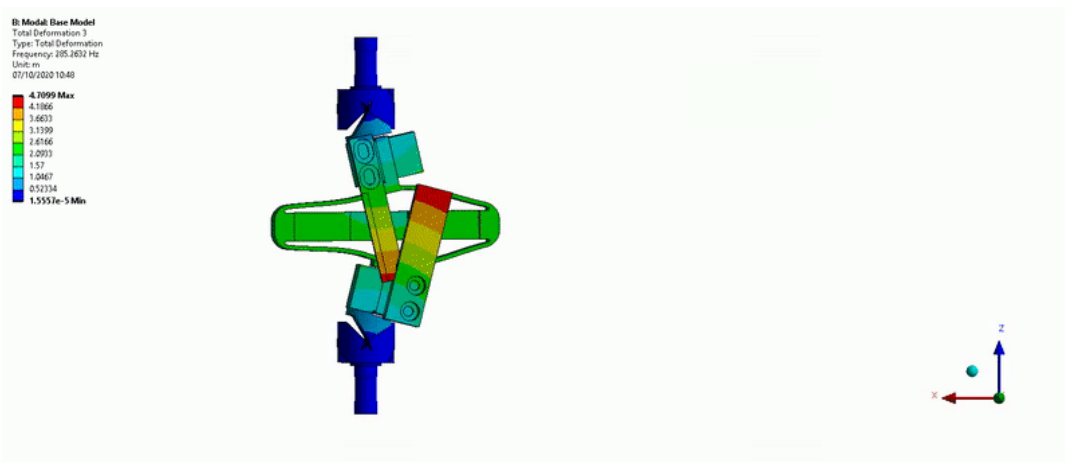


Figure 2.12: Y-bending mode (285Hz)

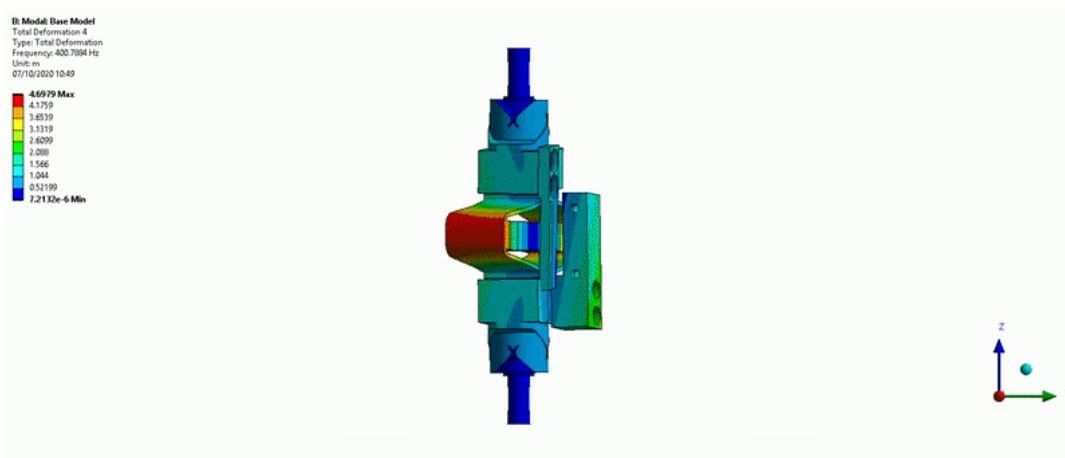


Figure 2.13: Z-torsion mode (400Hz)

Note

- Laser Doppler Vibrometer Polytec OFV512
- Instrumented hammer

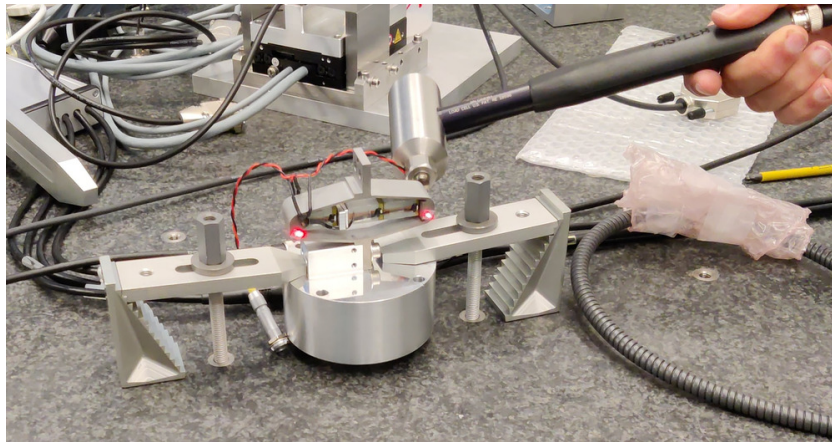


Figure 2.14: Measurement setup with a Laser Doppler Vibrometer and one instrumental hammer

2.4.3 Bending - X

The setup to measure the X-bending motion is shown in Figure 2.15. The APA is excited with an instrumented hammer having a solid metallic tip. The impact point is on the back-side of the APA aligned with the top measurement point.

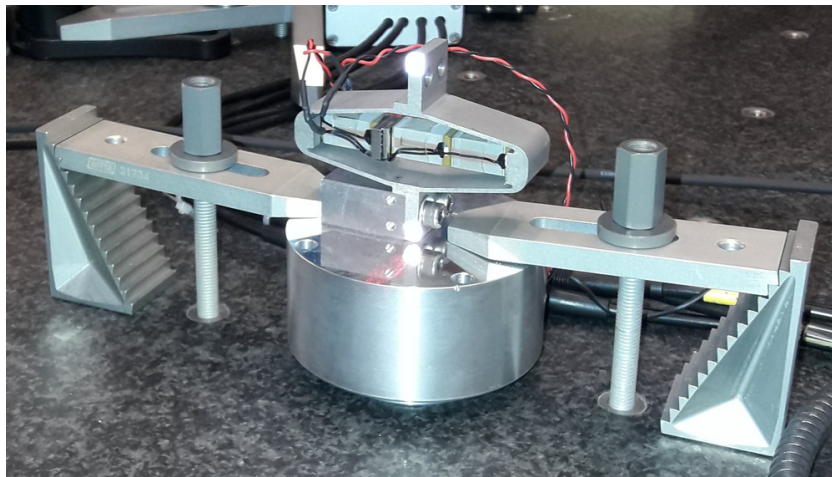


Figure 2.15: X-Bending measurement setup

The data is loaded.

```
bending_X = load('apa300ml_bending_X_top.mat');
```

Matlab

The config for `tfestimate` is performed:

```
Matlab
Ts = bending_X.Track1_X_Resolution; % Sampling frequency [Hz]
win = hann(ceil(1/Ts));
```

The transfer function from the input force to the output “rotation” (difference between the two measured distances).

```
Matlab
[G_bending_X, f] = tfestimate(bending_X.Track1, bending_X.Track2, win, [], [], 1/Ts);
```

The result is shown in Figure 2.16.

The can clearly observe a nice peak at 280Hz, and then peaks at the odd “harmonics” (third “harmonic” at 840Hz, and fifth “harmonic” at 1400Hz).

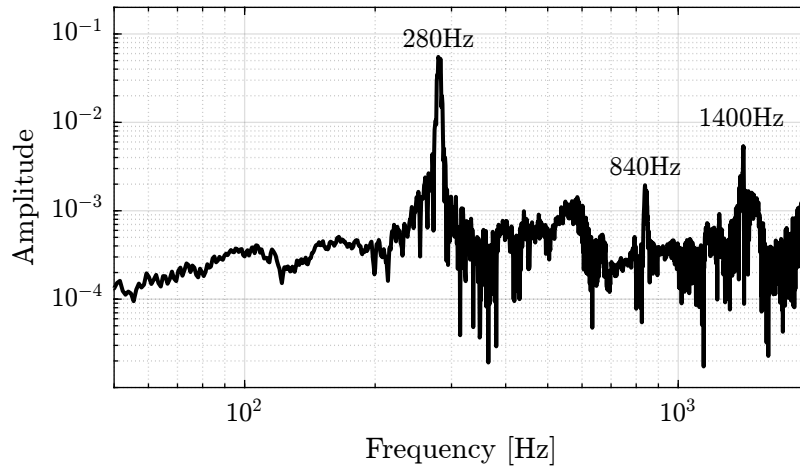


Figure 2.16: Obtained FRF for the X-bending

2.4.4 Bending - Y

The setup to measure the Y-bending is shown in Figure 2.17.

The impact point of the instrumented hammer is located on the back surface of the top interface (on the back of the 2 measurements points).

The data is loaded, and the transfer function from the force to the measured rotation is computed.

```
Matlab
bending_Y = load('apa300ml_bending_Y_top.mat');
[G_bending_Y, ~] = tfestimate(bending_Y.Track1, bending_Y.Track2, win, [], [], 1/Ts);
```

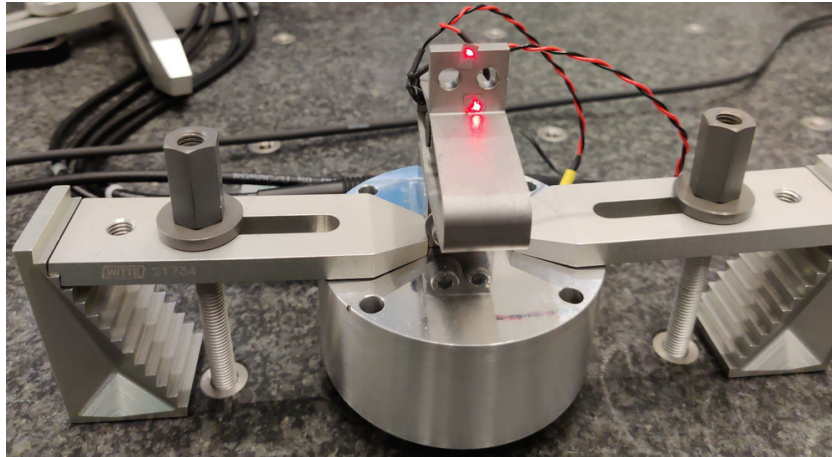


Figure 2.17: Y-Bending measurement setup

The results are shown in Figure 2.18. The main resonance is at 412Hz, and we also see the third “harmonic” at 1220Hz.

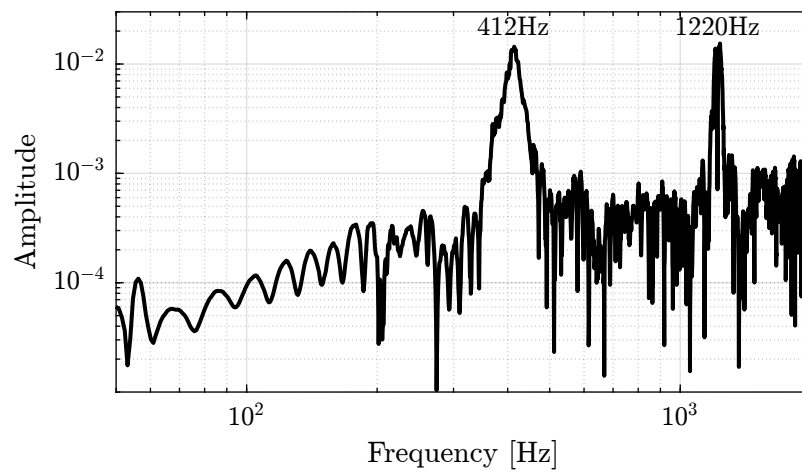


Figure 2.18: Obtained FRF for the Y-bending

2.4.5 Torsion - Z

Finally, we measure the Z-torsion resonance as shown in Figure 2.19.

The excitation is shown on the other side of the APA, on the side to excite the torsion motion.

The data is loaded, and the transfer function computed.

```

Matlab
torsion = load('apa300ml_torsion_left.mat');
[G_torsion, ~] = tfestimate(torsion.Track1, torsion.Track2, win, [], [], 1/Ts);

```

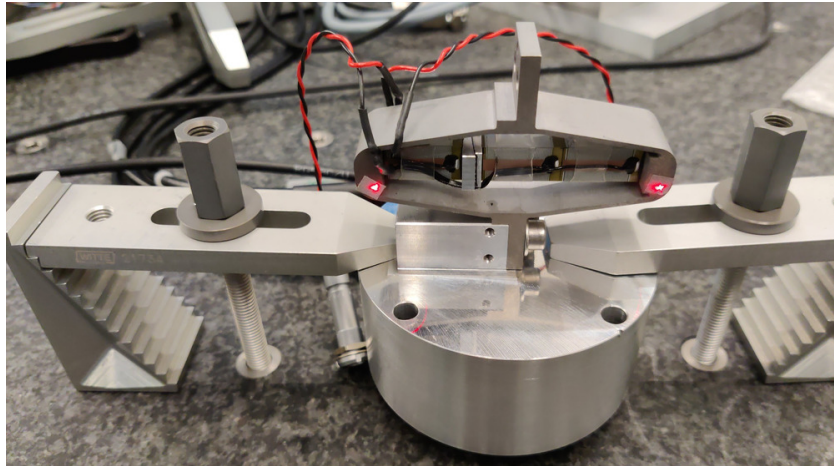


Figure 2.19: Z-Torsion measurement setup

The results are shown in Figure 2.20. We observe a first peak at 267Hz, which corresponds to the X-bending mode that was measured at 280Hz. And then a second peak at 415Hz, which corresponds to the X-bending mode that was measured at 412Hz. The mode in pure torsion is probably at higher frequency (peak around 1kHz?).

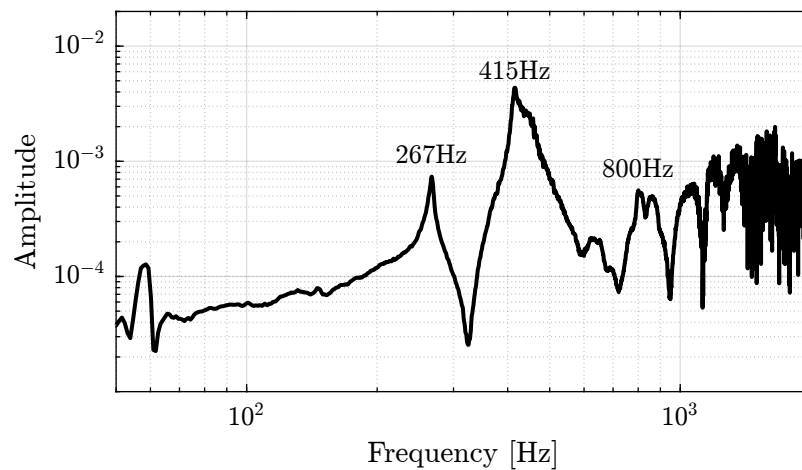


Figure 2.20: Obtained FRF for the Z-torsion

In order to verify that, the APA is excited on the top part such that the torsion mode should not be excited.

```

Matlab
torsion = load('apa300ml_torsion_top.mat');
[G_torsion_top, ~] = tfestimate(torsion.Track1, torsion.Track2, win, [], [], 1/Ts);

```

The two FRF are compared in Figure 2.21. It is clear that the first two modes does not correspond to the torsional mode. Maybe the resonance at 800Hz, or even higher resonances. It is difficult to conclude here.

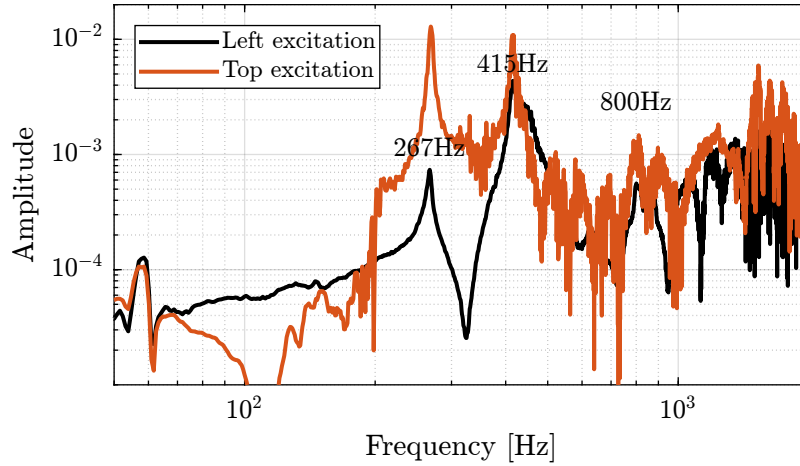


Figure 2.21: Obtained FRF for the Z-torsion

2.4.6 Compare

The three measurements are shown in Figure 2.22.

2.4.7 Conclusion

When two flexible joints are fixed at each ends of the APA, the APA is mostly in a free/free condition in terms of bending/torsion (the bending/torsional stiffness of the joints being very small).

In the current tests, the APA are in a fixed/free condition. Therefore, it is quite obvious that we measured higher resonance frequencies than what is foreseen for the struts. It is however quite interesting that there is a factor $\approx \sqrt{2}$ between the two (increased of the stiffness by a factor 2?).

Table 2.4: Measured frequency of the modes

Mode	Strut Mode	Measured Frequency
X-Bending	189Hz	280Hz
Y-Bending	285Hz	410Hz
Z-Torsion	400Hz	?

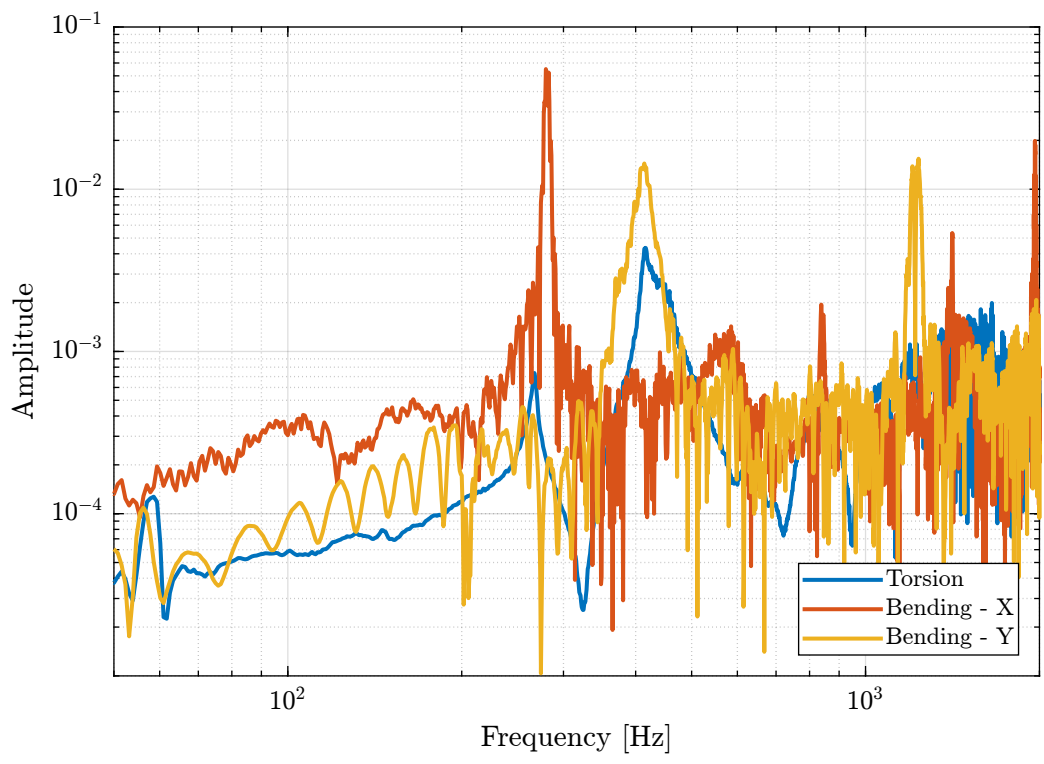


Figure 2.22: Obtained FRF - Comparison

3 Dynamical measurements - APA

In this section, a measurement test bench is used to identify the dynamics of the APA.

The bench is shown in Figure 3.1, and a zoom picture on the APA and encoder is shown in Figure 3.2.

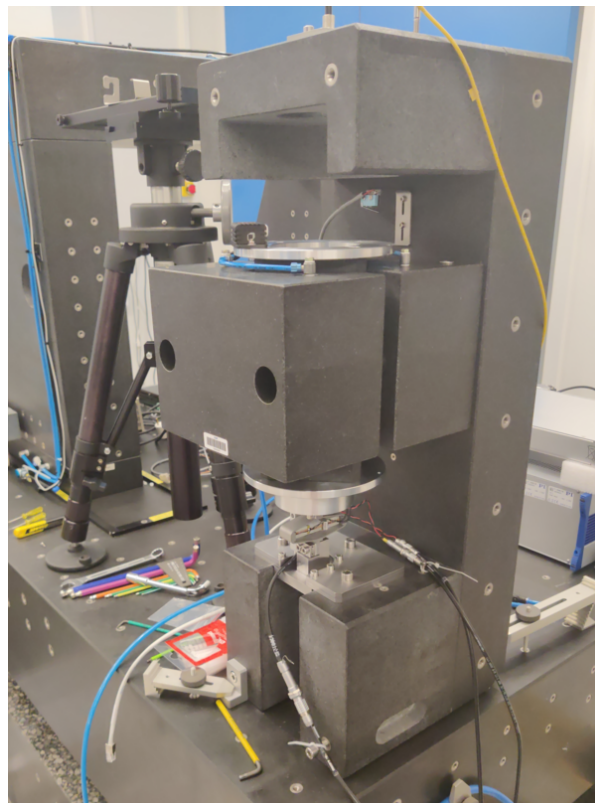


Figure 3.1: Picture of the test bench

Note

Here are the documentation of the equipment used for this test bench:

- Voltage Amplifier: [PD200](#)
- Amplified Piezoelectric Actuator: [APA300ML](#)
- DAC/ADC: Speedgoat [IO313](#)
- Encoder: [Renishaw Vionic](#) and used [Ruler](#)
- Interferometer: [Attocube IDS3010](#)

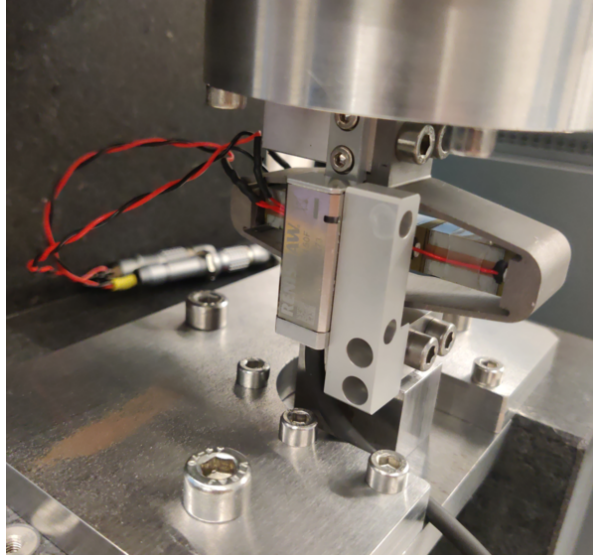


Figure 3.2: Zoom on the APA with the encoder

The bench is schematically shown in Figure 3.3 and the signal used are summarized in Table 3.1.

Table 3.1: Variables used during the measurements

Variable	Description	Unit	Hardware
Va	Output DAC voltage	[V]	DAC - Ch. 1 => PD200 => APA
Vs	Measured stack voltage (ADC)	[V]	APA => ADC - Ch. 1
de	Encoder Measurement	[m]	PEPU Ch. 1 - IO318(1) - Ch. 1
da	Attocube Measurement	[m]	PEPU Ch. 2 - IO318(1) - Ch. 2
t	Time	[s]	

This section is structured as follows:

- Section 3.1: the Speedgoat setup is described (excitation signals, saved signals, etc.)
- Section 3.2: the measurements are first performed on one APA.
- Section 3.3: the same measurements are performed on all the APA and are compared.

3.1 Speedgoat Setup

3.1.1 frf_setup.m - Measurement Setup

First is defined the sampling frequency:

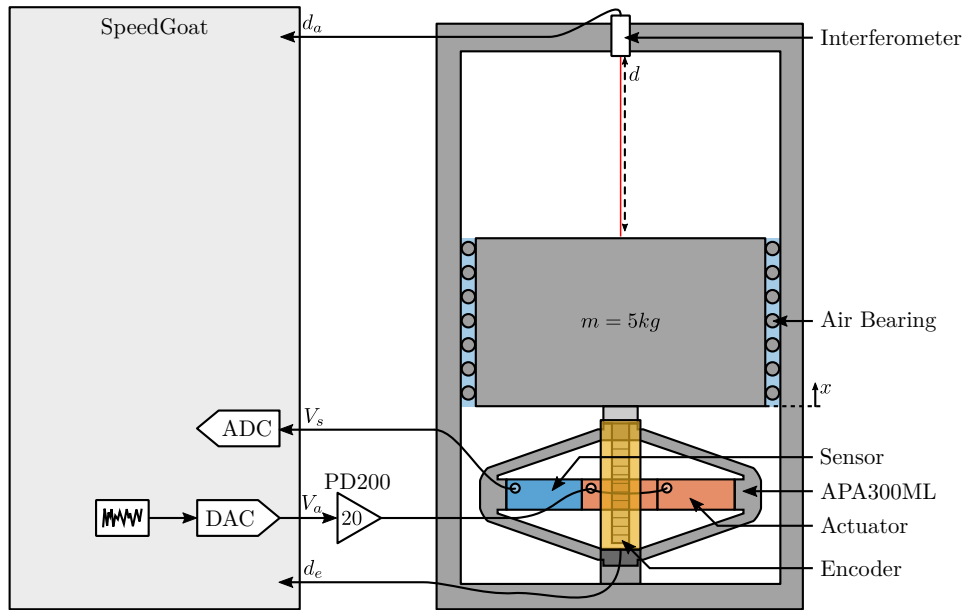


Figure 3.3: Schematic of the Test Bench

```

Matlab
%% Simulation configuration
Fs = 10e3; % Sampling Frequency [Hz]
Ts = 1/Fs; % Sampling Time [s]

```

```

Matlab
%% Data record configuration
Trec_start = 5; % Start time for Recording [s]
Trec_dur = 100; % Recording Duration [s]

```

```

Matlab
Tsim = 2*Trec_start + Trec_dur; % Simulation Time [s]

```

A white noise excitation signal can be very useful in order to obtain a first idea of the plant FRF. The gain can be gradually increased until satisfactory output is obtained.

```

Matlab
%% Shaped Noise
V_noise = generateShapedNoise('Ts', 1/Fs, ...
    'V_mean', 3.25, ...
    't_start', Trec_start, ...
    'exc_duration', Trec_dur, ...
    'smooth_ends', true, ...
    'V_exc', 0.05/(1 + s/2/pi/10));

```

The maximum excitation voltage at resonance is 9Vrms, therefore corresponding to 0.6V of output DAC voltage.

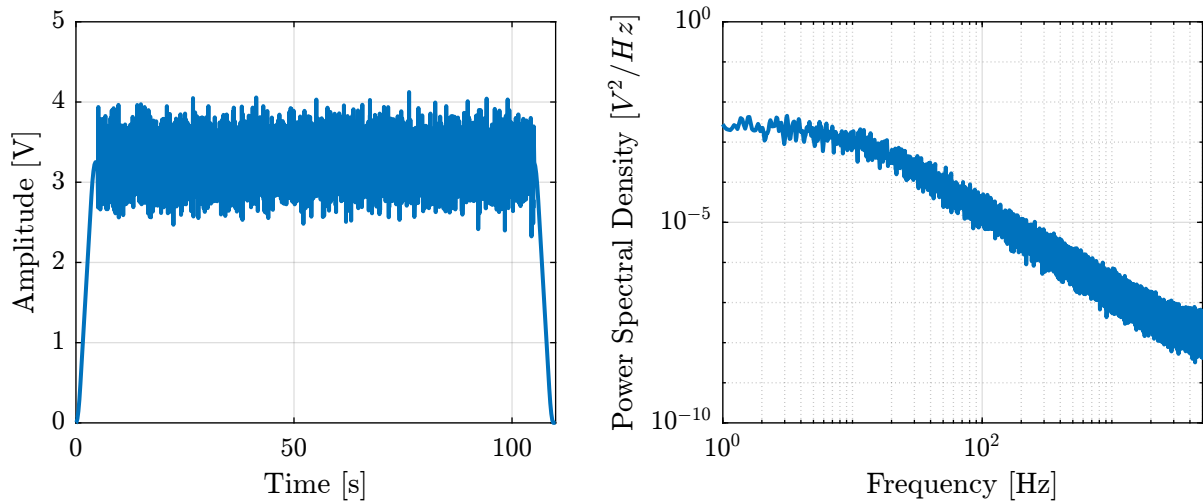


Figure 3.4: Example of Shaped noise excitation signal

```

%% Sweep Sine
gc = 0.1;
xi = 0.5;
wn = 2*pi*94.3;

% Notch filter at the resonance of the APA
G_sweep = 0.2*(s^2 + 2*gc*xi*wn*s + wn^2)/(s^2 + 2*xi*wn*s + wn^2);

V_sweep = generateSweepExc('Ts', Ts, ...
    'f_start', 10, ...
    'f_end', 400, ...
    'V_mean', 3.25, ...
    't_start', Trec_start, ...
    'exc_duration', Trec_dur, ...
    'sweep_type', 'log', ...
    'V_exc', G_sweep*1/(1 + s/2/pi/500));

```

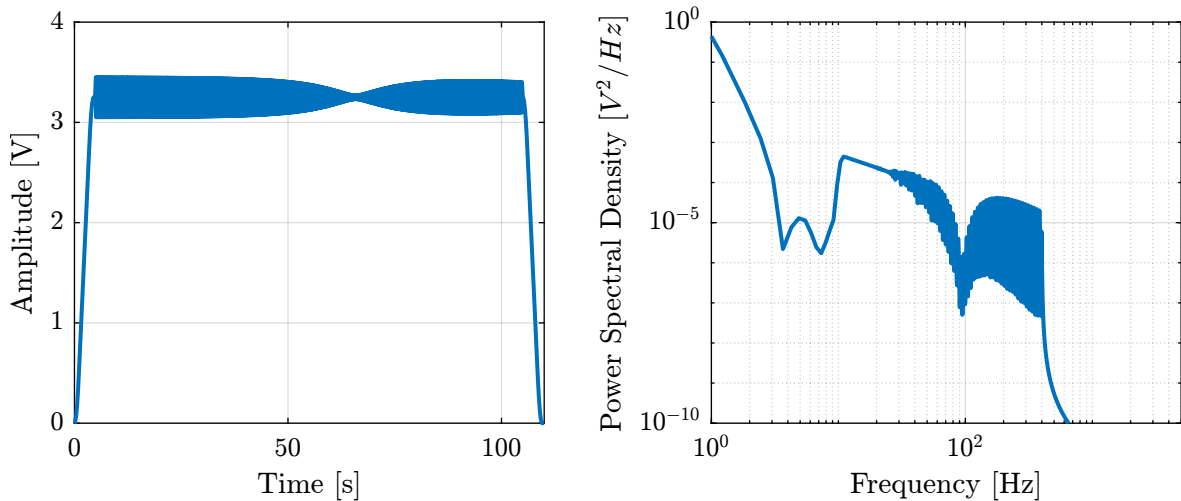


Figure 3.5: Example of Sweep Sin excitation signal

In order to better estimate the high frequency dynamics, a band-limited noise can be used (Figure 3.6).

The frequency content of the noise can be precisely controlled.

```

Matlab
%% High Frequency Shaped Noise
[b,a] = cheby1(10, 2, 2*pi*[300 2e3], 'bandpass', 's');
wL = 0.005*tf(b, a);

V_noise_hf = generateShapedNoise('Ts', 1/Fs, ...
    'V_mean', 3.25, ...
    't_start', Trec_start, ...
    'exc_duration', Trec_dur, ...
    'smooth_ends', true, ...
    'V_exc', wL);

```

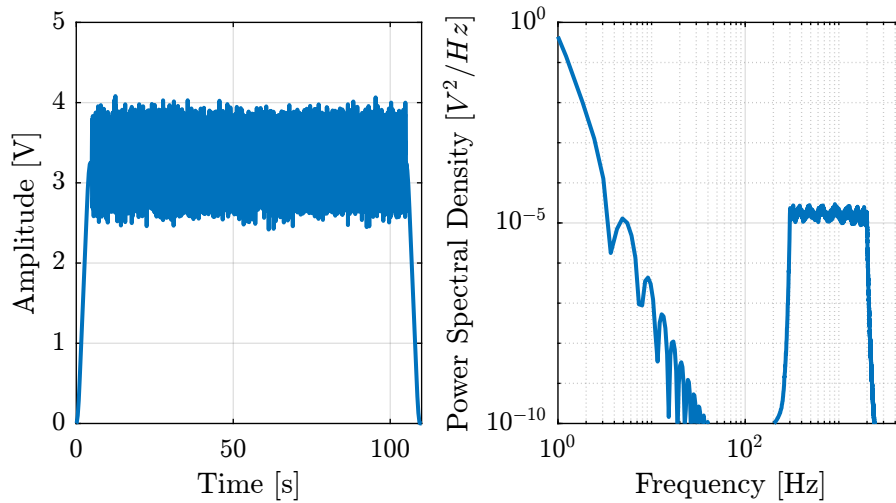


Figure 3.6: Example of band-limited noise excitation signal

Then a sinus excitation can be used to estimate the hysteresis.

```

Matlab
%% Sinus excitation with increasing amplitude
V_sin = generateSinIncreasingAmpl('Ts', 1/Fs, ...
    'V_mean', 3.25, ...
    'sin_ampls', [0.1, 0.2, 0.4, 1, 2, 4], ...
    'sin_period', 1, ...
    'sin_num', 5, ...
    't_start', Trec_start, ...
    'smooth_ends', true);

```

Then, we select the wanted excitation signal.

```

Matlab
%% Select the excitation signal
V_exc = timeseries(V_noise(2,:), V_noise(1,:));

```

```

Matlab
%% Save data that will be loaded in the Simulink file
save('./frf_data.mat', 'Fs', 'Ts', 'Tsim', 'Trec_start', 'Trec_dur', 'V_exc');

```

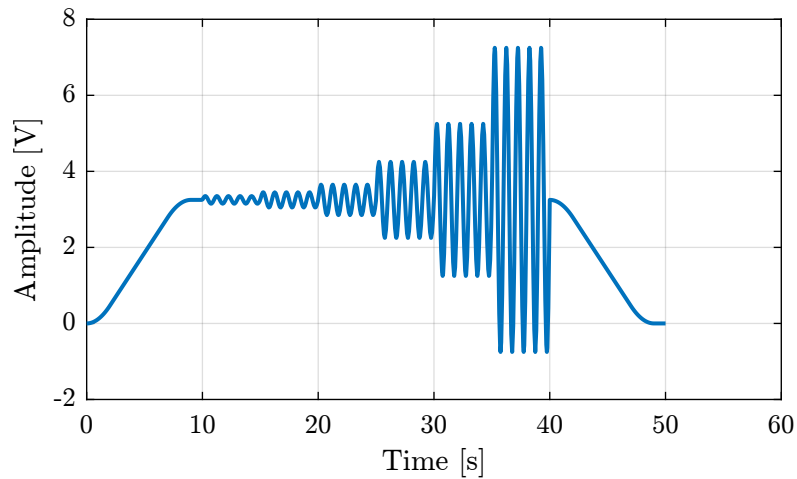


Figure 3.7: Example of Shaped noise excitation signal

3.1.2 frf_save.m - Save Data

First, we get data from the Speedgoat:

```
tg = slrt;
```

```
f = SimulinkRealTime.openFTP(tg);
mget(f, 'data/data.dat');
close(f);
```

And we load the data on the Workspace:

```
data = SimulinkRealTime.utils.getFileScopeData('data/data.dat').data;
da = data(:, 1); % Excitation Voltage (input of PD200) [V]
de = data(:, 2); % Measured voltage (force sensor) [V]
Vs = data(:, 3); % Measurement displacement (encoder) [m]
Va = data(:, 4); % Measurement displacement (attocube) [m]
t = data(:, end); % Time [s]
```

And we save this to a `mat` file:

```
apa_number = 1;
```

```
save(sprintf('mat/frf_data%i_huddle.mat', apa_number), 't', 'Va', 'Vs', 'de', 'da');
```

3.2 Measurements on APA 1

Measurements are first performed on only **one** APA. Once the measurement procedure is validated, it is performed on all the other APA.

3.2.1 Excitation Signal

For this first measurement, a basic logarithmic sweep is used between 10Hz and 2kHz.

The data are loaded.

```
Matlab
apa_sweep = load(sprintf('mat/frf_data%i_sweep.mat', 1), 't', 'Va', 'Vs', 'da', 'de');
```

The initial time is set to zero.

```
Matlab
%% Time vector
t = apa_sweep.t - apa_sweep.t(1) ; % Time vector [s]
```

The excitation signal is shown in Figure 3.8. It is a sweep sine from 10Hz up to 2kHz filtered with a notch centered with the main resonance of the system and a low pass filter.

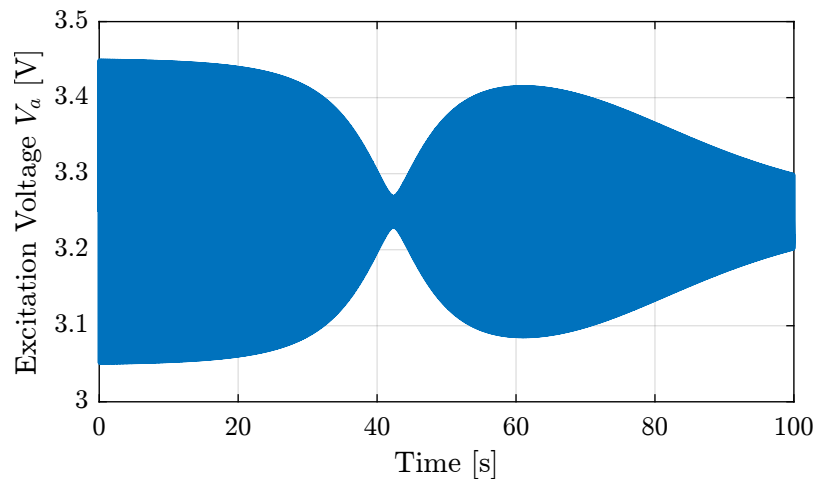


Figure 3.8: Excitation voltage

3.2.2 FRF Identification - Setup

Let's define the sampling time/frequency.

```
Matlab
%% Sampling
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a “Hanning” windows that will be used for the spectral analysis:

```
Matlab
win = hanning(ceil(1*Fs)); % Hanning Windows
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```
Matlab
% Only used to have the frequency vector "f"
[~, f] = tfestimate(apa_sweep.Va, apa_sweep.de, win, [], [], 1/Ts);
```

3.2.3 FRF Identification - Displacement

In this section, the transfer function from the excitation voltage V_a to the encoder measured displacement d_e and interferometer measurement d_a .

The coherence from V_a to d_e is computed and shown in Figure 3.9. It is quite good from 10Hz up to 500Hz.

```
Matlab
%% TF - Encoder
[coh_sweep, ~] = mscohere(apa_sweep.Va, apa_sweep.de, win, [], [], 1/Ts);
```

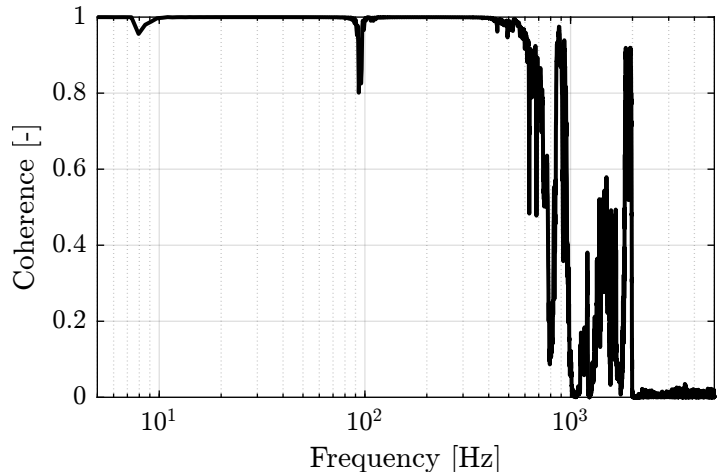


Figure 3.9: Coherence for the identification from V_a to d_e

The transfer functions are then estimated and shown in Figure 3.10.

```
Matlab
%% TF - Encoder
[dvf_sweep, ~] = tfestimate(apa_sweep.Va, apa_sweep.de, win, [], [], 1/Ts);

%% TF - Interferometer
[int_sweep, ~] = tfestimate(apa_sweep.Va, apa_sweep.da, win, [], [], 1/Ts);
```

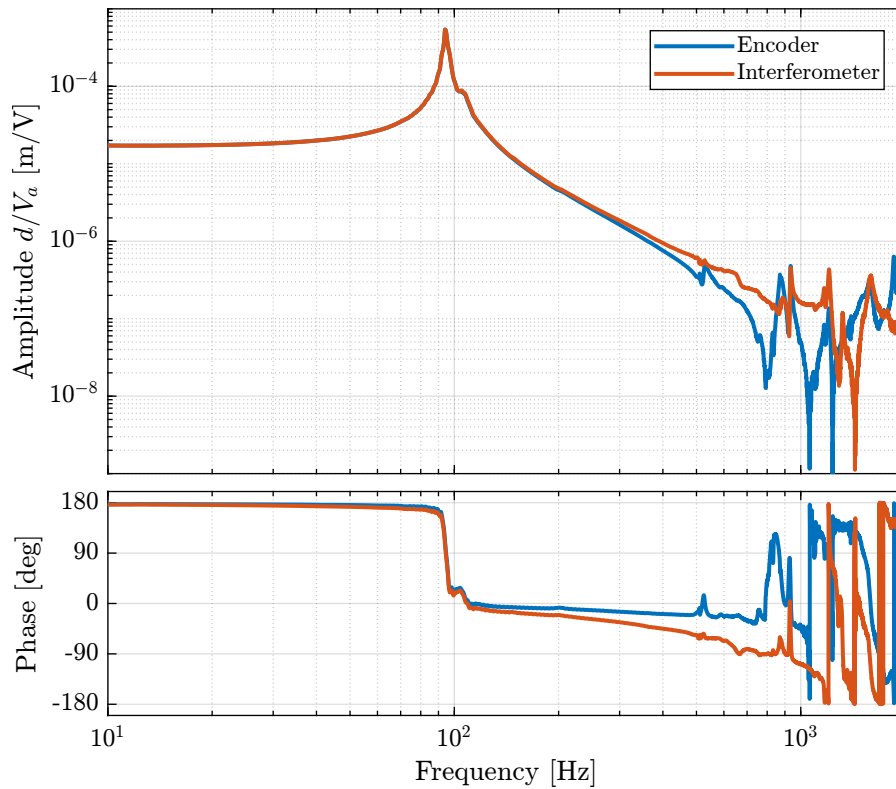


Figure 3.10: Obtained transfer functions from V_a to both d_e and d_a

3.2.4 FRF Identification - Force Sensor

Now the dynamics from excitation voltage V_a to the force sensor stack voltage V_s is identified.

The coherence is computed and shown in Figure 3.11 and found very good from 10Hz up to 2kHz.

```

Matlab
%% TF - Encoder
[coh_sweep, ~] = mscohere(apa_sweep.Va, apa_sweep.Vs, win, [], [], 1/Ts);

```

The transfer function is estimated and shown in Figure 3.12.

```

Matlab
%% Transfer function estimation
[iff_sweep, ~] = tfestimate(apa_sweep.Va, apa_sweep.Vs, win, [], [], 1/Ts);

```

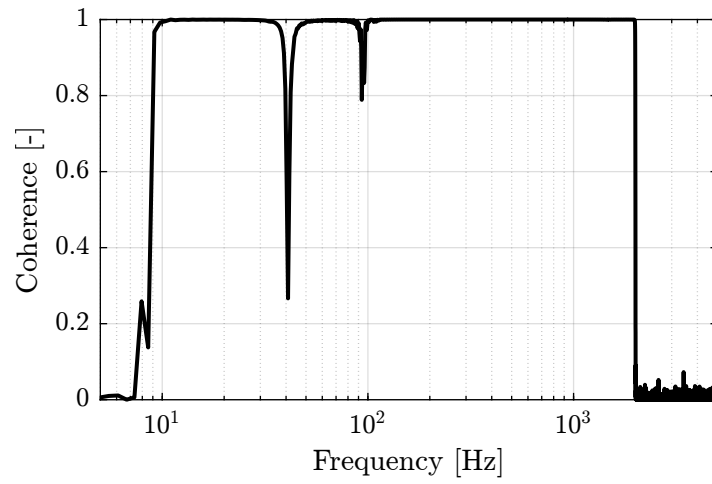


Figure 3.11: Coherence for the identification from V_a to V_s

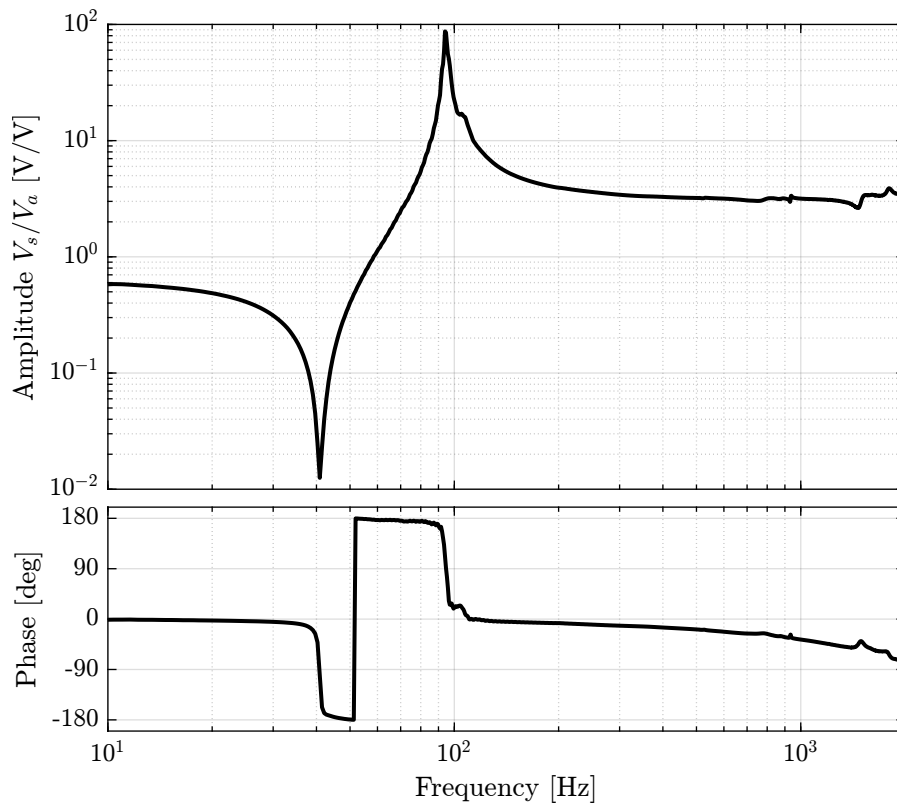


Figure 3.12: Obtained transfer functions from V_a to V_s

3.2.5 Extract Parameters (Actuator/Sensor constants)

Piezoelectric Actuator Constant

Using the measurement test-bench, it is rather easy to determine the static gain between the applied voltage V_a to the induced displacement d . Use a quasi static (1Hz) excitation signal V_a on the piezoelectric stack and measure the vertical displacement d . Perform a linear regression to obtain:

$$d = g_{d/V_a} \cdot V_a \quad (3.1)$$

Using the Simscape model of the APA, it is possible to determine the static gain between the actuator force F_a to the induced displacement d :

$$d = g_{d/F_a} \cdot F_a \quad (3.2)$$

From the two gains, it is then easy to determine g_a :

$$g_a = \frac{F_a}{V_a} = \frac{F_a}{d} \cdot \frac{d}{V_a} = \frac{g_{d/V_a}}{g_{d/F_a}} \quad (3.3)$$

Piezoelectric Sensor Constant

From a quasi static excitation of the piezoelectric stack, measure the gain from V_a to V_s :

$$V_s = g_{V_s/V_a} V_a \quad (3.4)$$

Note here that there is an high pass filter formed by the piezo capacitor and parallel resistor. The excitation frequency should then be in between the cut-off frequency of this high pass filter and the first resonance.

Alternatively, the gain can be computed from the dynamical identification and taking the gain at the wanted frequency.

Using the Simscape model, compute the static gain from the actuator force F_a to the strain of the sensor stack dl :

$$dl = g_{dl/F_a} F_a \quad (3.5)$$

Then, the static gain from the sensor stack strain dl to the general voltage V_s is:

$$g_s = \frac{V_s}{dl} = \frac{V_s}{V_a} \cdot \frac{V_a}{F_a} \cdot \frac{F_a}{dl} = \frac{g_{V_s/V_a}}{g_a \cdot g_{dl/F_a}} \quad (3.6)$$

Alternatively, we could impose an external force to add strain in the APA that should be equally present in all the 3 stacks and equal to 1/5 of the vertical strain. This external force can be some weight added, or a piezo in parallel.

Results

Quasi static gain between d and V_a :

```
g_d_Va = mean(abs(dvf_sweep(f > 10 & f < 15)));
```

```
g_d_Va = 1.7e-05 [m/V]
```

Quasi static gain between V_s and V_a :

```
g_Vs_Va = mean(abs(iff_sweep(f > 10 & f < 15)));
```

```
g_Vs_Va = 5.7e-01 [V/V]
```

3.2.6 Hysteresis

We here wish to visually see the amount of hysteresis present in the APA.

To do so, a quasi static sinusoidal excitation V_a at different voltages is used.

The offset is 65V, and the sin amplitude is ranging from 1V up to 80V.

For each excitation amplitude, the vertical displacement d of the mass is measured.

Then, d is plotted as a function of V_a for all the amplitudes.

We expect to obtain something like the hysteresis shown in Figure 3.13.

The data is loaded.

```
apa_hyst = load('frf_data_1_hysteresis.mat', 't', 'Va', 'de');  
% Initial time set to zero  
apa_hyst.t = apa_hyst.t - apa_hyst.t(1);
```

The excitation voltage amplitudes are:

```
ampls = [0.1, 0.2, 0.4, 1, 2, 4]; % Excitation voltage amplitudes
```

The excitation voltage and the measured displacement are shown in Figure 3.14.

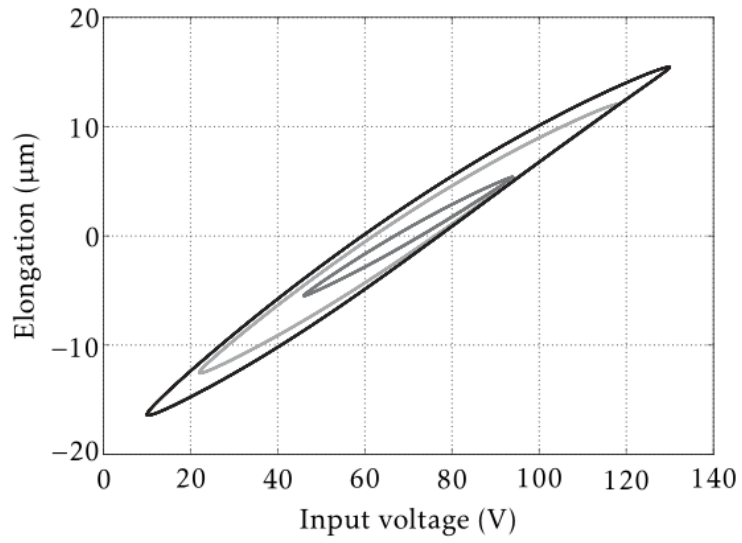


Figure 6.16: Measured hysteresis loops for a single PE actuator, 70 V bias + 1 Hz sine wave: 60 V amplitude (black), 48 V amplitude (light grey), 24 V amplitude (dark grey)

Figure 3.13: Expected Hysteresis [1]

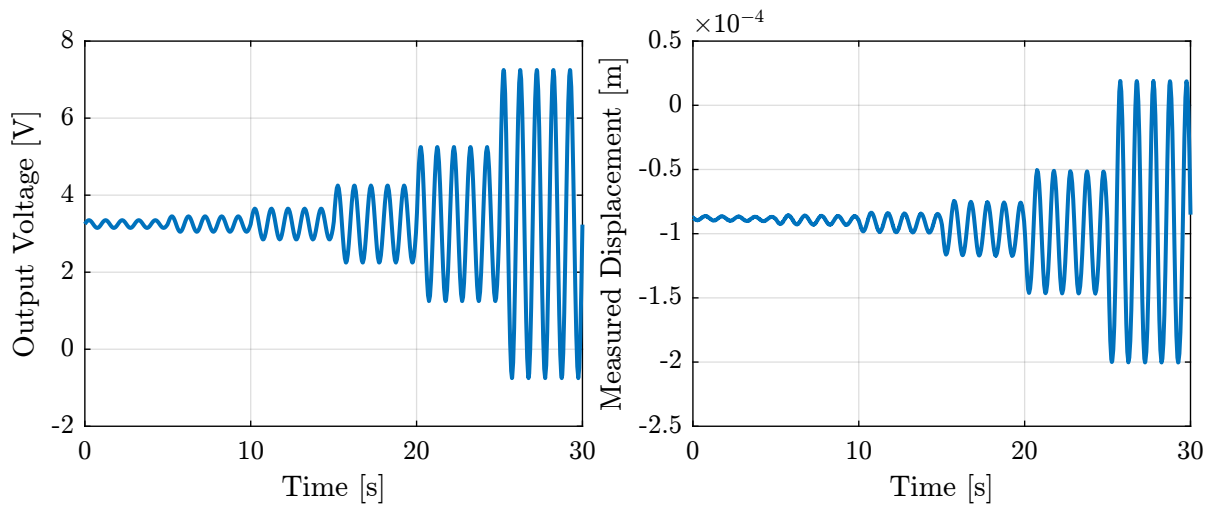


Figure 3.14: Excitation voltage and measured displacement

For each amplitude, we only take the last sinus in order to reduce possible transients. Also, it is centered on zero.

The measured displacement at a function of the output voltage are shown in Figure 3.15.

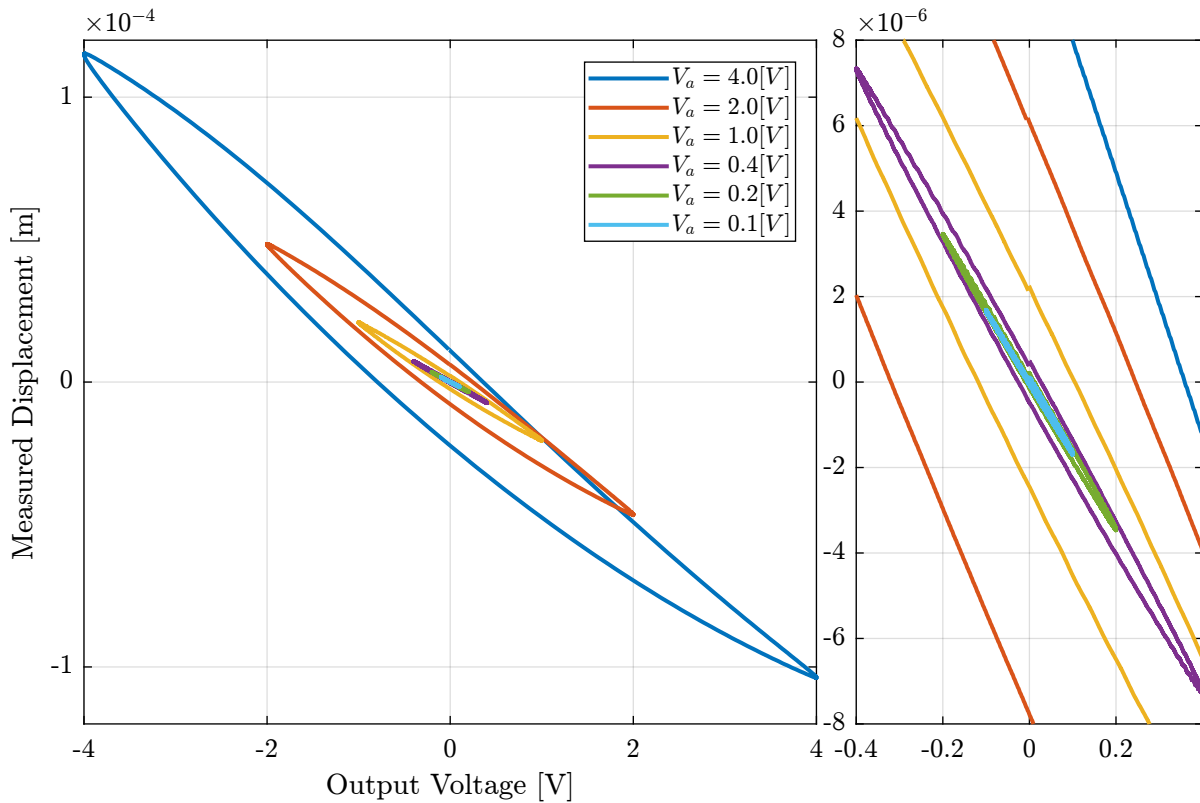


Figure 3.15: Obtained hysteresis for multiple excitation amplitudes

Important

It is quite clear that hysteresis is increasing with the excitation amplitude. Also, no hysteresis is found on the sensor stack voltage.

3.2.7 Estimation of the APA axial stiffness

In order to estimate the stiffness of the APA, a weight with known mass m_a is added on top of the suspended granite and the deflection d_e is measured using the encoder. The APA stiffness is then:

$$k_{\text{apa}} = \frac{m_a g}{d} \quad (3.7)$$

Here, a mass of 6.4 kg is used:

```
added_mass = 6.4; % Added mass [kg]
```

Matlab

The data is loaded, and the measured displacement is shown in Figure 3.16.

```

Matlab
apa_mass = load(sprintf('frf_data%i_add_mass_closed_circuit.mat', 1), 't', 'de');
apa_mass.de = apa_mass.de - mean(apa_mass.de(apa_mass.t<11));

```

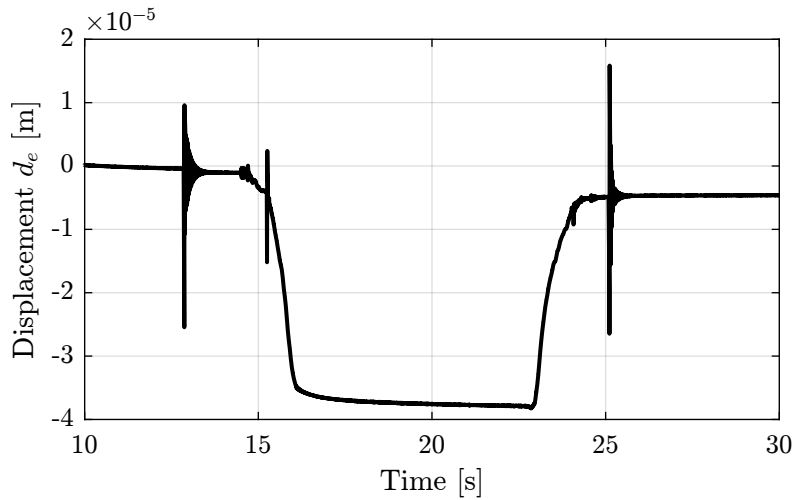


Figure 3.16: Measured displacement when adding the mass and removing the mass

There is some imprecision in the measurement as there are some drifts that are probably due to some creep.

The stiffness is then computed as follows:

```

Matlab
k = 9.8 * added_mass / (mean(apa_mass.de(apa_mass.t > 12 & apa_mass.t < 12.5)) - mean(apa_mass.de(apa_mass.t > 20 & apa_mass.t
-> < 20.5)));

```

And the stiffness obtained is very close to the one specified in the documentation ($k = 1.794 [N/\mu m]$).

```

Results
k = 1.68 [N/um]

```

3.2.8 Stiffness change due to electrical connections

We wish here to see if the stiffness changes when the actuator stacks are not connected to the amplifier and the sensor stacks are not connected to the ADC.

Note here that the resistor in parallel to the sensor stack is present in both cases.

First, the data are loaded.

```

Matlab
add_mass_oc = load(sprintf('frf_data%i_add_mass_open_circuit.mat', 1), 't', 'de');
add_mass_cc = load(sprintf('frf_data%i_add_mass_closed_circuit.mat', 1), 't', 'de');

```

And the initial displacement is set to zero.

```

Matlab
add_mass_oc.de = add_mass_oc.de - mean(add_mass_oc.de(add_mass_oc.t < 11));
add_mass_cc.de = add_mass_cc.de - mean(add_mass_cc.de(add_mass_cc.t < 11));

```

The measured displacements are shown in Figure 3.17.

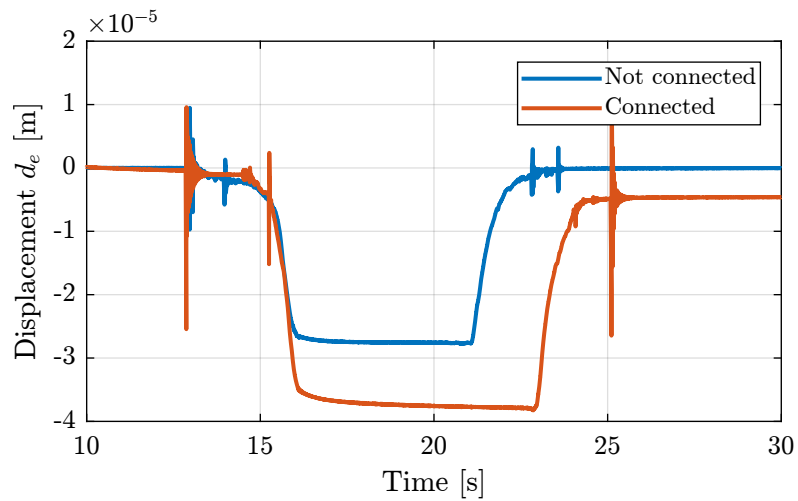


Figure 3.17: Measured displacement

And the stiffness is estimated in both case. The results are shown in Table 3.2.

```

Matlab
apa_k_oc = 9.8 * added_mass / (mean(add_mass_oc.de(add_mass_oc.t > 12 & add_mass_oc.t < 12.5)) -
-> mean(add_mass_oc.de(add_mass_oc.t > 20 & add_mass_oc.t < 20.5)));
apa_k_cc = 9.8 * added_mass / (mean(add_mass_cc.de(add_mass_cc.t > 12 & add_mass_cc.t < 12.5)) -
-> mean(add_mass_cc.de(add_mass_cc.t > 20 & add_mass_cc.t < 20.5)));

```

Table 3.2: Measured stiffnesses on “open” and “closed” circuits

	$k[N/\mu m]$
Not connected	2.3
Connected	1.7

Important

Clearly, connecting the actuator stacks to the amplified (basically equivalent as to short circuiting them) lowers the stiffness.

3.2.9 Effect of the resistor on the IFF Plant

A resistor $R \approx 80.6 \text{ k}\Omega$ is added in parallel with the sensor stack. This has the effect to form a high pass filter with the capacitance of the stack.

We here measured the low frequency transfer function from V_a to V_s with and without this resistor.

```
Matlab
% With the resistor
wi_k = load('frf_data_1_sweep_1f_with_R.mat', 't', 'Vs', 'Va');

% Without the resistor
wo_k = load('frf_data_1_sweep_1f.mat', 't', 'Vs', 'Va');
```

We use a very long “Hanning” window for the spectral analysis in order to estimate the low frequency behavior.

```
Matlab
win = hanning(ceil(50*Fs)); % Hanning Windows
```

And we estimate the transfer function from V_a to V_s in both cases:

```
Matlab
[frf_wo_k, f] = tfestimate(wo_k.Va, wo_k.Vs, win, [], [], 1/Ts);
[frf_wi_k, ~] = tfestimate(wi_k.Va, wi_k.Vs, win, [], [], 1/Ts);
```

With the following values of the resistor and capacitance, we obtain a first order high pass filter with a crossover frequency equal to:

```
Matlab
C = 5.1e-6; % Sensor Stack capacitance [F]
R = 80.6e3; % Parallel Resistor [Ohm]

f0 = 1/(2*pi*R*C); % Crossover frequency of RC HPF [Hz]
```

```
Results
f0 = 0.39 [Hz]
```

The transfer function of the corresponding high pass filter is:

```
Matlab
G_hpf = 0.6*(s/2*pi*f0)/(1 + s/2*pi*f0);
```

Let's compare the transfer function from actuator stack to sensor stack with and without the added resistor in Figure 3.18.

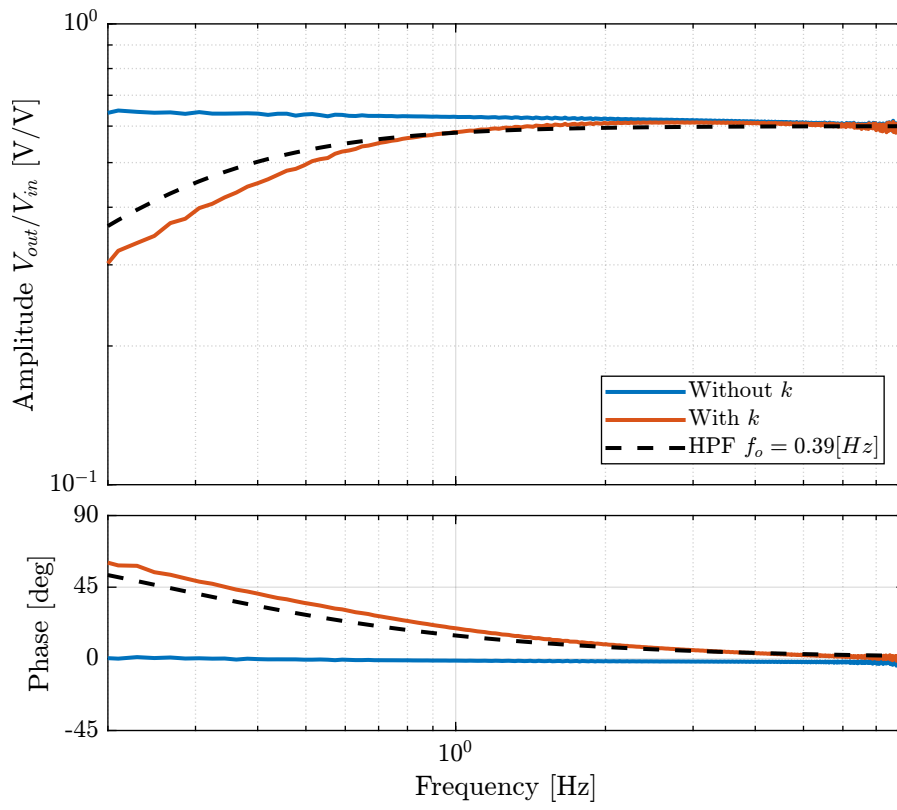


Figure 3.18: Transfer function from V_a to V_s with and without the resistor k

Important

The added resistor has indeed the expected effect.

3.3 Comparison of all the APA

The same measurements that was performed in Section 3.2 are now performed on all the APA and then compared.

3.3.1 Axial Stiffnesses - Comparison

Let's first compare the APA axial stiffnesses.

The added mass is:

```
added_mass = 6.4; % Added mass [kg]
```

Here are the number of the APA that have been measured:

```
apa_nums = [1 2 4 5 6 7 8];
```

The data are loaded.

```
apa_mass = {};  
for i = 1:length(apa_nums)  
    apa_mass(i) = {load(sprintf('frf_data_%i_add_mass_closed_circuit.mat', apa_nums(i)), 't', 'de')};  
    % The initial displacement is set to zero  
    apa_mass{i}.de = apa_mass{i}.de - mean(apa_mass{i}.de(apa_mass{i}.t<11));  
end
```

The raw measurements are shown in Figure 3.19. All the APA seems to have similar stiffness except the APA 7 which should have an higher stiffness.

Question

It is however strange that the displacement d_e when the mass is removed is higher for the APA 7 than for the other APA. What could cause that?

The stiffnesses are computed for all the APA and are summarized in Table 3.3.

Important

The APA300ML manual specifies the nominal stiffness to be $1.8 [N/\mu m]$ which is very close to what have been measured. Only the APA number 7 is a little bit off.

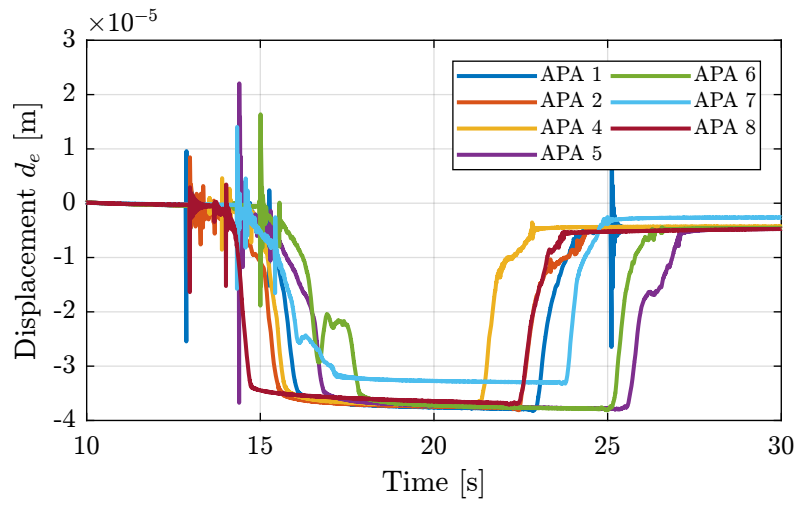


Figure 3.19: Raw measurements for all the APA. A mass of 6.4kg is added at around 15s and removed at around 22s

Table 3.3: Measured stiffnesses

APA Num	$k[N/\mu m]$
1	1.68
2	1.69
4	1.7
5	1.7
6	1.7
7	1.93
8	1.73

3.3.2 FRF Identification - Setup

The identification is performed in three steps:

1. White noise excitation with small amplitude. This is used to determine the main resonance of the system.
2. Sweep sine excitation with the amplitude lowered around the resonance. The sweep sine is from 10Hz to 400Hz.
3. High frequency noise. The noise is band-passed between 300Hz and 2kHz.

Then, the result of the second identification is used between 10Hz and 350Hz and the result of the third identification if used between 350Hz and 2kHz.

Here are the APA numbers that have been measured.

```
Matlab  
apa_nums = [1 2 4 5 6 7 8];
```

The data are loaded for both the second and third identification:

```
Matlab  
%% Second identification  
apa_sweep = {};  
for i = 1:length(apa_nums)  
    apa_sweep(i) = {load(sprintf('frf_data_%i_sweep.mat', apa_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};  
end  
  
%% Third identification  
apa_noise_hf = {};  
for i = 1:length(apa_nums)  
    apa_noise_hf(i) = {load(sprintf('frf_data_%i_noise_hf.mat', apa_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};  
end
```

The time is the same for all measurements.

```
Matlab  
%% Time vector  
t = apa_sweep{1}.t - apa_sweep{1}.t(1) ; % Time vector [s]  
  
%% Sampling  
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]  
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a “Hanning” windows that will be used for the spectral analysis:

```
Matlab  
win = hanning(ceil(0.5*Fs)); % Hanning Windows
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```

Matlab
% Only used to have the frequency vector "f"
[~, f] = tfestimate(apa_sweep{1}.Va, apa_sweep{1}.de, win, [], [], 1/Ts);

```

3.3.3 FRF Identification - DVF

In this section, the dynamics from excitation voltage V_a to encoder measured displacement d_e is identified.

We compute the coherence for 2nd and 3rd identification:

```

Matlab
%% Coherence computation
coh_sweep = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_sweep{i}.Va, apa_sweep{i}.de, win, [], [], 1/Ts);
    coh_sweep(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_noise_hf{i}.Va, apa_noise_hf{i}.de, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;
end

```

The coherence is shown in Figure 3.20. It is clear that the Sweep sine gives good coherence up to 400Hz and that the high frequency noise excitation signal helps increasing a little bit the coherence at high frequency.

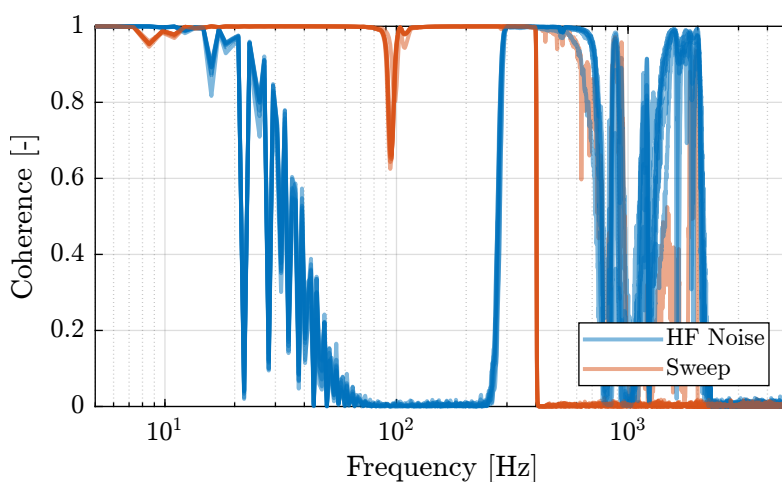


Figure 3.20: Obtained coherence for the plant from V_a to d_e

Then, the transfer function from the DAC output voltage V_a to the measured displacement by the encoders is computed:

```

Matlab
%% Transfer function estimation
dvf_sweep = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [frf, ~] = tfestimate(apa_sweep{i}.Va, apa_sweep{i}.de, win, [], [], 1/Ts);
    dvf_sweep(:, i) = frf;
end

```

```

end
dvf_noise_hf = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [frf, ~] = tfestimate(apa_noise_hf{i}.Va, apa_noise_hf{i}.de, win, [], [], 1/Ts);
    dvf_noise_hf(:, i) = frf;
end

```

The obtained transfer functions are shown in Figure 3.21. They are all superimposed except for the APA7.

Question

Why is the APA7 off? We could think that the APA7 is stiffer, but also the mass line is off. It seems that there is a “gain” problem. The encoder seems fine (it measured the same as the Interferometer). Maybe it could be due to the amplifier?

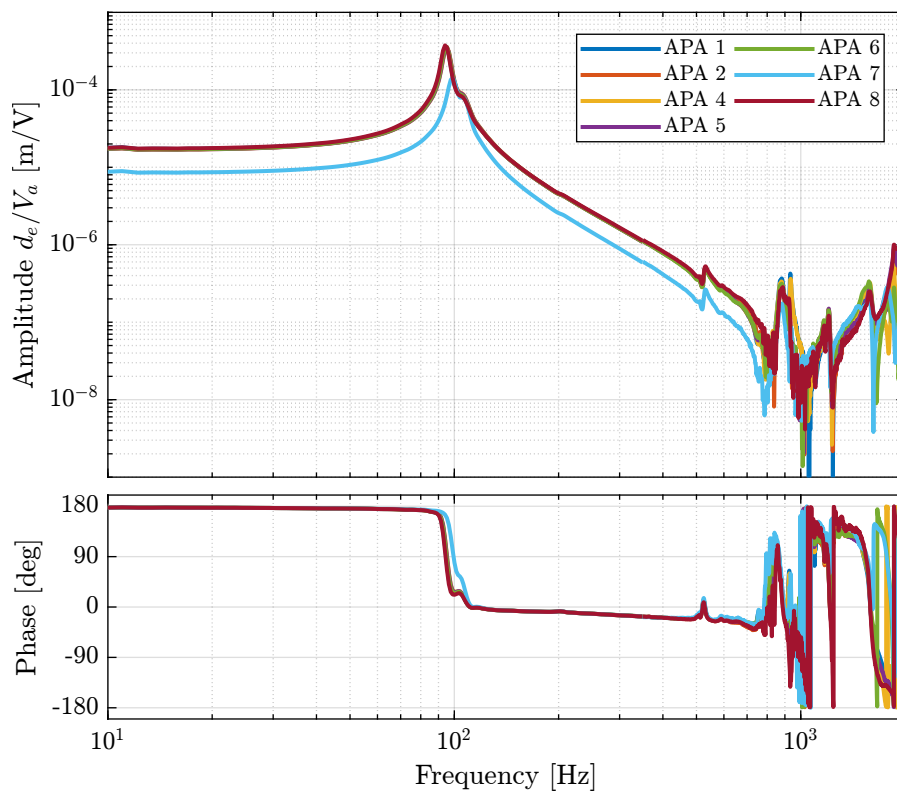


Figure 3.21: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

A zoom on the main resonance is shown in Figure 3.22. It is clear that expect for the APA 7, the response around the resonances are well matching for all the APA.

It is also clear that there is not a single resonance but two resonances, a first one at 95Hz and a second one at 105Hz.

Question

Why is there a double resonance at around 94Hz?

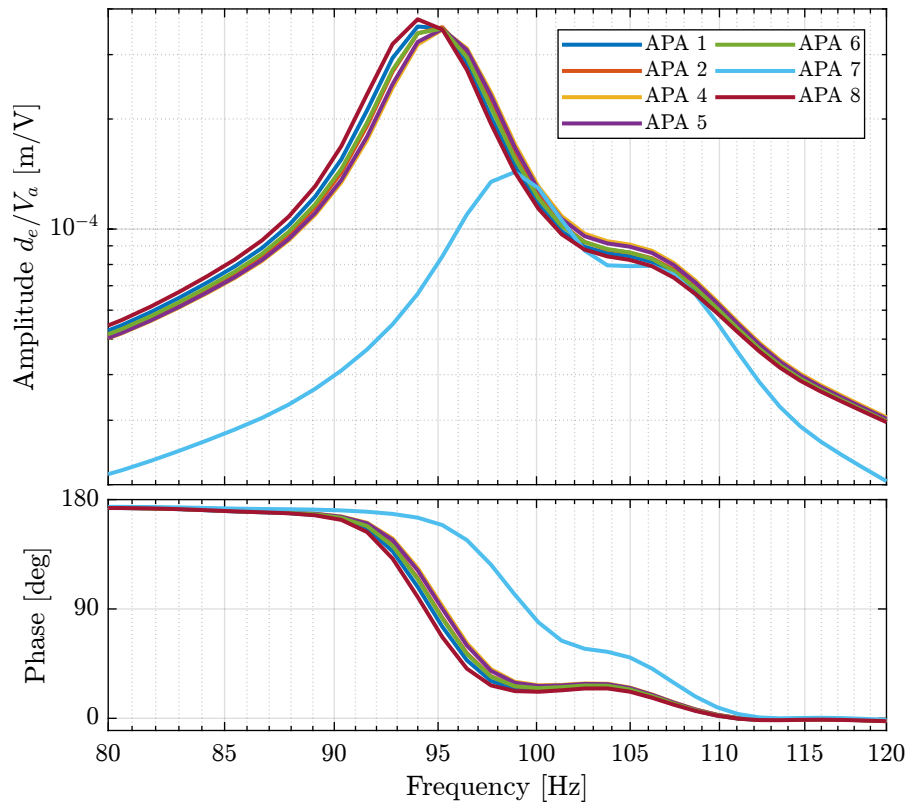


Figure 3.22: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e) - Zoom on the main resonance

3.3.4 FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

First the coherence is computed and shown in Figure 3.23. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

```
Matlab
%% Coherence
coh_sweep = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_sweep{i}.Va, apa_sweep{i}.Vs, win, [], [], 1/Ts);
    coh_sweep(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_noise_hf{i}.Va, apa_noise_hf{i}.Vs, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;
end
```

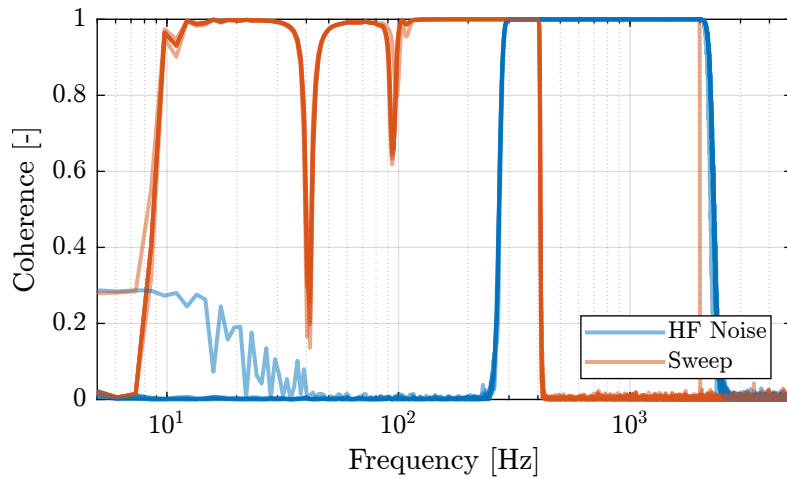


Figure 3.23: Obtained coherence for the IFF plant

Then the FRF are estimated and shown in Figure 3.24

```

Matlab
%% FRF estimation of the transfer function from Va to Vs
iff_sweep = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [frf, ~] = tfestimate(apa_sweep{i}.Va, apa_sweep{i}.Vs, win, [], [], 1/Ts);
    iff_sweep(:, i) = frf;
end

iff_noise_hf = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [frf, ~] = tfestimate(apa_noise_hf{i}.Va, apa_noise_hf{i}.Vs, win, [], [], 1/Ts);
    iff_noise_hf(:, i) = frf;
end

```

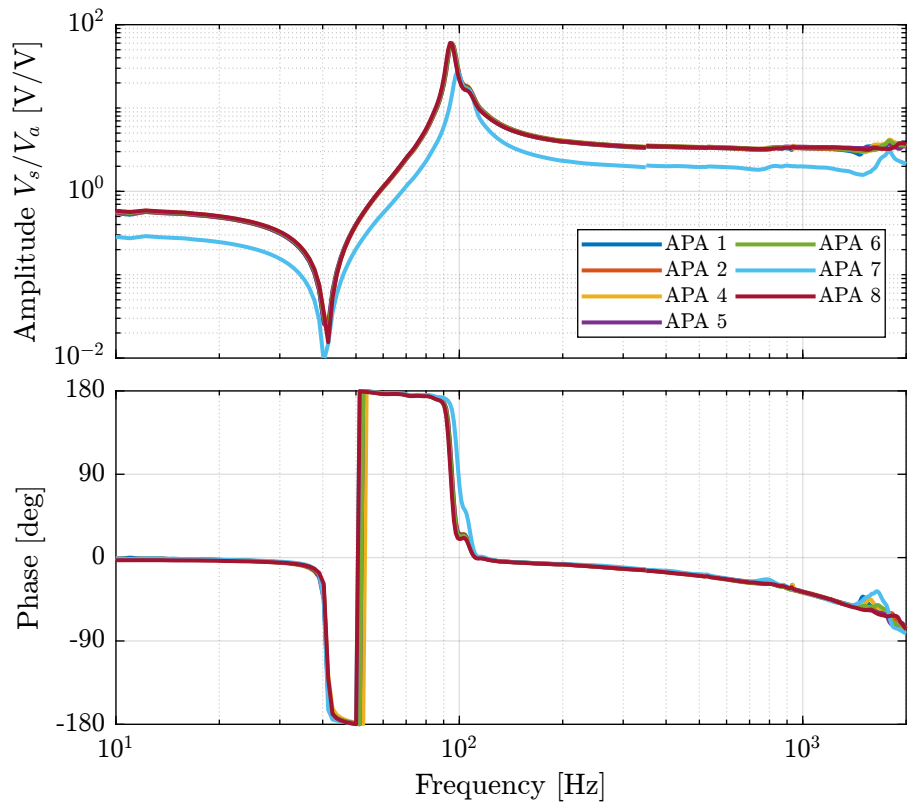


Figure 3.24: Identified IFF Plant

4 Dynamical measurements - Struts

The same bench used in Section 3 is here used with the strut instead of only the APA.

The bench is shown in Figure 4.3. Measurements are performed either when no encoder is fixed to the strut (Figure 4.2) or when one encoder is fixed to the strut (Figure 4.3).

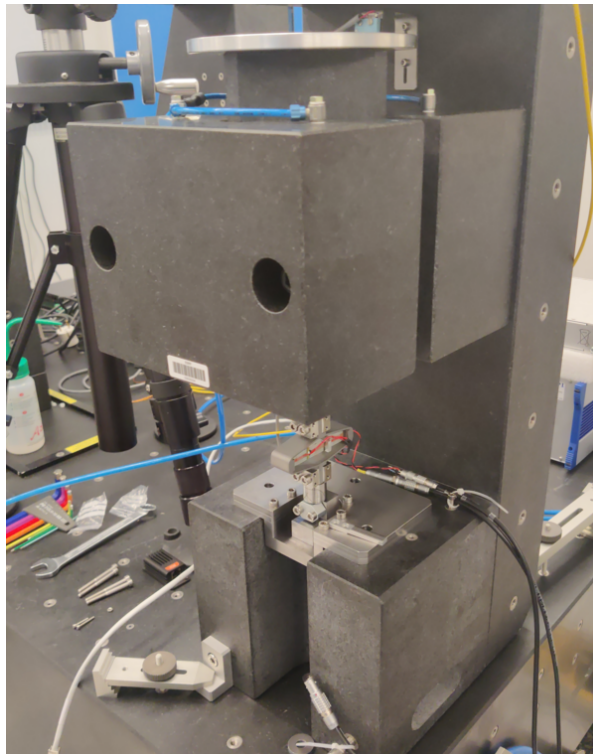


Figure 4.1: Test Bench with Strut - Overview

4.1 Measurement on Strut 1

Measurements are first performed on the strut 1 that contains:

- APA 1
- flex 1 and flex 2

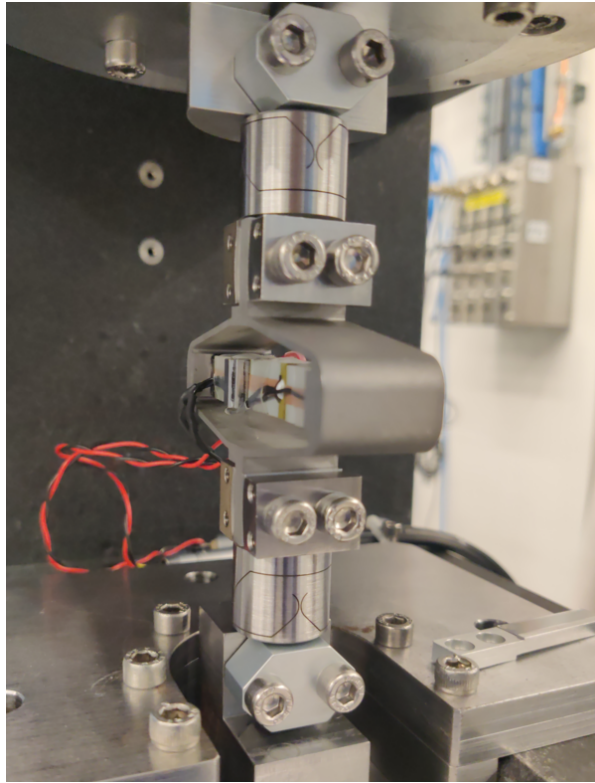


Figure 4.2: Test Bench with Strut - Zoom on the strut

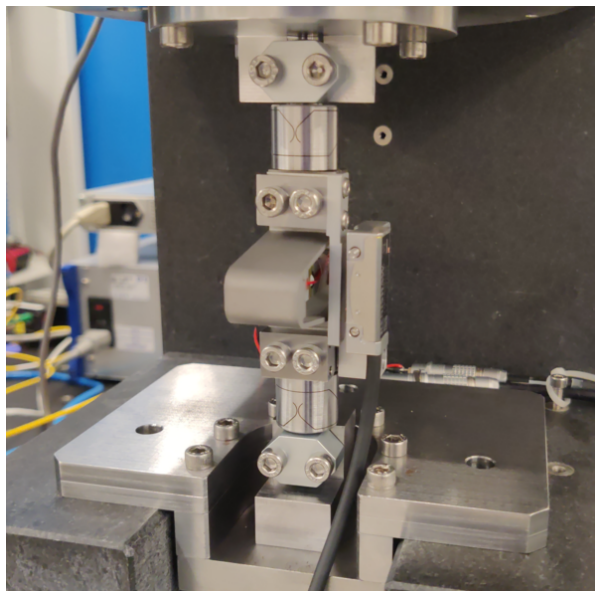


Figure 4.3: Test Bench with Strut - Zoom on the strut with the encoder

4.1.1 Without Encoder

FRF Identification - Setup

The identification is performed in three steps:

1. White noise excitation with small amplitude. This is used to determine the main resonance of the system.
2. Sweep sine excitation with the amplitude lowered around the resonance. The sweep sine is from 10Hz to 400Hz.
3. High frequency noise. The noise is band-passed between 300Hz and 2kHz.

Then, the result of the second identification is used between 10Hz and 350Hz and the result of the third identification if used between 350Hz and 2kHz.

```
Matlab
leg_sweep = load(sprintf('frf_data_leg_%i_sweep.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
leg_noise_hf = load(sprintf('frf_data_leg_%i_noise_hf.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
```

The time is the same for all measurements.

```
Matlab
%% Time vector
t = leg_sweep.t - leg_sweep.t(1); % Time vector [s]

%% Sampling
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a “Hanning” windows that will be used for the spectral analysis:

```
Matlab
win = hanning(ceil(0.5*Fs)); % Hanning Windows
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```
Matlab
% Only used to have the frequency vector "f"
[~, f] = tfestimate(leg_sweep.Va, leg_sweep.de, win, [], [], 1/Ts);
```

FRF Identification - Displacement

In this section, the dynamics from the excitation voltage V_a to the interferometer d_a is identified.

We compute the coherence for 2nd and 3rd identification:

```

Matlab
[coh_sweep, ~] = mscohere(leg_sweep.Va, leg_sweep.da, win, [], [], 1/Ts);
[coh_noise_hf, ~] = mscohere(leg_noise_hf.Va, leg_noise_hf.da, win, [], [], 1/Ts);

```

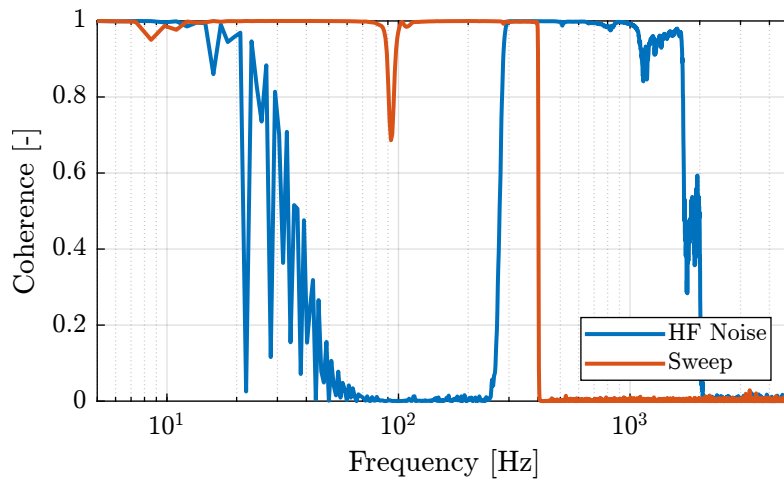


Figure 4.4: Obtained coherence for the plant from V_a to d_a

The transfer function from V_a to the interferometer measured displacement d_a is estimated and shown in Figure 4.5.

```

Matlab
[dvf_sweep, ~] = tfestimate(leg_sweep.Va, leg_sweep.da, win, [], [], 1/Ts);
[dvf_noise_hf, ~] = tfestimate(leg_noise_hf.Va, leg_noise_hf.da, win, [], [], 1/Ts);

```

FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

First the coherence is computed and shown in Figure 4.15. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

```

Matlab
[coh_sweep, ~] = mscohere(leg_sweep.Va, leg_sweep.Vs, win, [], [], 1/Ts);
[coh_noise_hf, ~] = mscohere(leg_noise_hf.Va, leg_noise_hf.Vs, win, [], [], 1/Ts);

```

Then the FRF are estimated and shown in Figure 4.7

```

Matlab
[iff_sweep, ~] = tfestimate(leg_sweep.Va, leg_sweep.Vs, win, [], [], 1/Ts);
[iff_noise_hf, ~] = tfestimate(leg_noise_hf.Va, leg_noise_hf.Vs, win, [], [], 1/Ts);

```

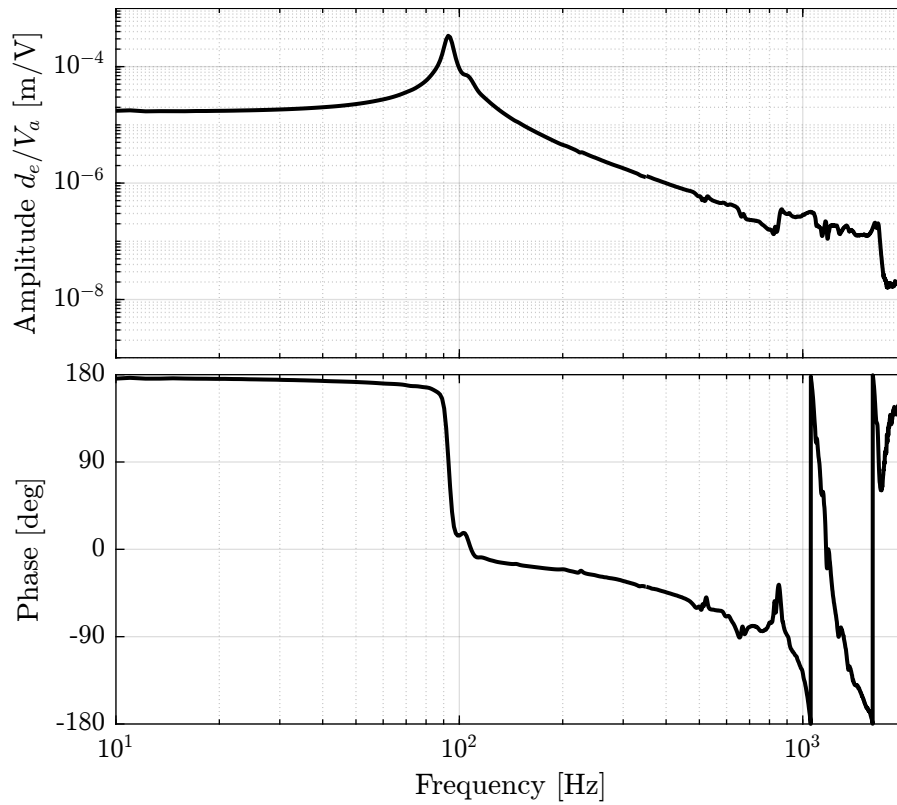


Figure 4.5: Estimated FRF for the DVF plant (transfer function from V_a to the interferometer d_a)

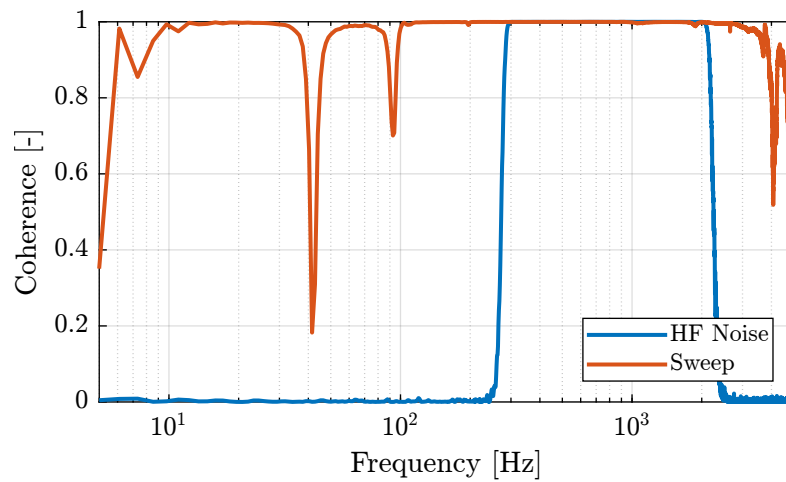


Figure 4.6: Obtained coherence for the IFF plant

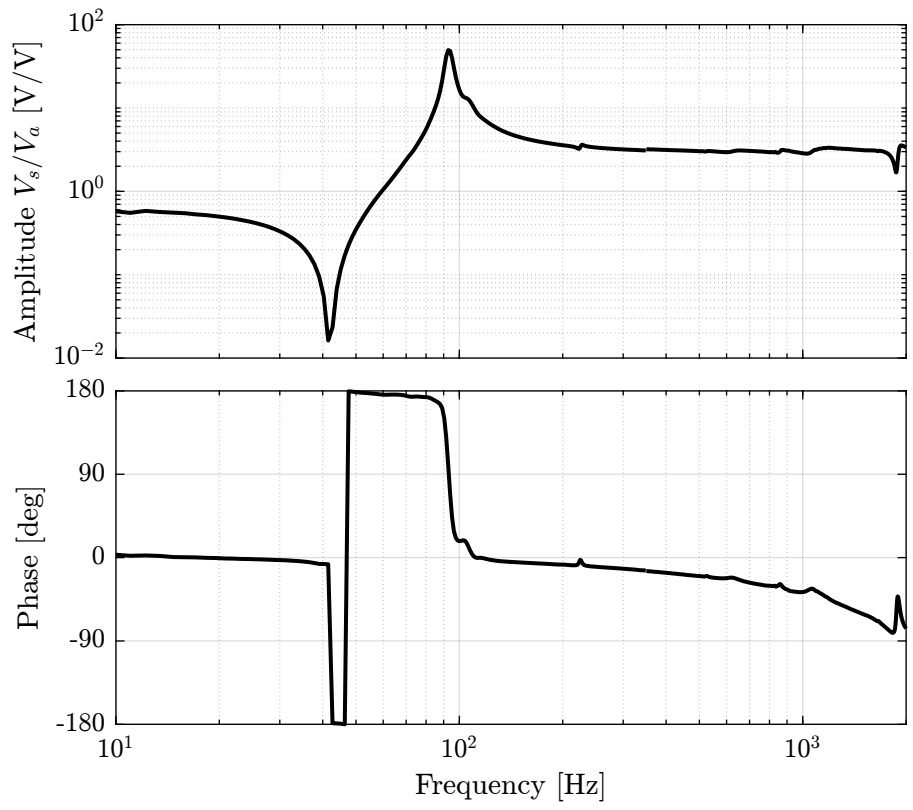


Figure 4.7: Identified IFF Plant for the Strut 1

4.1.2 With Encoder

Measurement Data

```
Matlab
leg_enc_sweep = load(sprintf('frf_data_leg_coder_badly_align_%i_noise.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
leg_enc_noise_hf = load(sprintf('frf_data_leg_coder_badly_align_%i_noise_hf.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
```

FRF Identification - DVF

In this section, the dynamics from V_a to d_e is identified.

We compute the coherence for 2nd and 3rd identification:

```
Matlab
[coh_enc_sweep, ~] = mscohere(leg_enc_sweep.Va, leg_enc_sweep.de, win, [], [], 1/Ts);
[coh_enc_noise_hf, ~] = mscohere(leg_enc_noise_hf.Va, leg_enc_noise_hf.de, win, [], [], 1/Ts);
```

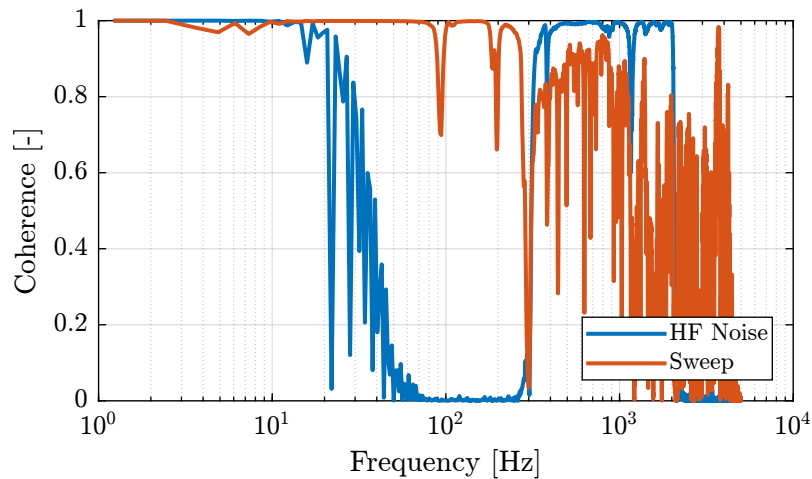


Figure 4.8: Obtained coherence for the plant from V_a to d_e

```
Matlab
[dvf_enc_sweep, ~] = tfestimate(leg_enc_sweep.Va, leg_enc_sweep.de, win, [], [], 1/Ts);
[dvf_enc_noise_hf, ~] = tfestimate(leg_enc_noise_hf.Va, leg_enc_noise_hf.de, win, [], [], 1/Ts);
```

```
Matlab
[dvf_int_sweep, ~] = tfestimate(leg_enc_sweep.Va, leg_enc_sweep.da, win, [], [], 1/Ts);
[dvf_int_noise_hf, ~] = tfestimate(leg_enc_noise_hf.Va, leg_enc_noise_hf.da, win, [], [], 1/Ts);
```

The obtained transfer functions are shown in Figure 4.9.

They are all superimposed except for the APA7.

Question

Why is the APA7 off? We could think that the APA7 is stiffer, but also the mass line is off. It seems that there is a “gain” problem. The encoder seems fine (it measured the same as the Interferometer). Maybe it could be due to the amplifier?

Question

Why is there a double resonance at around 94Hz?

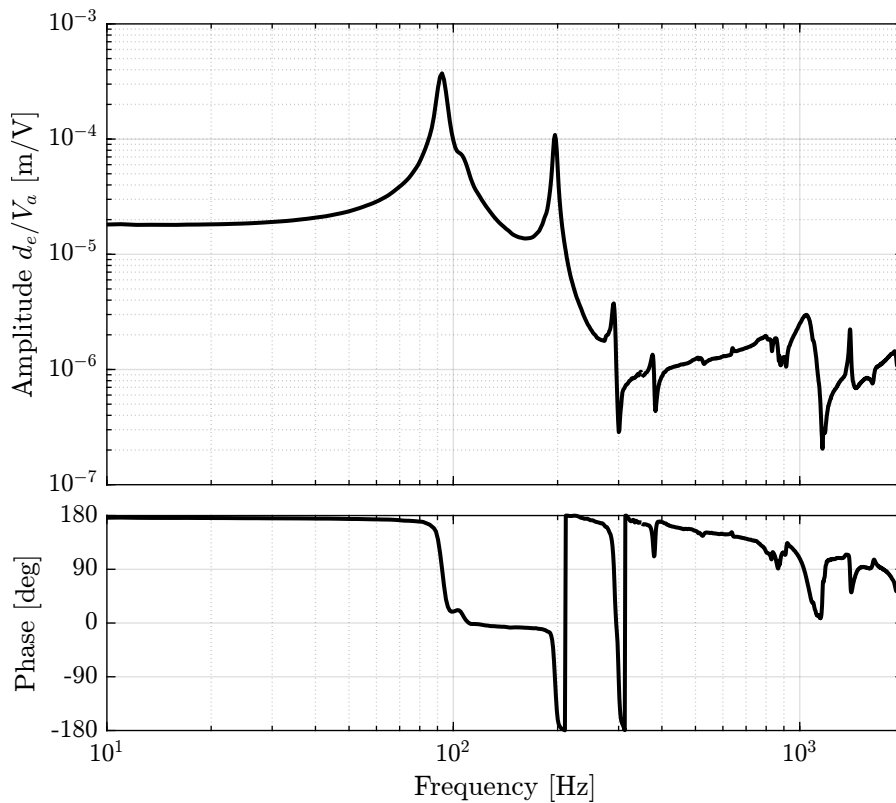


Figure 4.9: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

Comparison of the Encoder and Interferometer

The interferometer could here represent the case where the encoders are fixed to the plates and not the APA.

The dynamics from V_a to d_e and from V_a to d_a are compared in Figure 4.10.

Important

It will clearly be difficult to do something (except some low frequency positioning) with the encoders fixed to the APA.

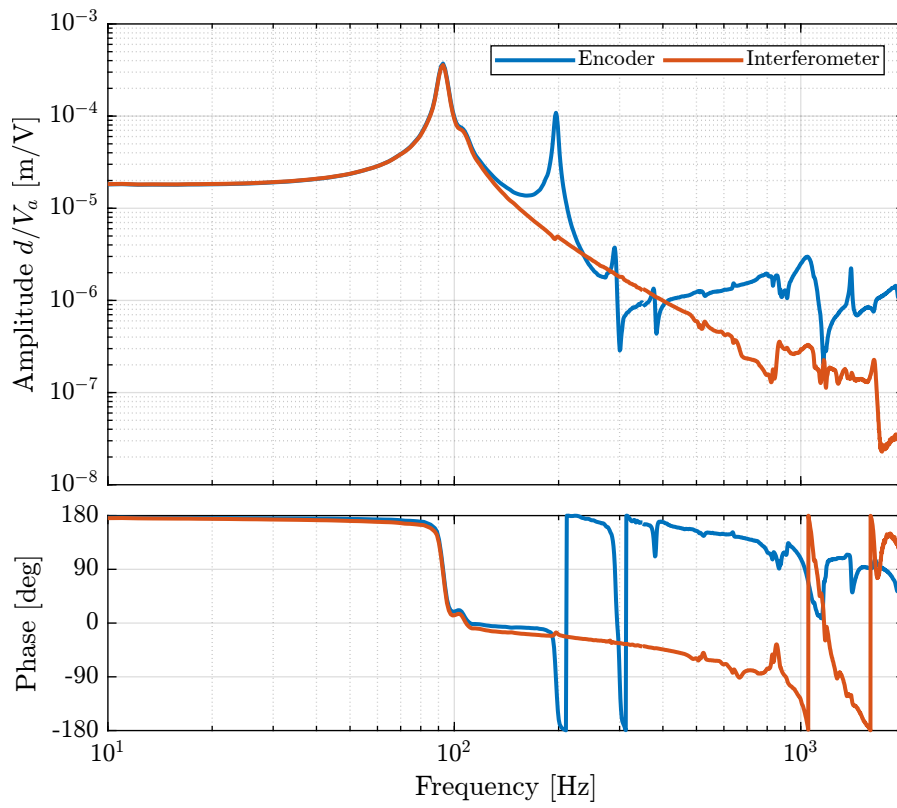


Figure 4.10: Comparison of the transfer functions from excitation voltage V_a to either the encoder d_e or the interferometer d_a

APA Resonances Frequency

As shown in Figure 4.11, we can clearly see three spurious resonances at 197Hz, 290Hz and 376Hz.

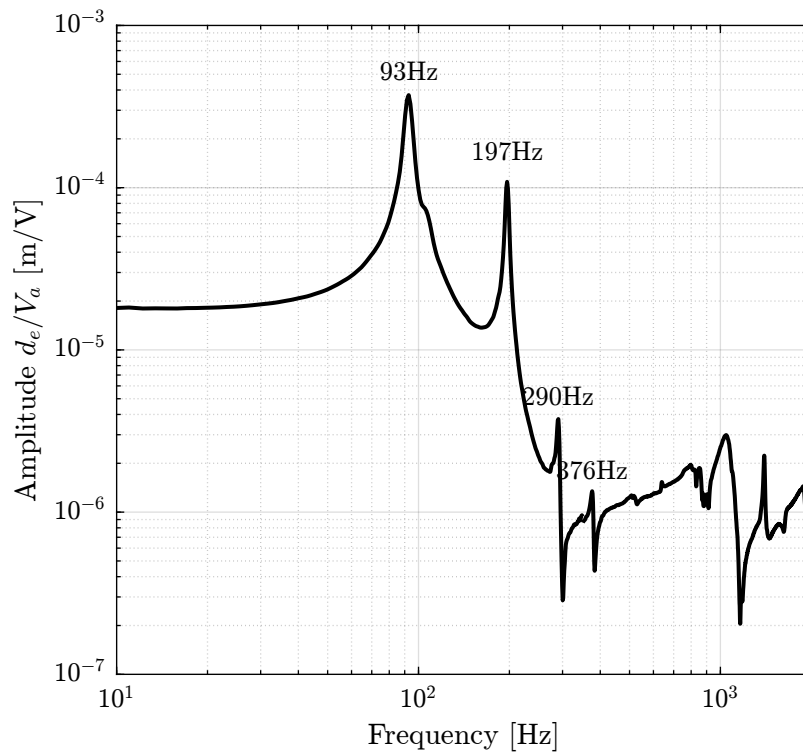


Figure 4.11: Magnitude of the transfer function from excitation voltage V_a to encoder measurement d_e . The frequency of the resonances are noted.

These resonances correspond to parasitic resonances of the APA itself. They are very close to what was estimated using the FEM:

- X-bending mode at around 190Hz (Figure 4.12)
- Y-bending mode at around 290Hz (Figure 4.13)
- Z-torsion mode at around 400Hz (Figure 4.14)

Important

The resonances are indeed due to limited stiffness of the APA.

Estimated Flexible Joint axial stiffness

FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

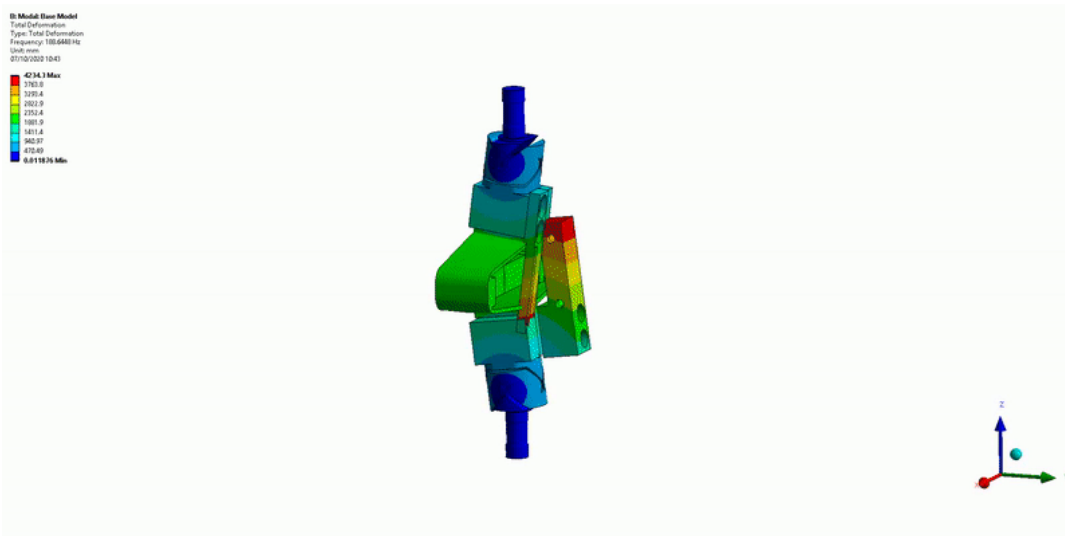


Figure 4.12: X-bending mode (189Hz)

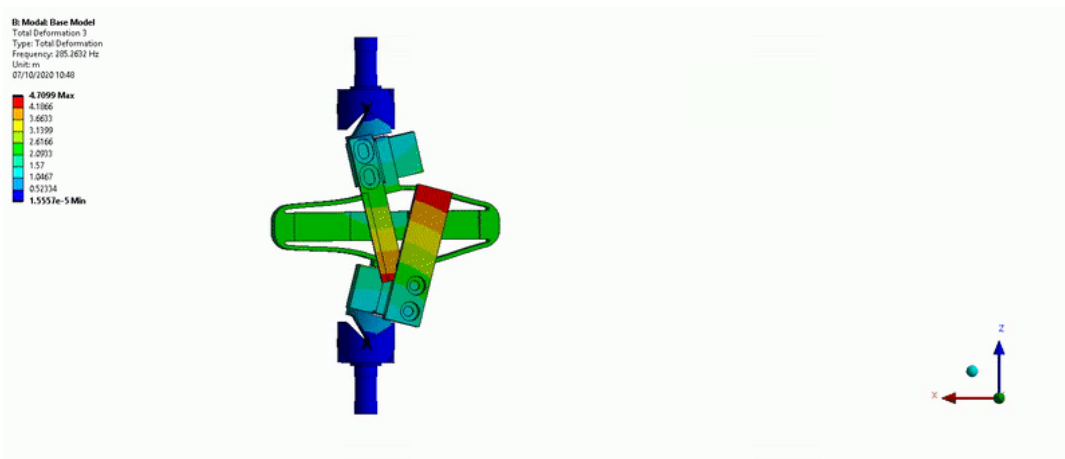


Figure 4.13: Y-bending mode (285Hz)

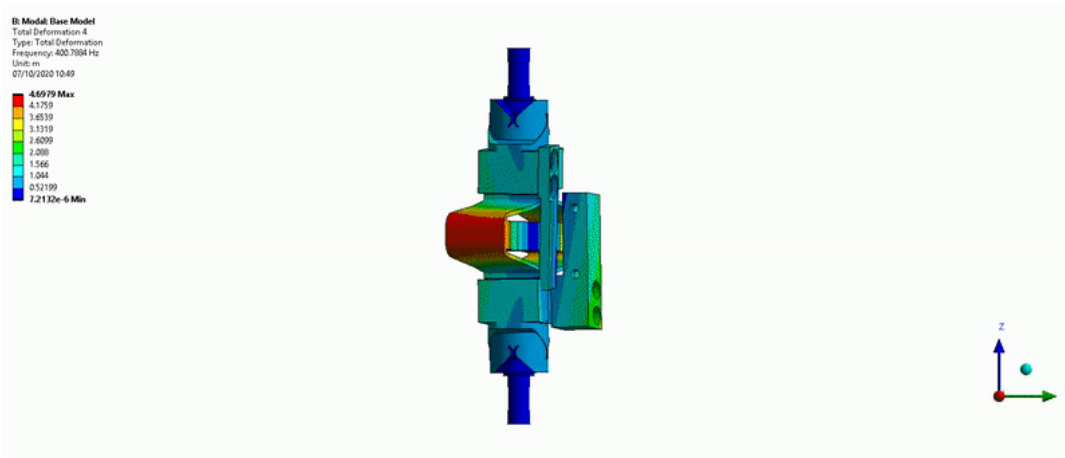


Figure 4.14: Z-torsion mode (400Hz)

First the coherence is computed and shown in Figure 4.15. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

```

Matlab
[coh_enc_sweep, ~] = mscohere(leg_enc_sweep.Va, leg_enc_sweep.Vs, win, [], [], 1/Ts);
[coh_enc_noise_hf, ~] = mscohere(leg_enc_noise_hf.Va, leg_enc_noise_hf.Vs, win, [], [], 1/Ts);

```

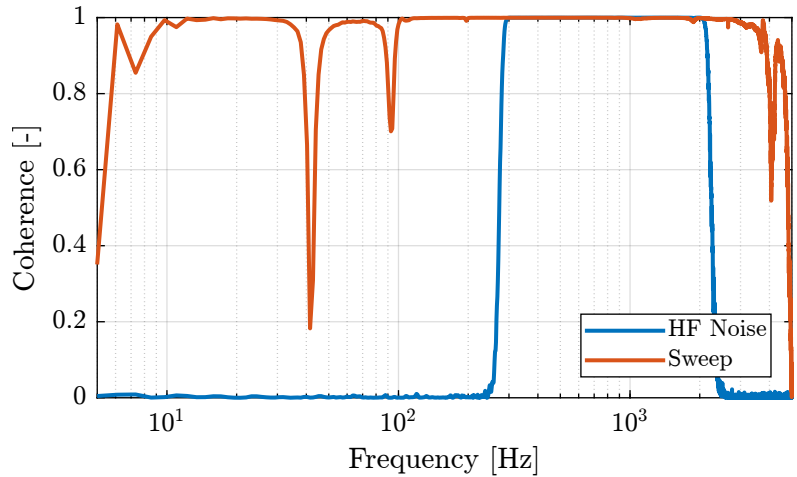


Figure 4.15: Obtained coherence for the IFF plant

Then the FRF are estimated and shown in Figure 4.16

```

Matlab
[iff_enc_sweep, ~] = tfestimate(leg_enc_sweep.Va, leg_enc_sweep.Vs, win, [], [], 1/Ts);
[iff_enc_noise_hf, ~] = tfestimate(leg_enc_noise_hf.Va, leg_enc_noise_hf.Vs, win, [], [], 1/Ts);

```

Let's now compare the IFF plants whether the encoders are fixed to the APA or not (Figure 4.17).

Important

We can see that the IFF does not change whether of not the encoder are fixed to the struts.

4.2 Comparison of all the Struts

Now all struts are measured using the same procedure and test bench.

4.2.1 FRF Identification - Setup

The identification is performed in two steps:

1. White noise excitation with small amplitude. This is used to estimate the low frequency dynamics.

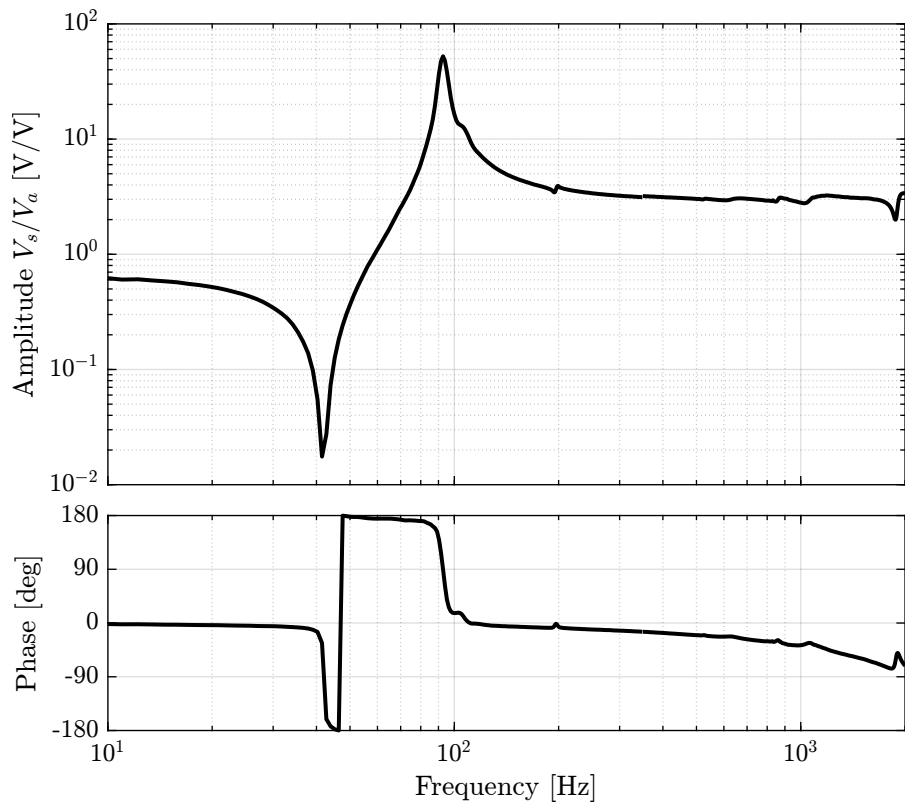


Figure 4.16: Identified IFF Plant

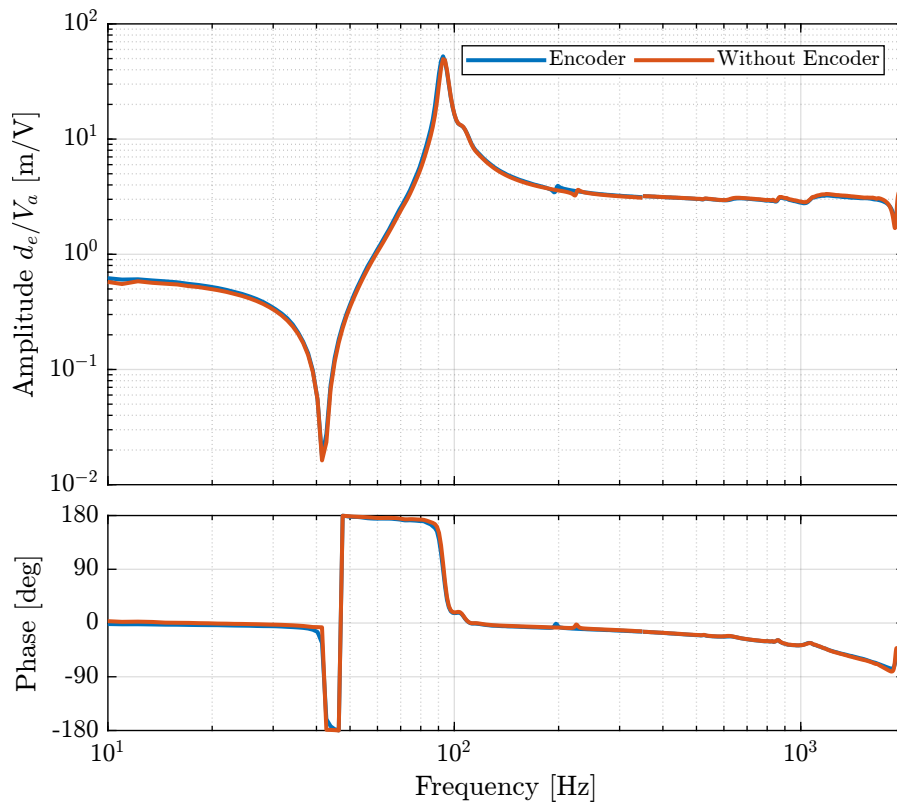


Figure 4.17: Effect of the encoder on the IFF plant

2. High frequency noise. The noise is band-passed between 300Hz and 2kHz.

Then, the result of the first identification is used between 10Hz and 350Hz and the result of the second identification if used between 350Hz and 2kHz.

Here are the LEG numbers that have been measured.

```
----- Matlab -----  
leg_nums = [1 2 3 4 5];  
-----
```

The data are loaded for both the first and second identification:

```
----- Matlab -----  
%% Second identification  
leg_noise = {};  
for i = 1:length(leg_nums)  
    leg_noise(i) = {load(sprintf('frf_data_leg_coder_%i_noise.mat', leg_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};  
end  
  
%% Third identification  
leg_noise_hf = {};  
for i = 1:length(leg_nums)  
    leg_noise_hf(i) = {load(sprintf('frf_data_leg_coder_%i_noise_hf.mat', leg_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};  
end  
-----
```

The time is the same for all measurements.

```
----- Matlab -----  
%% Time vector  
t = leg_noise{1}.t - leg_noise{1}.t(1) ; % Time vector [s]  
  
%% Sampling  
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]  
Fs = 1/Ts; % Sampling Frequency [Hz]  
-----
```

Then we defined a “Hanning” windows that will be used for the spectral analysis:

```
----- Matlab -----  
win = hanning(ceil(0.5*Fs)); % Hanning Windows  
-----
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```
----- Matlab -----  
% Only used to have the frequency vector "f"  
[~, f] = tfestimate(leg_noise{1}.Va, leg_noise{1}.de, win, [], [], 1/Ts);  
-----
```

4.2.2 FRF Identification - DVF

In this section, the dynamics from V_a to d_e is identified.

We compute the coherence for 2nd and 3rd identification:

```

Matlab
%% Coherence computation
coh_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise{i}.Va, leg_noise{i}.de, win, [], [], 1/Ts);
    coh_noise(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise_hf{i}.Va, leg_noise_hf{i}.de, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;
end

```

The coherence is shown in Figure 4.18. It is clear that the Noise sine gives good coherence up to 400Hz and that the high frequency noise excitation signal helps increasing a little bit the coherence at high frequency.

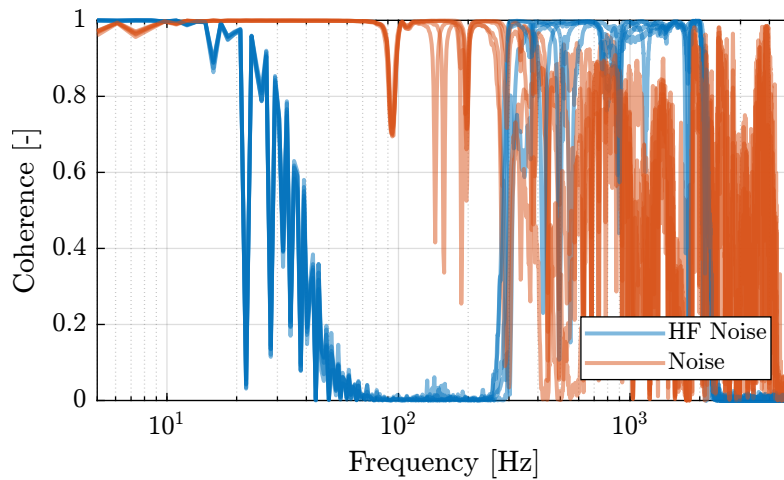


Figure 4.18: Obtained coherence for the plant from V_a to d_e

Then, the transfer function from the DAC output voltage V_a to the measured displacement by the encoders is computed:

```

Matlab
%% Transfer function estimation
dvf_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise{i}.Va, leg_noise{i}.de, win, [], [], 1/Ts);
    dvf_noise(:, i) = frf;
end

dvf_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise_hf{i}.Va, leg_noise_hf{i}.de, win, [], [], 1/Ts);
    dvf_noise_hf(:, i) = frf;
end

```

The obtained transfer functions are shown in Figure 4.19.

They are all superimposed except for the LEG7.

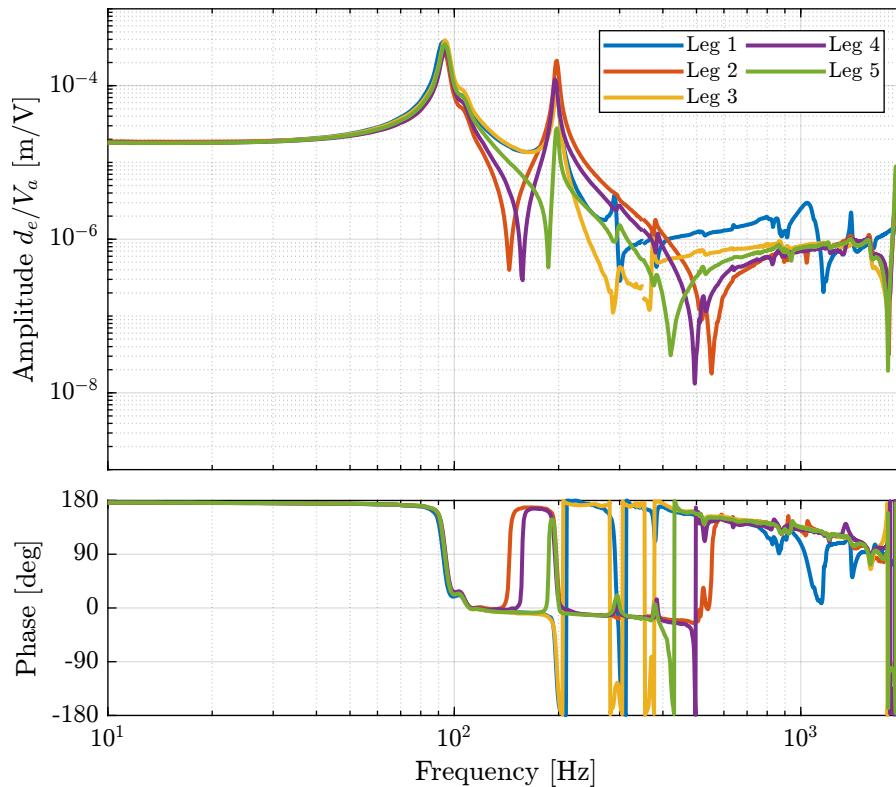


Figure 4.19: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

Important

Depending on how the APA are mounted with the flexible joints, the dynamics can change a lot as shown in Figure 4.19. In the future, a “pin” will be used to better align the APA with the flexible joints. We can expect the amplitude of the spurious resonances to decrease.

4.2.3 FRF Identification - DVF with interferometer

In this section, the dynamics from V_a to d_a is identified.

We compute the coherence.

```

Matlab
%% Coherence computation
coh_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise{i}.Va, leg_noise{i}.da, win, [], [], 1/Ts);
    coh_noise(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise_hf{i}.Va, leg_noise_hf{i}.da, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;
end

```

The coherence is shown in Figure 4.20. It is clear that the Noise sine gives good coherence up to 400Hz and that the high frequency noise excitation signal helps increasing a little bit the coherence at high frequency.

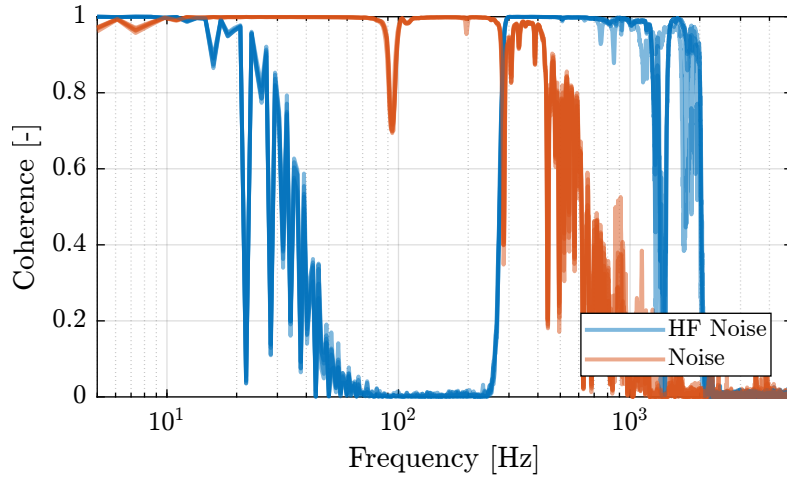


Figure 4.20: Obtained coherence for the plant from V_a to d_e

Then, the transfer function from the DAC output voltage V_a to the measured displacement by the Attocube is computed:

```

Matlab
%% Transfer function estimation
dvh_a_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise{i}.Va, leg_noise{i}.da, win, [], [], 1/Ts);
    dvh_a_noise(:, i) = frf;
end

dvh_a_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise_hf{i}.Va, leg_noise_hf{i}.da, win, [], [], 1/Ts);
    dvh_a_noise_hf(:, i) = frf;
end

```

The obtained transfer functions are shown in Figure 4.21.

They are all superimposed except for the LEG7.

4.2.4 FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

First the coherence is computed and shown in Figure 4.22. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

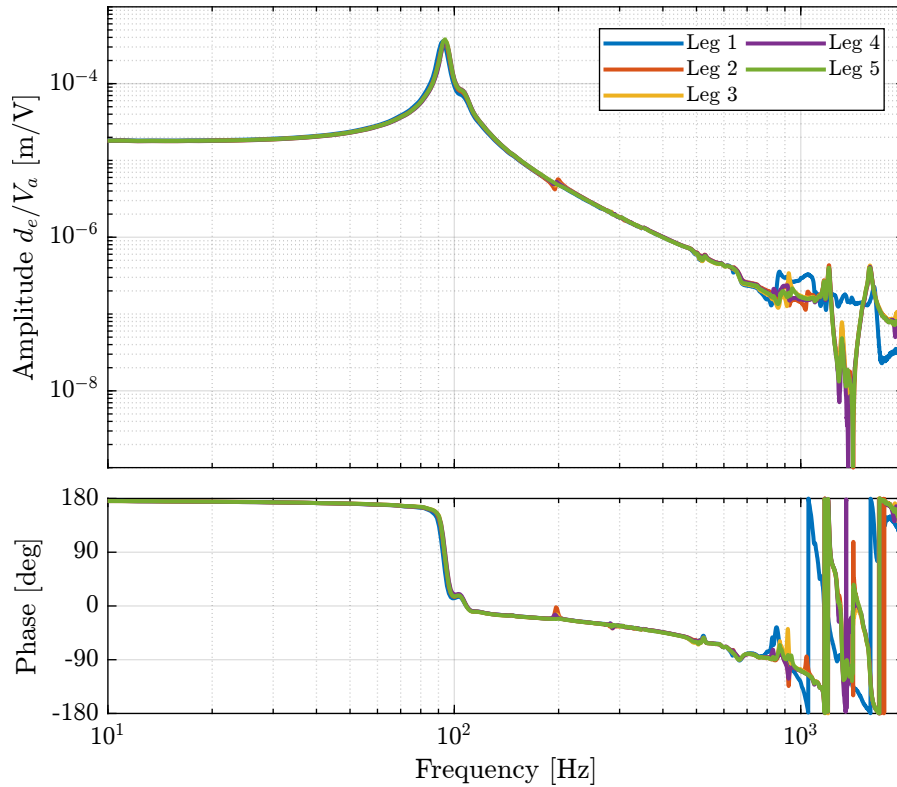


Figure 4.21: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

```

Matlab
%% Coherence
coh_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise{i}.Va, leg_noise{i}.Vs, win, [], [], 1/Ts);
    coh_noise(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise_hf{i}.Va, leg_noise_hf{i}.Vs, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;
end

```

Then the FRF are estimated and shown in Figure 4.23

```

Matlab
%% FRF estimation of the transfer function from Va to Vs
iff_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise{i}.Va, leg_noise{i}.Vs, win, [], [], 1/Ts);
    iff_noise(:, i) = frf;
end

iff_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise_hf{i}.Va, leg_noise_hf{i}.Vs, win, [], [], 1/Ts);
    iff_noise_hf(:, i) = frf;
end

```

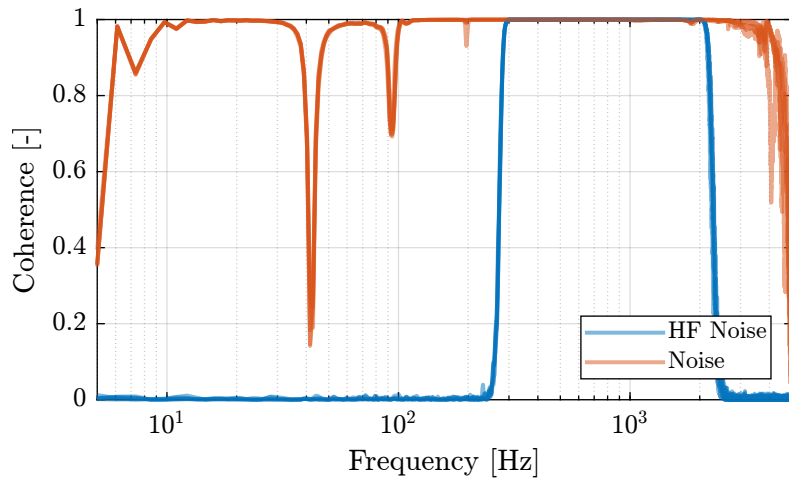


Figure 4.22: Obtained coherence for the IFF plant

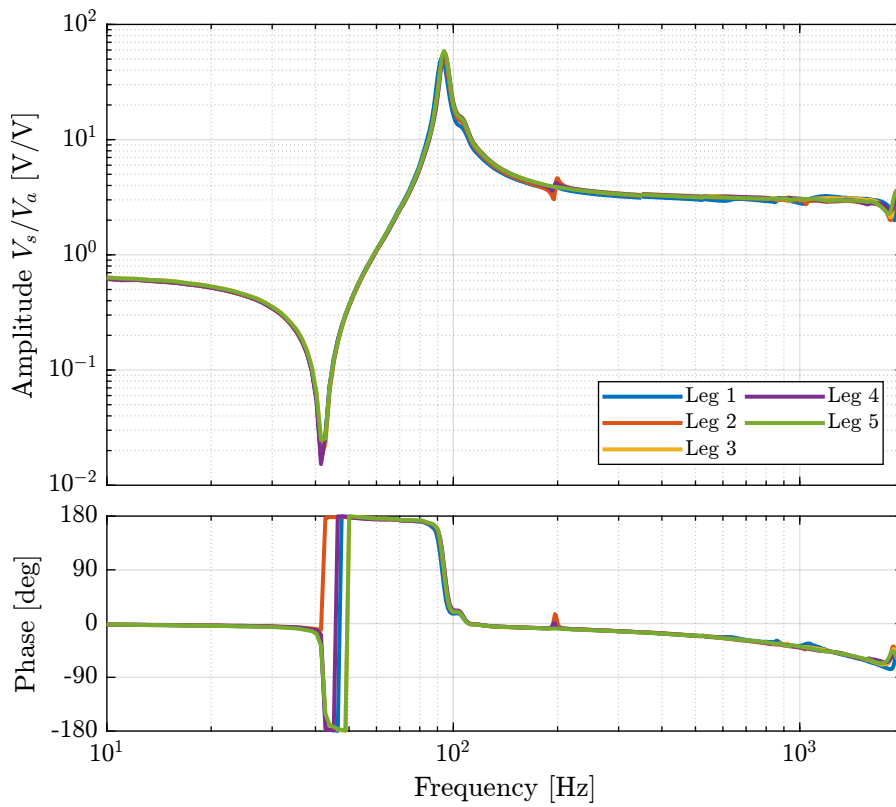


Figure 4.23: Identified IFF Plant

5 Test Bench APA300ML - Simscape Model

5.1 Introduction

5.2 Nano Hexapod object

```
n_hexapod = struct();
```

5.2.1 APA - 2 DoF

```
n_hexapod.actuator = struct();  
n_hexapod.actuator.type = 1;  
  
n_hexapod.actuator.k = ones(6,1)*0.35e6; % [N/m]  
n_hexapod.actuator.ke = ones(6,1)*1.5e6; % [N/m]  
n_hexapod.actuator.ka = ones(6,1)*43e6; % [N/m]  
  
n_hexapod.actuator.c = ones(6,1)*3e1; % [N/(m/s)]  
n_hexapod.actuator.ce = ones(6,1)*1e1; % [N/(m/s)]  
n_hexapod.actuator.ca = ones(6,1)*1e1; % [N/(m/s)]  
  
n_hexapod.actuator.Leq = ones(6,1)*0.056; % [m]  
  
n_hexapod.actuator.Ga = ones(6,1)*1; % Actuator gain [N/V]  
n_hexapod.actuator.Gs = ones(6,1)*1; % Sensor gain [V/m]
```

5.2.2 APA - Flexible Frame

```
n_hexapod.actuator.type = 2;  
  
n_hexapod.actuator.K = readmatrix('APA300ML_b_mat_K.CSV'); % Stiffness Matrix  
n_hexapod.actuator.M = readmatrix('APA300ML_b_mat_M.CSV'); % Mass Matrix  
n_hexapod.actuator.xi = 0.01; % Damping ratio  
n_hexapod.actuator.P = extractNodes('APA300ML_b_out_nodes_3D.txt'); % Node coordinates [m]  
  
n_hexapod.actuator.ks = 235e6; % Stiffness of one stack [N/m]  
n_hexapod.actuator.cs = 1e1; % Stiffness of one stack [N/m]  
  
n_hexapod.actuator.Ga = ones(6,1)*1; % Actuator gain [N/V]  
n_hexapod.actuator.Gs = ones(6,1)*1; % Sensor gain [V/m]
```

5.2.3 APA - Fully Flexible

```
Matlab
n_hexapod.actuator.type = 3;

n_hexapod.actuator.K = readmatrix('APA300ML_full_mat_K.CSV'); % Stiffness Matrix
n_hexapod.actuator.M = readmatrix('APA300ML_full_mat_M.CSV'); % Mass Matrix
n_hexapod.actuator.xi = 0.01; % Damping ratio
n_hexapod.actuator.P = extractNodes('APA300ML_full_out_nodes_3D.txt'); % Node coordiantes [m]

n_hexapod.actuator.Ga = ones(6,1)*1; % Actuator gain [N/V]
n_hexapod.actuator.Gs = ones(6,1)*1; % Sensor gain [V/m]
```

5.3 Identification

```
Matlab
%% Options for Linearized
options = linearizeOptions;
options.SampleTime = 0;

%% Name of the Simulink File
mdl = 'test_bench_apa300ml';

%% Input/Output definition
clear io; io_i = 1;
io(io_i) = linio([mdl, '/Va'], 1, 'openinput'); io_i = io_i + 1; % Actuator Voltage
io(io_i) = linio([mdl, '/Vs'], 1, 'openoutput'); io_i = io_i + 1; % Sensor Voltage
io(io_i) = linio([mdl, '/dL'], 1, 'openoutput'); io_i = io_i + 1; % Relative Motion Outputs
io(io_i) = linio([mdl, '/z'], 1, 'openoutput'); io_i = io_i + 1; % Vertical Motion

%% Run the linearization
Ga = linearize(mdl, io, 0.0, options);
Ga.InputName = {'Va'};
Ga.OutputName = {'Vs', 'dL', 'z'};
```

5.4 Compare 2-DoF with flexible

5.4.1 APA - 2 DoF

```
Matlab
n_hexapod = struct();

n_hexapod.actuator = struct();

n_hexapod.actuator.type = 1;

n_hexapod.actuator.k = ones(6,1)*0.35e6; % [N/m]
n_hexapod.actuator.ke = ones(6,1)*1.5e6; % [N/m]
n_hexapod.actuator.ka = ones(6,1)*43e6; % [N/m]

n_hexapod.actuator.c = ones(6,1)*3e1; % [N/(m/s)]
n_hexapod.actuator.ce = ones(6,1)*1e1; % [N/(m/s)]
n_hexapod.actuator.ca = ones(6,1)*1e1; % [N/(m/s)]

n_hexapod.actuator.Leq = ones(6,1)*0.056; % [m]

n_hexapod.actuator.Ga = ones(6,1)*-2.15; % Actuator gain [N/V]
n_hexapod.actuator.Gs = ones(6,1)*2.305e-08; % Sensor gain [V/m]
```

```
Matlab
G_2dof = linearize mdl, io, 0.0, options);
G_2dof.InputName = {'Va'};
G_2dof.OutputName = {'Vs', 'dL', 'z'};
```

5.4.2 APA - Fully Flexible

```
Matlab
n_hexapod = struct();
n_hexapod.actuator.type = 3;
n_hexapod.actuator.K = readmatrix('APA300ML_full_mat_K.CSV'); % Stiffness Matrix
n_hexapod.actuator.M = readmatrix('APA300ML_full_mat_M.CSV'); % Mass Matrix
n_hexapod.actuator.xi = 0.01; % Damping ratio
n_hexapod.actuator.P = extractNodes('APA300ML_full_out_nodes_3D.txt'); % Node coordinates [m]
n_hexapod.actuator.Ga = ones(6,1)*1; % Actuator gain [N/V]
n_hexapod.actuator.Gs = ones(6,1)*1; % Sensor gain [V/m]
```

```
Matlab
G_flex = linearize(mdl, io, 0.0, options);
G_flex.InputName = {'Va'};
G_flex.OutputName = {'Vs', 'dL', 'z'};
```

5.4.3 Comparison

6 Test Bench Struts - Simscape Model

6.1 Introduction

6.2 Nano Hexapod object

```
Matlab  
n_hexapod = struct();
```

6.2.1 Flexible Joint - Bot

```
Matlab  
n_hexapod.flex_bot = struct();  
  
n_hexapod.flex_bot.type = 1; % 1: 2dof / 2: 3dof / 3: 4dof  
  
n_hexapod.flex_bot.kRx = ones(6,1)*5; % X bending stiffness [Nm/rad]  
n_hexapod.flex_bot.kRy = ones(6,1)*5; % Y bending stiffness [Nm/rad]  
n_hexapod.flex_bot.kRz = ones(6,1)*260; % Torsionnal stiffness [Nm/rad]  
n_hexapod.flex_bot.kz = ones(6,1)*1e8; % Axial stiffness [N/m]  
  
n_hexapod.flex_bot.cRx = ones(6,1)*0.1; % [Nm/(rad/s)]  
n_hexapod.flex_bot.cRy = ones(6,1)*0.1; % [Nm/(rad/s)]  
n_hexapod.flex_bot.cRz = ones(6,1)*0.1; % [Nm/(rad/s)]  
n_hexapod.flex_bot.cz = ones(6,1)*1e2; % [N/(m/s)]
```

6.2.2 Flexible Joint - Top

```
Matlab  
n_hexapod.flex_top = struct();  
  
n_hexapod.flex_top.type = 2; % 1: 2dof / 2: 3dof / 3: 4dof  
  
n_hexapod.flex_top.kRx = ones(6,1)*5; % X bending stiffness [Nm/rad]  
n_hexapod.flex_top.kRy = ones(6,1)*5; % Y bending stiffness [Nm/rad]  
n_hexapod.flex_top.kRz = ones(6,1)*260; % Torsionnal stiffness [Nm/rad]  
n_hexapod.flex_top.kz = ones(6,1)*1e8; % Axial stiffness [N/m]  
  
n_hexapod.flex_top.cRx = ones(6,1)*0.1; % [Nm/(rad/s)]  
n_hexapod.flex_top.cRy = ones(6,1)*0.1; % [Nm/(rad/s)]  
n_hexapod.flex_top.cRz = ones(6,1)*0.1; % [Nm/(rad/s)]  
n_hexapod.flex_top.cz = ones(6,1)*1e2; % [N/(m/s)]
```


6.2.3 APA - 2 DoF

```
Matlab
n_hexapod.actuator = struct();
n_hexapod.actuator.type = 1;
n_hexapod.actuator.k = ones(6,1)*0.35e6; % [N/m]
n_hexapod.actuator.ke = ones(6,1)*1.5e6; % [N/m]
n_hexapod.actuator.ka = ones(6,1)*43e6; % [N/m]
n_hexapod.actuator.c = ones(6,1)*3e1; % [N/(m/s)]
n_hexapod.actuator.ce = ones(6,1)*1e1; % [N/(m/s)]
n_hexapod.actuator.ca = ones(6,1)*1e1; % [N/(m/s)]
n_hexapod.actuator.Leq = ones(6,1)*0.056; % [m]
n_hexapod.actuator.Ga = ones(6,1)*1; % Actuator gain [N/V]
n_hexapod.actuator.Gs = ones(6,1)*1; % Sensor gain [V/m]
```

6.2.4 APA - Flexible Frame

```
Matlab
n_hexapod.actuator.type = 2;
n_hexapod.actuator.K = readmatrix('APA300ML_b_mat_K.CSV'); % Stiffness Matrix
n_hexapod.actuator.M = readmatrix('APA300ML_b_mat_M.CSV'); % Mass Matrix
n_hexapod.actuator.xi = 0.01; % Damping ratio
n_hexapod.actuator.P = extractNodes('APA300ML_b_out_nodes_3D.txt'); % Node coordinates [m]
n_hexapod.actuator.ks = 235e6; % Stiffness of one stack [N/m]
n_hexapod.actuator.cs = 1e1; % Stiffness of one stack [N/m]
n_hexapod.actuator.Ga = ones(6,1)*1; % Actuator gain [N/V]
n_hexapod.actuator.Gs = ones(6,1)*1; % Sensor gain [V/m]
```

6.2.5 APA - Fully Flexible

```
Matlab
n_hexapod.actuator.type = 3;
n_hexapod.actuator.K = readmatrix('APA300ML_full_mat_K.CSV'); % Stiffness Matrix
n_hexapod.actuator.M = readmatrix('APA300ML_full_mat_M.CSV'); % Mass Matrix
n_hexapod.actuator.xi = 0.01; % Damping ratio
n_hexapod.actuator.P = extractNodes('APA300ML_full_out_nodes_3D.txt'); % Node coordinates [m]
n_hexapod.actuator.Ga = ones(6,1)*1; % Actuator gain [N/V]
n_hexapod.actuator.Gs = ones(6,1)*1; % Sensor gain [V/m]
```

6.3 Identification

```
Matlab
%% Options for Linearized
options = linearizeOptions;
```

```

options.SampleTime = 0;

%% Name of the Simulink File
mdl = 'test_bench_struts';

%% Input/Output definition
clear io; io_i = 1;
io(io_i) = linio([mdl, '/Va'], 1, 'openinput'); io_i = io_i + 1; % Actuator Voltage
io(io_i) = linio([mdl, '/Vs'], 1, 'openoutput'); io_i = io_i + 1; % Sensor Voltage
io(io_i) = linio([mdl, '/dL'], 1, 'openoutput'); io_i = io_i + 1; % Relative Motion Outputs
io(io_i) = linio([mdl, '/z'], 1, 'openoutput'); io_i = io_i + 1; % Vertical Motion

%% Run the linearization
Gs = linearize(mdl, io, 0.0, options);
Gs.InputName = {'Va'};
Gs.OutputName = {'Vs', 'dL', 'z'};

```

6.4 Compare flexible joints

6.4.1 Perfect

```

n_hexapod.flex_bot.type = 1; % 1: 2dof / 2: 3dof / 3: 4dof
n_hexapod.flex_top.type = 2; % 1: 2dof / 2: 3dof / 3: 4dof

```

```

Gp = linearize(mdl, io, 0.0, options);
Gp.InputName = {'Va'};
Gp.OutputName = {'Vs', 'dL', 'z'};

```

6.4.2 Top Flexible

```

n_hexapod.flex_bot.type = 1; % 1: 2dof / 2: 3dof / 3: 4dof
n_hexapod.flex_top.type = 3; % 1: 2dof / 2: 3dof / 3: 4dof

```

```

Gt = linearize(mdl, io, 0.0, options);
Gt.InputName = {'Va'};
Gt.OutputName = {'Vs', 'dL', 'z'};

```

6.4.3 Bottom Flexible

```

n_hexapod.flex_bot.type = 3; % 1: 2dof / 2: 3dof / 3: 4dof
n_hexapod.flex_top.type = 2; % 1: 2dof / 2: 3dof / 3: 4dof

```

```
Matlab
Gb = linearize mdl, io, 0.0, options);
Gb.InputName = {'Va'};
Gb.OutputName = {'Vs', 'dL', 'z'};
```

6.4.4 Both Flexible

```
Matlab
n_hexapod.flex_bot.type = 3; % 1: 2dof / 2: 3dof / 3: 4dof
n_hexapod.flex_top.type = 3; % 1: 2dof / 2: 3dof / 3: 4dof
```

```
Matlab
Gf = linearize(mdl, io, 0.0, options);
Gf.InputName = {'Va'};
Gf.OutputName = {'Vs', 'dL', 'z'};
```

6.4.5 Comparison

7 Function

7.1 generateSweepExc: Generate sweep sinus excitation

Function description

```
----- Matlab -----  
function [U_exc] = generateSweepExc(args)  
% generateSweepExc - Generate a Sweep Sine excitation signal  
%  
% Syntax: [U_exc] = generateSweepExc(args)  
%  
% Inputs:  
% - args - Optimal arguments:  
%   - Ts           - Sampling Time                - [s]  
%   - f_start      - Start frequency of the sweep  - [Hz]  
%   - f_end        - End frequency of the sweep    - [Hz]  
%   - V_mean       - Mean value of the excitation voltage - [V]  
%   - V_exc        - Excitation Amplitude for the Sweep, could be numeric or TF - [V]  
%   - t_start      - Time at which the sweep begins - [s]  
%   - exc_duration - Duration of the sweep        - [s]  
%   - sweep_type   - 'logarithmic' or 'linear'     - [-]  
%   - smooth_ends  - 'true' or 'false': smooth transition between 0 and V_mean - [-]
```

Optional Parameters

```
----- Matlab -----  
arguments  
args.Ts           (1,1) double {mustBeNumeric, mustBePositive} = 1e-4  
args.f_start      (1,1) double {mustBeNumeric, mustBePositive} = 1  
args.f_end        (1,1) double {mustBeNumeric, mustBePositive} = 1e3  
args.V_mean       (1,1) double {mustBeNumeric} = 0  
args.V_exc        = 1  
args.t_start      (1,1) double {mustBeNumeric, mustBeNonnegative} = 5  
args.exc_duration (1,1) double {mustBeNumeric, mustBePositive} = 10  
args.sweep_type   char {mustBeMember(args.sweep_type,{'log', 'lin'})} = 'lin'  
args.smooth_ends  logical {mustBeNumericOrLogical} = true  
end
```

Sweep Sine part

```
----- Matlab -----  
t_sweep = 0:args.Ts:args.exc_duration;  
  
if strcmp(args.sweep_type, 'log')  
    V_exc = sin(2*pi*args.f_start * args.exc_duration/log(args.f_end/args.f_start) *  
↳ (exp(log(args.f_end/args.f_start)*t_sweep/args.exc_duration) - 1));  
elseif strcmp(args.sweep_type, 'lin')
```

```

V_exc = sin(2*pi*(args.f_start + (args.f_end - args.f_start)/2/args.exc_duration*t_sweep).*t_sweep);
else
error('sweep_type should either be equal to "log" or to "lin"');
end

```

```

----- Matlab -----
if isnumeric(args.V_exc)
V_sweep = args.V_mean + args.V_exc*t_sweep;
elseif isct(args.V_exc)
if strcmp(args.sweep_type, 'log')
V_sweep = args.V_mean + abs(squeeze(freqresp(args.V_exc,
→ args.f_start*(args.f_end/args.f_start).^(t_sweep/args.exc_duration), 'Hz'))'.*V_exc;
elseif strcmp(args.sweep_type, 'lin')
V_sweep = args.V_mean + abs(squeeze(freqresp(args.V_exc,
→ args.f_start+(args.f_end-args.f_start)/args.exc_duration*t_sweep, 'Hz'))'.*V_exc;
end
end
end

```

Smooth Ends

```

----- Matlab -----
if args.t_start > 0
t_smooth_start = args.Ts:args.Ts:args.t_start;

V_smooth_start = zeros(size(t_smooth_start));
V_smooth_end = zeros(size(t_smooth_start));

if args.smooth_ends
Vd_max = args.V_mean/(0.7*args.t_start);

V_d = zeros(size(t_smooth_start));
V_d(t_smooth_start < 0.2*args.t_start) = t_smooth_start(t_smooth_start < 0.2*args.t_start)*Vd_max/(0.2*args.t_start);
V_d(t_smooth_start > 0.2*args.t_start & t_smooth_start < 0.7*args.t_start) = Vd_max;
V_d(t_smooth_start > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start >
→ 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) - 0.7*args.t_start)*Vd_max/(0.2*args.t_start);

V_smooth_start = cumtrapz(V_d)*args.Ts;

V_smooth_end = args.V_mean - V_smooth_start;
end
else
V_smooth_start = [];
V_smooth_end = [];
end
end

```

Combine Excitation signals

```

----- Matlab -----
V_exc = [V_smooth_start, V_sweep, V_smooth_end];
t_exc = args.Ts*[0:1:length(V_exc)-1];

```

```

----- Matlab -----
U_exc = [t_exc; V_exc];

```

7.2 generateShapedNoise: Generate Shaped Noise excitation

Function description

```
Matlab
function [U_exc] = generateShapedNoise(args)
% generateShapedNoise - Generate a Shaped Noise excitation signal
%
% Syntax: [U_exc] = generateShapedNoise(args)
%
% Inputs:
%   - args - Optinal arguments:
%     - Ts           - Sampling Time           - [s]
%     - V_mean       - Mean value of the excitation voltage - [V]
%     - V_exc        - Excitation Amplitude, could be numeric or TF - [V rms]
%     - t_start      - Time at which the noise begins - [s]
%     - exc_duration - Duration of the noise - [s]
%     - smooth_ends  - 'true' or 'false': smooth transition between 0 and V_mean - [-]
```

Optional Parameters

```
Matlab
arguments
args.Ts           (1,1) double {mustBeNumeric, mustBePositive} = 1e-4
args.V_mean       (1,1) double {mustBeNumeric} = 0
args.V_exc        = 1
args.t_start      (1,1) double {mustBeNumeric, mustBePositive} = 5
args.exc_duration (1,1) double {mustBeNumeric, mustBePositive} = 10
args.smooth_ends  logical {mustBeNumericOrLogical} = true
end
```

Shaped Noise

```
Matlab
t_noise = 0:args.Ts:args.exc_duration;
```

```
Matlab
if isnumeric(args.V_exc)
    V_noise = args.V_mean + args.V_exc*sqrt(1/args.Ts/2)*randn(length(t_noise), 1)';
elseif isct(args.V_exc)
    V_noise = args.V_mean + lsim(args.V_exc, sqrt(1/args.Ts/2)*randn(length(t_noise), 1), t_noise)';
end
```

Smooth Ends

```
Matlab
t_smooth_start = args.Ts:args.Ts:args.t_start;
V_smooth_start = zeros(size(t_smooth_start));
V_smooth_end   = zeros(size(t_smooth_start));
```

```

if args.smooth_ends
    Vd_max = args.V_mean/(0.7*args.t_start);

    V_d = zeros(size(t_smooth_start));
    V_d(t_smooth_start < 0.2*args.t_start) = t_smooth_start(t_smooth_start < 0.2*args.t_start)*Vd_max/(0.2*args.t_start);
    V_d(t_smooth_start > 0.2*args.t_start & t_smooth_start < 0.7*args.t_start) = Vd_max;
    V_d(t_smooth_start > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start >
    ↪ 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) - 0.7*args.t_start)*Vd_max/(0.2*args.t_start);

    V_smooth_start = cumtrapz(V_d)*args.Ts;

    V_smooth_end = args.V_mean - V_smooth_start;
end

```

Combine Excitation signals

```

V_exc = [V_smooth_start, V_noise, V_smooth_end];
t_exc = args.Ts*[0:1:length(V_exc)-1];

```

```

U_exc = [t_exc; V_exc];

```

7.3 generateSinIncreasingAmpl: Generate Sinus with increasing amplitude

Function description

```

function [U_exc] = generateSinIncreasingAmpl(args)
% generateSinIncreasingAmpl - Generate Sinus with increasing amplitude
%
% Syntax: [U_exc] = generateSinIncreasingAmpl(args)
%
% Inputs:
%   - args - Optinal arguments:
%     - Ts           - Sampling Time           - [s]
%     - V_mean       - Mean value of the excitation voltage - [V]
%     - sin_ampls    - Excitation Amplitudes    - [V]
%     - sin_freq     - Excitation Frequency     - [Hz]
%     - sin_num      - Number of period for each amplitude - [-]
%     - t_start      - Time at which the excitation begins - [s]
%     - smooth_ends  - 'true' or 'false': smooth transition between 0 and V_mean - [-]

```

Optional Parameters

```

arguments
args.Ts           (1,1) double {mustBeNumeric, mustBePositive} = 1e-4

```

```

args.V_mean      (1,1) double {mustBeNumeric} = 0
args.sin_ampls   double {mustBeNumeric, mustBePositive} = [0.1, 0.2, 0.3]
args.sin_period  (1,1) double {mustBeNumeric, mustBePositive} = 1
args.sin_num     (1,1) double {mustBeNumeric, mustBePositive, mustBeInteger} = 3
args.t_start     (1,1) double {mustBeNumeric, mustBePositive} = 5
args.smooth_ends logical {mustBeNumericOrLogical} = true
end

```

Sinus excitation

```

----- Matlab -----
t_noise = 0:args.Ts:args.sin_period*args.sin_num;
sin_exc = [];

```

```

----- Matlab -----
for sin_ampl = args.sin_ampls
    sin_exc = [sin_exc, args.V_mean + sin_ampl*sin(2*pi/args.sin_period*t_noise)];
end

```

Smooth Ends

```

----- Matlab -----
t_smooth_start = args.Ts:args.Ts:args.t_start;

V_smooth_start = zeros(size(t_smooth_start));
V_smooth_end   = zeros(size(t_smooth_start));

if args.smooth_ends
    Vd_max = args.V_mean/(0.7*args.t_start);

    V_d = zeros(size(t_smooth_start));
    V_d(t_smooth_start < 0.2*args.t_start) = t_smooth_start(t_smooth_start < 0.2*args.t_start)*Vd_max/(0.2*args.t_start);
    V_d(t_smooth_start > 0.2*args.t_start & t_smooth_start < 0.7*args.t_start) = Vd_max;
    V_d(t_smooth_start > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start >
→ 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) - 0.7*args.t_start)*Vd_max/(0.2*args.t_start);

    V_smooth_start = cumtrapz(V_d)*args.Ts;

    V_smooth_end = args.V_mean - V_smooth_start;
end

```

Combine Excitation signals

```

----- Matlab -----
V_exc = [V_smooth_start, sin_exc, V_smooth_end];
t_exc = args.Ts*[0:1:length(V_exc)-1];

```

```

----- Matlab -----
U_exc = [t_exc; V_exc];

```


Bibliography

- [1] Gerrit Wijnand van der Poel. “An Exploration of Active Hard Mount Vibration Isolation for Precision Equipment”. PhD thesis. University of Twente, 2010. ISBN: 978-90-365-3016-3. DOI: [10.3990/1.9789036530163](https://doi.org/10.3990/1.9789036530163). URL: <https://doi.org/10.3990/1.9789036530163>.