Amplifier Piezoelectric Actuator APA300ML - Test Bench

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June 14, 2021

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The goal of this test bench is to extract all the important parameters of the Amplified Piezoelectric Actuator APA300ML.

This include:

- \bullet Stroke
- Stiffness
- Hysteresis
- \bullet Gain from the applied voltage V_a to the generated Force F_a
- \bullet Gain from the sensor stack strain δL to the generated voltage V_s
- Dynamical behavior

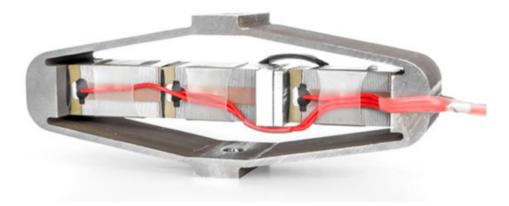


Figure 0.1: Picture of the APA300ML

1 Model of an Amplified Piezoelectric Actuator and Sensor

Consider a schematic of the Amplified Piezoelectric Actuator in Figure 1.1.

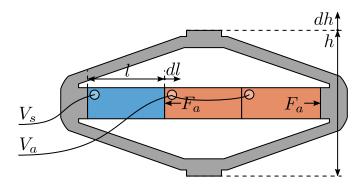


Figure 1.1: Amplified Piezoelectric Actuator Schematic

A voltage V_a applied to the actuator stacks will induce an actuator force F_a :

$$F_a = g_a \cdot V_a \tag{1.1}$$

A change of length dl of the sensor stack will induce a voltage V_s :

$$V_s = g_s \cdot dl \tag{1.2}$$

We wish here to experimental measure g_a and g_s .

The block-diagram model of the piezoelectric actuator is then as shown in Figure 1.2.

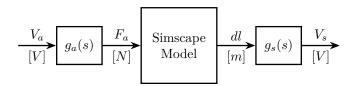


Figure 1.2: Model of the APA with Simscape/Simulink

2 First Basic Measurements

- Section 2.1:
- Section 2.2:
- Section 2.3:
- Section 2.4:

2.1 Geometrical Measurements

The received APA are shown in Figure 2.1.

2.1.1 Measurement Setup

The flatness corresponding to the two interface planes are measured as shown in Figure 2.2.

2.1.2 Measurement Results

The height (Z) measurements at the 8 locations (4 points by plane) are defined below.

```
apa1 = 1e-6*[0, -0.5 , 3.5 , 3.5 , 42 , 45.5, 52.5 , 46];
apa2 = 1e-6*[0, -2.5 , -3 , 0 , -1.5 , 1 , -2 , -4];
apa3 = 1e-6*[0, -1.5 , 15 , 17.5 , 6.5 , 6.5 , 21 , 23];
apa4 = 1e-6*[0, 6.5 , 14.5 , 9 , 16 , 22 , 29.5 , 21];
apa5 = 1e-6*[0, -12.5, 16.5 , 28.5 , -43 , -52 , -22.5, -13.5];
apa6 = 1e-6*[0, -8 , -2 , 5 , -57.5, -62 , -55.5 , -52.5];
apa7 = 1e-6*[0, 19.5 , -8 , -29.5, 75 , 97.5, 70 , 48];
apa7b = 1e-6*[0, 9 , -18.5, -30 , 31 , 46.5, 16.5 , 7.5];
apa = {apa1, apa2, apa3, apa4, apa5, apa6, apa7b};
```

The X/Y Positions of the 8 measurement points are defined below.

```
W = 20e-3; % Width [m]
L = 61e-3; % Length [m]
d = 1e-3; % Distance from border [m]
1 = 15.5e-3; % [m]

pos = [[-L/2 + d; W/2 - d], [-L/2 + 1 - d; W/2 - d], [-L/2 + 1 - d; -W/2 + d], [-L/2 + d; -W/2 + d], [L/2 - 1 + d; W/2 - d],

∴ [L/2 - d; W/2 - d], [L/2 - d; -W/2 + d], [L/2 - 1 + d; -W/2 + d]];
```

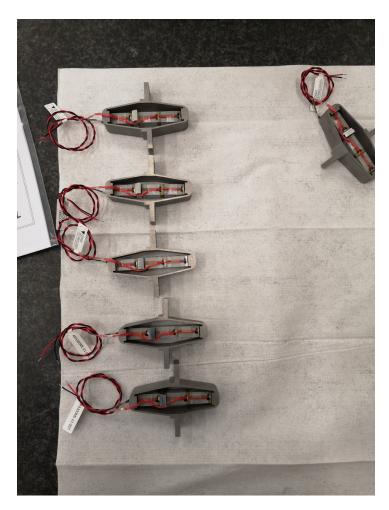


Figure 2.1: Received APA

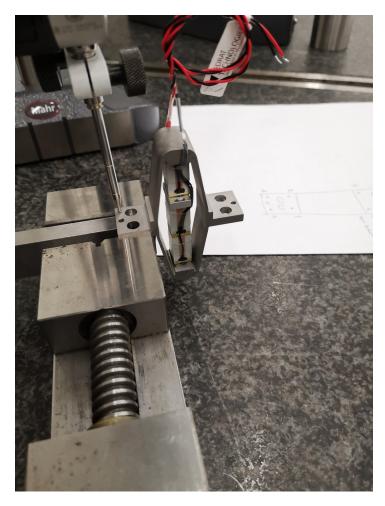


Figure 2.2: Measurement Setup

Finally, the flatness is estimated by fitting a plane through the 8 points using the fminsearch command.

```
apa_d = zeros(1, 7);
for i = 1:7
    fun = @(x)max(abs(([pos; apa{i}]-[0;0;x(1)])'*([x(2:3);1]/norm([x(2:3);1]))));
    x0 = [0;0;0];
    [x, min_d] = fminsearch(fun,x0);
    apa_d(i) = min_d;
end
```

The obtained flatness are shown in Table 2.1.

Table 2.1: Estimated flatness

	Flatness $[\mu m]$
APA 1	8.9
APA 2	3.1
APA 3	9.1
APA 4	3.0
APA 5	1.9
APA 6	7.1
APA 7	18.7

2.2 Electrical Measurements

Note

The capacitance of the stacks is measure with the LCR-800 Meter (doc)

The excitation frequency is set to be 1kHz.

Table 2.2: Capacitance measured with the LCR meter. The excitation signal is a sinus at 1kHz

	Sensor Stack	Actuator Stacks
APA 1	5.10	10.03
APA 2	4.99	9.85
APA 3	1.72	5.18
APA 4	4.94	9.82
APA5	4.90	9.66
APA 6	4.99	9.91
APA 7	4.85	9.85

Warning

There is clearly a problem with APA300ML number 3

The APA number 3 has ben sent back to Cedrat, and a new APA300ML has been shipped back.



Figure 2.3: LCR Meter used for the measurements

2.3 Stroke measurement

We here wish to estimate the stroke of the APA.

To do so, one side of the APA is fixed, and a displacement probe is located on the other side as shown in Figure 2.4.

Then, a voltage is applied on either one or two stacks using a DAC and a voltage amplifier.

Note

Here are the documentation of the equipment used for this test bench:

• Voltage Amplifier: PD200 with a gain of 20

• 16bits DAC: IO313 Speedgoat card

• Displacement Probe: Millimar C1216 electronics and Millimar 1318 probe



Figure 2.4: Bench to measured the APA stroke

2.3.1 Voltage applied on one stack

Let's first look at the relation between the voltage applied to **one** stack to the displacement of the APA as measured by the displacement probe.

The applied voltage is shown in Figure 2.5.

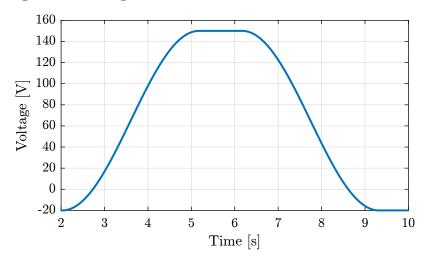


Figure 2.5: Applied voltage as a function of time

The obtained displacement is shown in Figure 2.6. The displacement is set to zero at initial time when the voltage applied is -20V.

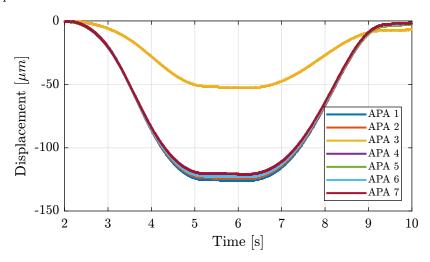


Figure 2.6: Displacement as a function of time for all the APA300ML

Finally, the displacement is shown as a function of the applied voltage in Figure 2.7. We can clearly see that there is a problem with the APA 3. Also, there is a large hysteresis.

Important

We can clearly see from Figure 2.7 that there is a problem with the APA number 3.

2.3.2 Voltage applied on two stacks

Now look at the relation between the voltage applied to the **two** other stacks to the displacement of the APA as measured by the displacement probe.

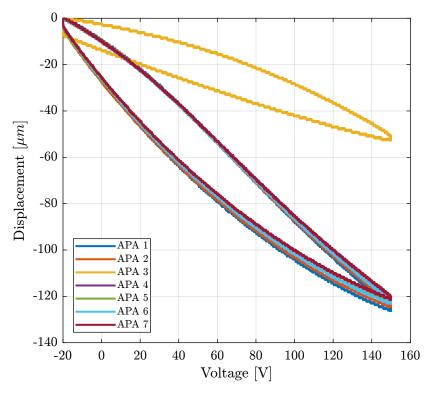


Figure 2.7: Displacement as a function of the applied voltage

The obtained displacement is shown in Figure 2.8. The displacement is set to zero at initial time when the voltage applied is -20V.

Finally, the displacement is shown as a function of the applied voltage in Figure 2.9. We can clearly see that there is a problem with the APA 3. Also, there is a large hysteresis.

2.3.3 Voltage applied on all three stacks

Finally, we can combine the two measurements to estimate the relation between the displacement and the voltage applied to the **three** stacks (Figure 2.10).

The obtained maximum stroke for all the APA are summarized in Table 2.3.

Table 2.3: Measured maximum stroke

	Stroke $[\mu m]$
APA 1	373.2
APA 2	365.5
APA 3	181.7
APA 4	359.7
APA5	361.5
APA 6	363.9
APA 7	358.4

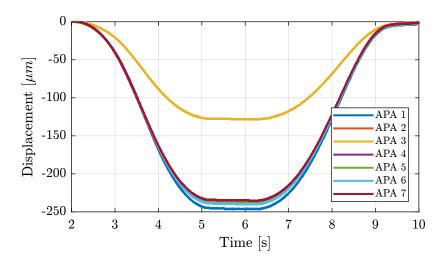


Figure 2.8: Displacement as a function of time for all the APA300ML

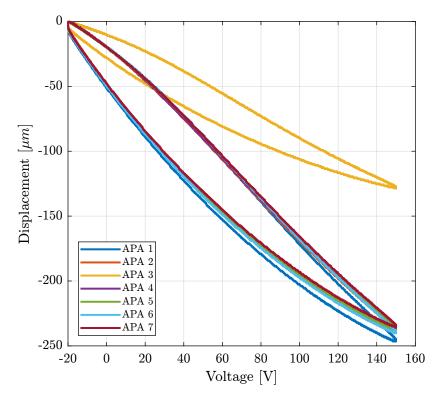


Figure 2.9: Displacement as a function of the applied voltage

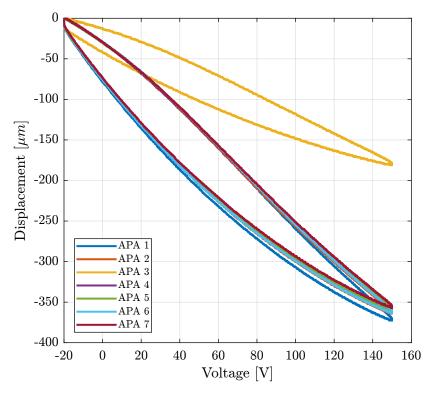


Figure 2.10: Displacement as a function of the applied voltage

2.4 Spurious resonances

2.4.1 Introduction

Three main resonances are foreseen to be problematic for the control of the APA300ML (Figure 2.11):

- Mode in X-bending at 189Hz
- Mode in Y-bending at 285Hz
- Mode in Z-torsion at 400Hz

These modes are present when flexible joints are fixed to the ends of the APA300ML.

In this section, we try to find the resonance frequency of these modes when one end of the APA is fixed and the other is free.

2.4.2 Setup

The measurement setup is shown in Figure 2.12. A Laser vibrometer is measuring the difference of motion of two points. The APA is excited with an instrumented hammer and the transfer function from the hammer to the measured rotation is computed.

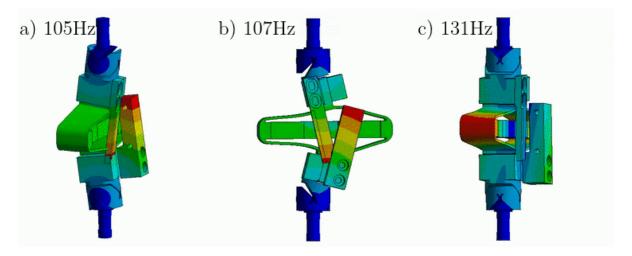


Figure 2.11: Spurious resonances. a) X-bending mode at 189Hz. b) Y-bending mode at 285Hz. c) Z-torsion mode at 400Hz

Note

- Laser Doppler Vibrometer Polytec OFV512
- Instrumented hammer



Figure 2.12: Measurement setup with a Laser Doppler Vibrometer and one instrumental hammer

2.4.3 Bending - X

The setup to measure the X-bending motion is shown in Figure 2.13. The APA is excited with an instrumented hammer having a solid metallic tip. The impact point is on the back-side of the APA aligned with the top measurement point.

The data is loaded.

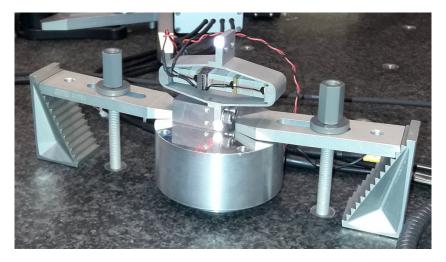


Figure 2.13: X-Bending measurement setup

The config for tfestimate is performed:

```
Ts = bending_X.Track1_X_Resolution; % Sampling frequency [Hz]
win = hann(ceil(1/Ts));
```

The transfer function from the input force to the output "rotation" (difference between the two measured distances).

```
[G_bending_X, f] = tfestimate(bending_X.Track1, bending_X.Track2, win, [], [], 1/Ts);
```

The result is shown in Figure 2.14.

The can clearly observe a nice peak at 280Hz, and then peaks at the odd "harmonics" (third "harmonic" at 840Hz, and fifth "harmonic" at 1400Hz).

2.4.4 Bending - Y

The setup to measure the Y-bending is shown in Figure 2.15.

The impact point of the instrumented hammer is located on the back surface of the top interface (on the back of the 2 measurements points).

The data is loaded, and the transfer function from the force to the measured rotation is computed.

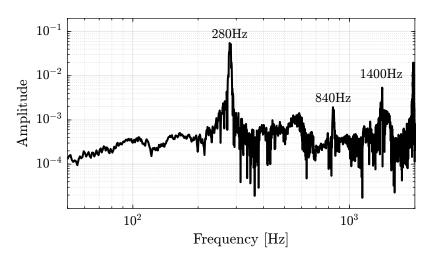


Figure 2.14: Obtained FRF for the X-bending

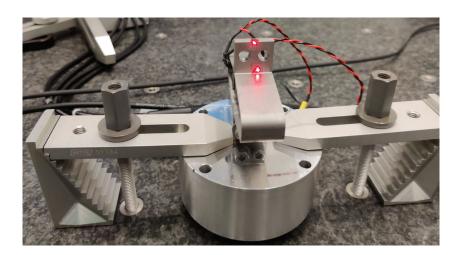


Figure 2.15: Y-Bending measurement setup

The results are shown in Figure 2.16. The main resonance is at 412Hz, and we also see the third "harmonic" at 1220Hz.

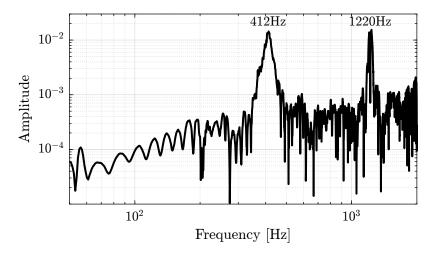


Figure 2.16: Obtained FRF for the Y-bending

2.4.5 Torsion - Z

Finally, we measure the Z-torsion resonance as shown in Figure 2.17.

The excitation is shown on the other side of the APA, on the side to excite the torsion motion.



Figure 2.17: Z-Torsion measurement setup

The data is loaded, and the transfer function computed.

```
torsion = load('apa300ml_torsion_left.mat');
[G_torsion, ~] = tfestimate(torsion.Track1, torsion.Track2, win, [], [], 1/Ts);
```

The results are shown in Figure 2.18. We observe a first peak at 267Hz, which corresponds to the X-bending mode that was measured at 280Hz. And then a second peak at 415Hz, which corresponds to the X-bending mode that was measured at 412Hz. The mode in pure torsion is probably at higher frequency (peak around 1kHz?).

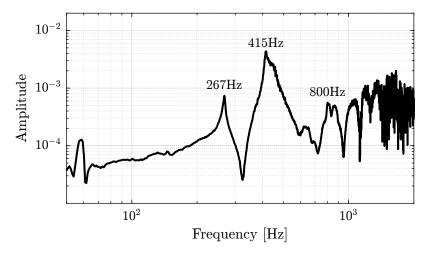


Figure 2.18: Obtained FRF for the Z-torsion

In order to verify that, the APA is excited on the top part such that the torsion mode should not be excited.

```
torsion = load('apa300ml_torsion_top.mat');
[G_torsion_top, ~] = tfestimate(torsion.Track1, torsion.Track2, win, [], [], 1/Ts);
```

The two FRF are compared in Figure 2.19. It is clear that the first two modes does not correspond to the torsional mode. Maybe the resonance at 800Hz, or even higher resonances. It is difficult to conclude here.

2.4.6 Compare

The three measurements are shown in Figure 2.20.

2.4.7 Conclusion

When two flexible joints are fixed at each ends of the APA, the APA is mostly in a free/free condition in terms of bending/torsion (the bending/torsional stiffness of the joints being very small).

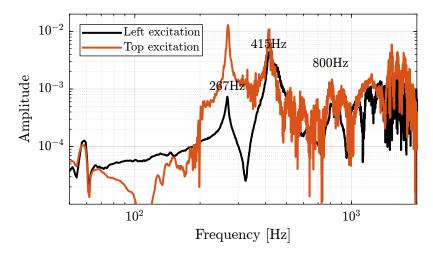


Figure 2.19: Obtained FRF for the Z-torsion

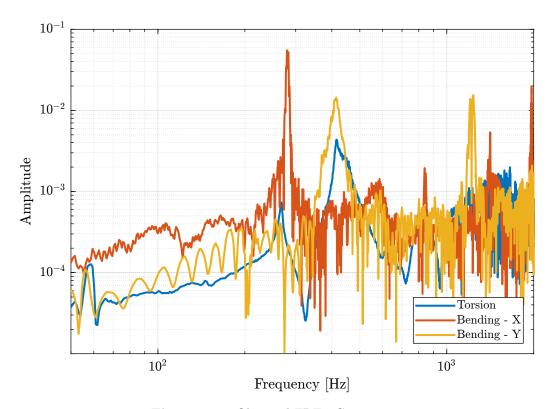


Figure 2.20: Obtained FRF - Comparison

In the current tests, the APA are in a fixed/free condition. Therefore, it is quite obvious that we measured higher resonance frequencies than what is foreseen for the struts. It is however quite interesting that there is a factor $\approx \sqrt{2}$ between the two (increased of the stiffness by a factor 2?).

Table 2.4: Measured frequency of the modes

Mode	Strut Mode	Measured Frequency	
X-Bending	189 Hz	$280 \mathrm{Hz}$	
Y-Bending	285 Hz	$410 \mathrm{Hz}$	
Z-Torsion	$400 \mathrm{Hz}$?	

3 Dynamical measurements - APA

In this section, a measurement test bench is used to identify the dynamics of the APA.

The bench is shown in Figure 3.1, and a zoom picture on the APA and encoder is shown in Figure 3.2.

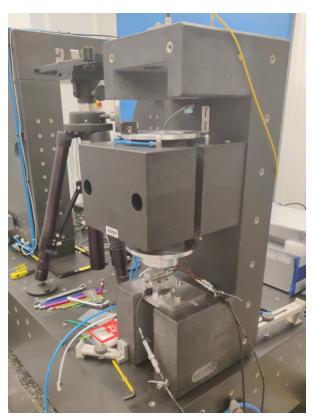


Figure 3.1: Picture of the test bench

Note

Here are the documentation of the equipment used for this test bench:

• Voltage Amplifier: PD200

• Amplified Piezoelectric Actuator: APA300ML

• DAC/ADC: Speedgoat IO313

• Encoder: Renishaw Vionic and used Ruler

• Interferometer: Attocube IDS3010

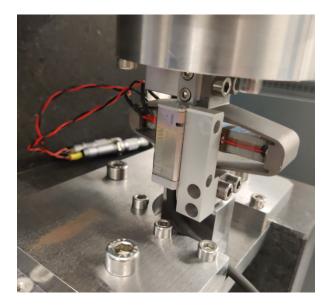


Figure 3.2: Zoom on the APA with the encoder

The bench is schematically shown in Figure 3.3 and the signal used are summarized in Table 3.1.

Table 3.1: Variables used during the measurements

Variable	Description	Unit	Hardware
Va	Output DAC voltage	[V]	DAC - Ch. 1 => PD200 => APA
Vs	Measured stack voltage (ADC)	[V]	APA => ADC - Ch .
de	Encoder Measure- ment	[m]	PEPU Ch. 1 - IO318(1) - Ch. 1
da	Attocube Measure- ment	[m]	PEPU Ch. 2 - IO318(1) - Ch. 2
t	Time	[s]	()

This section is structured as follows:

- Section 3.1: the Speedgoat setup is described (excitation signals, saved signals, etc.)
- Section 3.2: the measurements are first performed on one APA.
- \bullet Section 3.3: the same measurements are performed on all the APA and are compared.

3.1 Speedgoat Setup

3.1.1 frf_setup.m - Measurement Setup

First is defined the sampling frequency:

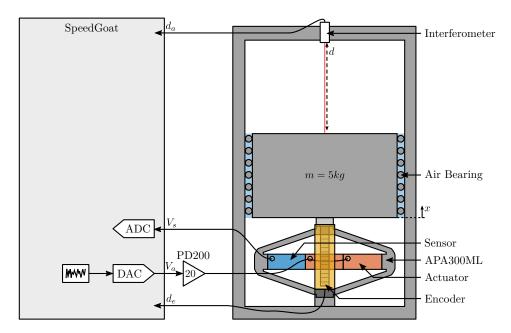


Figure 3.3: Schematic of the Test Bench

```
%% Simulation configuration
Fs = 10e3; % Sampling Frequency [Hz]
Ts = 1/Fs; % Sampling Time [s]

%% Data record configuration
Trec_start = 5; % Start time for Recording [s]
Trec_dur = 100; % Recording Duration [s]

### Matlab

Tsim = 2*Trec_start + Trec_dur; % Simulation Time [s]
```

A white noise excitation signal can be very useful in order to obtain a first idea of the plant FRF. The gain can be gradually increased until satisfactory output is obtained.

The maximum excitation voltage at resonance is 9Vrms, therefore corresponding to 0.6V of output DAC voltage.

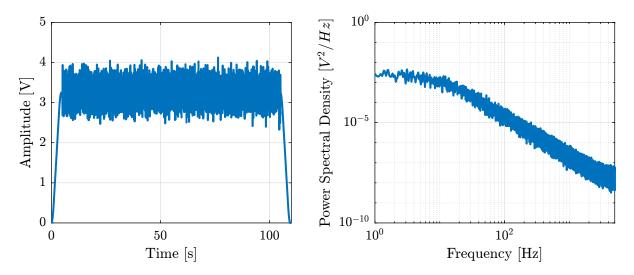


Figure 3.4: Example of Shaped noise excitation signal

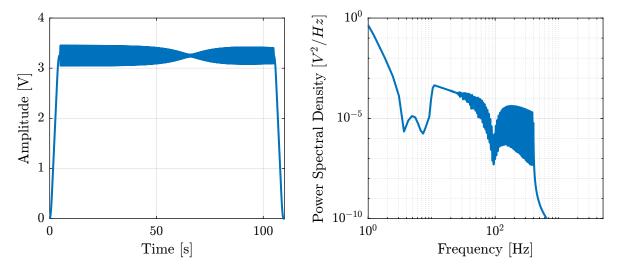


Figure 3.5: Example of Sweep Sin excitation signal

In order to better estimate the high frequency dynamics, a band-limited noise can be used (Figure 3.6).

The frequency content of the noise can be precisely controlled.

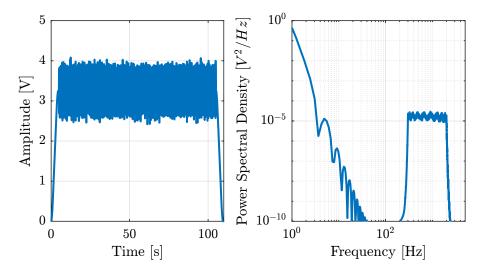


Figure 3.6: Example of band-limited noise excitation signal

Then a sinus excitation can be used to estimate the hysteresis.

Then, we select the wanted excitation signal.

```
%% Select the excitation signal
V_exc = timeseries(V_noise(2,:), V_noise(1,:));
```

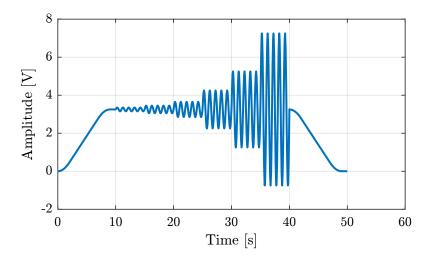


Figure 3.7: Example of Shaped noise excitation signal

3.1.2 frf_save.m - Save Data

First, we get data from the Speedgoat:

```
tg = slrt;

f = SimulinkRealTime.openFTP(tg);
mget(f, 'data/data.dat');
close(f);
```

And we load the data on the Workspace:

```
data = SimulinkRealTime.utils.getFileScopeData('data/data.dat').data;
da = data(:, 1); % Excitation Voltage (input of PD200) [V]
de = data(:, 2); % Measured voltage (force sensor) [V]
Vs = data(:, 3); % Measurement displacement (encoder) [m]
Va = data(:, 4); % Measurement displacement (attocube) [m]
t = data(:, end); % Time [s]
```

And we save this to a mat file:

```
apa_number = 1;
save(sprintf('mat/frf_data_%i_huddle.mat', apa_number), 't', 'Va', 'Vs', 'de', 'da');
```

3.2 Measurements on APA 1

Measurements are first performed on only **one** APA. Once the measurement procedure is validated, it is performed on all the other APA.

3.2.1 Excitation Signal

For this first measurement, a basic logarithmic sweep is used between 10Hz and 2kHz.

The data are loaded.

```
_____Matlab ____apa_sweep = load(sprintf('mat/frf_data_%i_sweep.mat', 1), 't', 'Va', 'Vs', 'da', 'de');
```

The initial time is set to zero.

```
## Matlab ## Matlab ## Time vector
t = apa_sweep.t - apa_sweep.t(1); % Time vector [s]
```

The excitation signal is shown in Figure 3.8. It is a sweep sine from 10Hz up to 2kHz filtered with a notch centered with the main resonance of the system and a low pass filter.

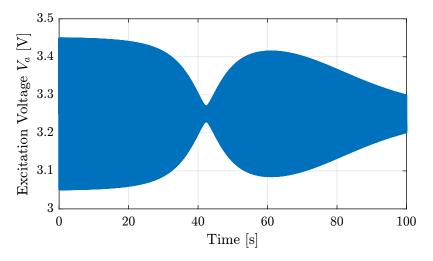


Figure 3.8: Excitation voltage

3.2.2 FRF Identification - Setup

Let's define the sampling time/frequency.

```
%% Sampling
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a "Hanning" windows that will be used for the spectral analysis:

```
win = hanning(ceil(1*Fs)); % Hannning Windows
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```
% Only used to have the frequency vector "f" [~, f] = tfestimate(apa_sweep.Va, apa_sweep.de, win, [], [], 1/Ts);
```

3.2.3 FRF Identification - Displacement

In this section, the transfer function from the excitation voltage V_a to the encoder measured displacement d_e and interferometer measurement d_a .

The coherence from V_a to d_e is computed and shown in Figure 3.9. It is quite good from 10Hz up to 500Hz.

```
Matlab

%% TF - Encoder
[coh_sweep, ~] = mscohere(apa_sweep.Va, apa_sweep.de, win, [], [], 1/Ts);
```

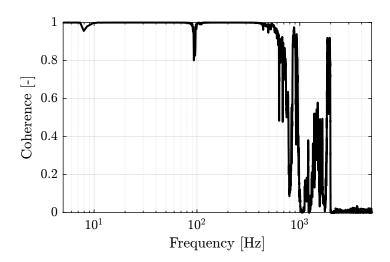


Figure 3.9: Coherence for the identification from V_a to d_e

The transfer functions are then estimated and shown in Figure 3.10.

```
%% TF - Encoder
[dvf_sweep, ~] = tfestimate(apa_sweep.Va, apa_sweep.de, win, [], [], 1/Ts);

%% TF - Interferometer
[int_sweep, ~] = tfestimate(apa_sweep.Va, apa_sweep.da, win, [], [], 1/Ts);
```

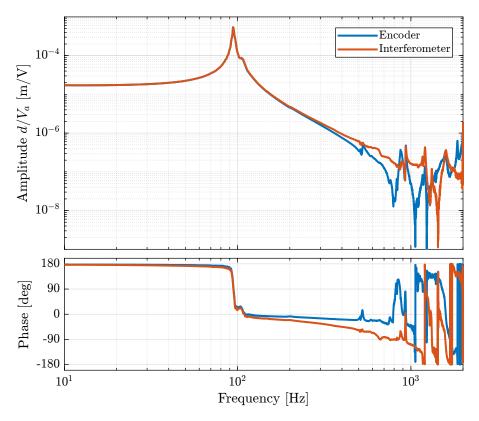


Figure 3.10: Obtained transfer functions from V_a to both d_e and d_a

Important

It seems that using the interferometer, we have a lot more time delay than when using the encoder.

3.2.4 FRF Identification - Force Sensor

Now the dynamics from excitation voltage V_a to the force sensor stack voltage V_s is identified.

The coherence is computed and shown in Figure 3.11 and found very good from $10\mathrm{Hz}$ up to $2\mathrm{kHz}$.

```
%% TF - Encoder
[coh_sweep, ~] = mscohere(apa_sweep.Va, apa_sweep.Vs, win, [], [], 1/Ts);
```

The transfer function is estimated and shown in Figure 3.12.

```
Matlab
%% Transfer function estimation
[iff_sweep, ~] = tfestimate(apa_sweep.Va, apa_sweep.Vs, win, [], [], 1/Ts);
```

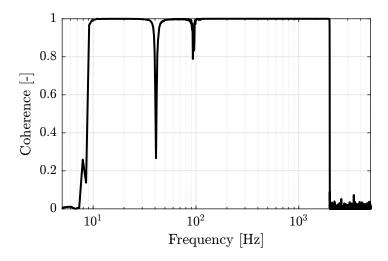


Figure 3.11: Coherence for the identification from V_a to V_s

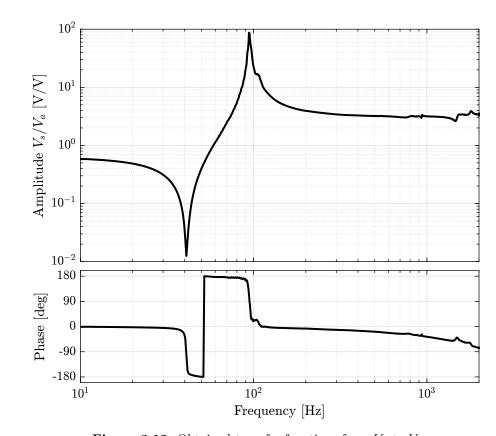


Figure 3.12: Obtained transfer functions from V_a to V_s

3.2.5 Hysteresis

We here wish to visually see the amount of hysteresis present in the APA.

To do so, a quasi static sinusoidal excitation V_a at different voltages is used.

The offset is 65V, and the sin amplitude is ranging from 1V up to 80V.

For each excitation amplitude, the vertical displacement d of the mass is measured.

Then, d is plotted as a function of V_a for all the amplitudes.

We expect to obtained something like the hysteresis shown in Figure 3.13.

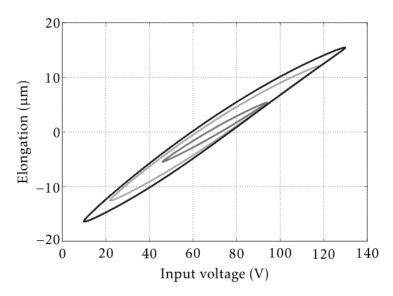


Figure 6.16: Measured hysteresis loops for a single PE actuator, 70 V bias + 1 Hz sine wave: 60 V amplitude (black), 48 V amplitude (light grey), 24 V amplitude (dark grey)

Figure 3.13: Expected Hysteresis [1]

The data is loaded.

```
apa_hyst = load('frf_data_1_hysteresis.mat', 't', 'Va', 'de');
% Initial time set to zero
apa_hyst.t = apa_hyst.t - apa_hyst.t(1);
```

The excitation voltage amplitudes are:

The excitation voltage and the measured displacement are shown in Figure 3.14.

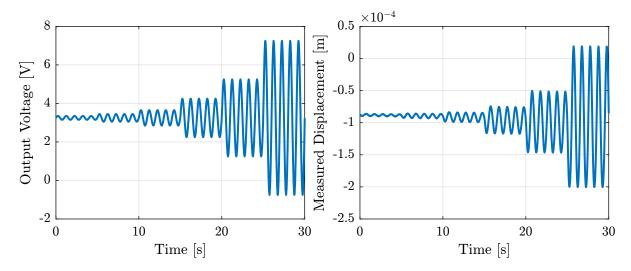


Figure 3.14: Excitation voltage and measured displacement

For each amplitude, we only take the last sinus in order to reduce possible transients. Also, it is centered on zero.

The measured displacement at a function of the output voltage are shown in Figure 3.15.

Important

It is quite clear that hysteresis is increasing with the excitation amplitude. Also, no hysteresis is found on the sensor stack voltage.

3.2.6 Estimation of the APA axial stiffness

In order to estimate the stiffness of the APA, a weight with known mass m_a is added on top of the suspended granite and the deflection d_e is measured using the encoder. The APA stiffness is then:

$$k_{\rm apa} = \frac{m_a g}{d} \tag{3.1}$$

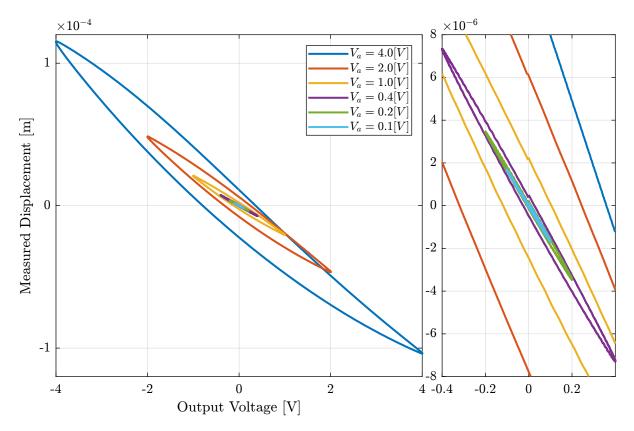
Here, a mass of 6.4 kg is used:

```
added_mass = 6.4; % Added mass [kg]
```

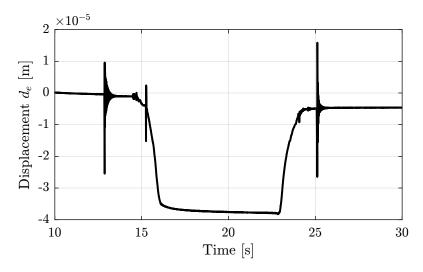
The data is loaded, and the measured displacement is shown in Figure 3.16.

```
apa_mass = load(sprintf('frf_data_%i_add_mass_closed_circuit.mat', 1), 't', 'de');
apa_mass.de = apa_mass.de - mean(apa_mass.de(apa_mass.t<11));</pre>
```

There is some imprecision in the measurement as there are some drifts that are probably due to some creep.



 ${\bf Figure~3.15:~Obtained~hysteres is~for~multiple~excitation~amplitudes}$



 $\textbf{Figure 3.16:} \ \ \textbf{Measured displacement when adding the mass and removing the mass} \\$

The stiffness is then computed as follows:

And the stiffness obtained is very close to the one specified in the documentation $(k = 1.794 [N/\mu m])$.

```
k = 1.68 [N/um] Results _____
```

3.2.7 Stiffness change due to electrical connections

We wish here to see if the stiffness changes when the actuator stacks are not connected to the amplifier and the sensor stacks are not connected to the ADC.

Note here that the resistor in parallel to the sensor stack is present in both cases.

First, the data are loaded.

```
_____Matlab
add_mass_oc = load(sprintf('frf_data_%i_add_mass_open_circuit.mat', 1), 't', 'de');
add_mass_cc = load(sprintf('frf_data_%i_add_mass_closed_circuit.mat', 1), 't', 'de');
```

And the initial displacement is set to zero.

```
_____Matlab ______add_mass_oc.de = add_mass_oc.de - mean(add_mass_oc.t<11));
add_mass_cc.de = add_mass_cc.de - mean(add_mass_cc.de(add_mass_cc.t<11));
```

The measured displacements are shown in Figure 3.17.

And the stiffness is estimated in both case. The results are shown in Table 3.2.

```
______Matlab
apa_k_oc = 9.8 * added_mass / (mean(add_mass_oc.de(add_mass_oc.t > 12 & add_mass_oc.t < 12.5)) -

→ mean(add_mass_oc.de(add_mass_oc.t > 20 & add_mass_oc.t < 20.5)));
apa_k_cc = 9.8 * added_mass / (mean(add_mass_cc.de(add_mass_cc.t > 12 & add_mass_cc.t < 12.5)) -

→ mean(add_mass_cc.de(add_mass_cc.t < 20.5)));
```

Table 3.2: Measured stiffnesses on "open" and "closed" circuits

	$k[N/\mu m]$
Not connected	2.3
Connected	1.7

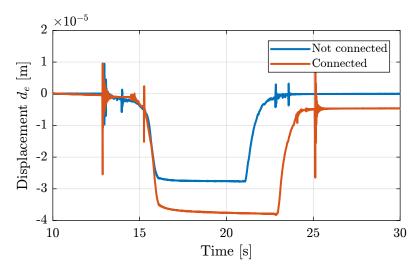


Figure 3.17: Measured displacement

Important

Clearly, connecting the actuator stacks to the amplified (basically equivalent as to short circuiting them) lowers the stiffness.

3.2.8 Effect of the resistor on the IFF Plant

A resistor $R \approx 80.6 \, k\Omega$ is added in parallel with the sensor stack. This has the effect to form a high pass filter with the capacitance of the stack.

We here measured the low frequency transfer function from V_a to V_s with and without this resistor.

```
% With the resistor
wi_k = load('frf_data_1_sweep_lf_with_R.mat', 't', 'Vs', 'Va');

% Without the resistor
wo_k = load('frf_data_1_sweep_lf.mat', 't', 'Vs', 'Va');
```

We use a very long "Hanning" window for the spectral analysis in order to estimate the low frequency behavior.

```
win = hanning(ceil(50*Fs)); % Hannning Windows
```

And we estimate the transfer function from V_a to V_s in both cases:

With the following values of the resistor and capacitance, we obtain a first order high pass filter with a crossover frequency equal to:

```
C = 5.1e-6; % Sensor Stack capacitance [F]
R = 80.6e3; % Parallel Resistor [Ohm]

f0 = 1/(2*pi*R*C); % Crossover frequency of RC HPF [Hz]
```

```
f0 = 0.39 [Hz] Results
```

The transfer function of the corresponding high pass filter is:

```
G_hpf = 0.6*(s/2*pi*f0)/(1 + s/2*pi*f0); Matlab
```

Let's compare the transfer function from actuator stack to sensor stack with and without the added resistor in Figure 3.18.

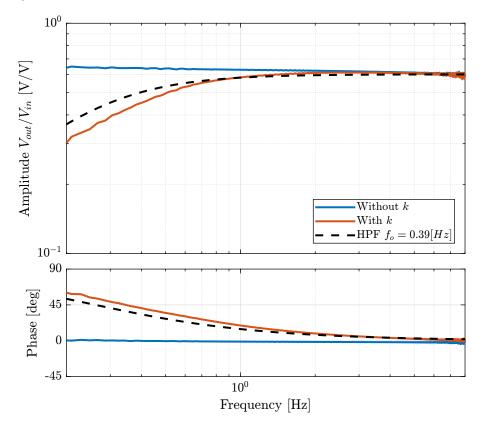


Figure 3.18: Transfer function from V_a to V_s with and without the resistor k

Important

The added resistor has indeed the expected effect.

3.3 Comparison of all the APA

The same measurements that was performed in Section 3.2 are now performed on all the APA and then compared.

3.3.1 Axial Stiffnesses - Comparison

Let's first compare the APA axial stiffnesses.

The added mass is:

```
added_mass = 6.4; % Added mass [kg]
```

Here are the number of the APA that have been measured:

```
apa_nums = [1 2 4 5 6 7 8];
```

The data are loaded.

```
apa_mass = {};
for i = 1:length(apa_nums)
    apa_mass(i) = {load(sprintf('frf_data_%i_add_mass_closed_circuit.mat', apa_nums(i)), 't', 'de')};
% The initial displacement is set to zero
    apa_mass{i}.de = apa_mass{i}.de - mean(apa_mass{i}.de(apa_mass{i}.t<11));
end</pre>
```

The raw measurements are shown in Figure 3.19. All the APA seems to have similar stiffness except the APA 7 which should have an higher stiffness.

Question

It is however strange that the displacement d_e when the mass is removed is higher for the APA 7 than for the other APA. What could cause that?

The stiffnesses are computed for all the APA and are summarized in Table 3.3.

Important

The APA300ML manual specifies the nominal stiffness to be $1.8 [N/\mu m]$ which is very close to what have been measured. Only the APA number 7 is a little bit off.

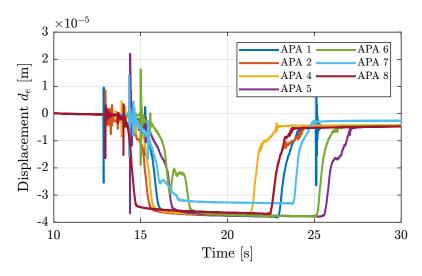


Figure 3.19: Raw measurements for all the APA. A mass of $6.4 \mathrm{kg}$ is added at arround $15 \mathrm{s}$ and removed at arround $22 \mathrm{s}$

Table 3.3: Measured stiffnesses

APA Num	$k[N/\mu m]$
1	1.68
2	1.69
4	1.7
5	1.7
6	1.7
7	1.93
8	1.73

3.3.2 FRF Identification - Setup

The identification is performed in three steps:

- 1. White noise excitation with small amplitude. This is used to determine the main resonance of the system.
- 2. Sweep sine excitation with the amplitude lowered around the resonance. The sweep sine is from 10Hz to 400Hz.
- 3. High frequency noise. The noise is band-passed between 300Hz and 2kHz.

Then, the result of the second identification is used between 10Hz and 350Hz and the result of the third identification if used between 350Hz and 2kHz.

Here are the APA numbers that have been measured.

```
apa_nums = [1 2 4 5 6 7 8]; Matlab _____
```

The data are loaded for both the second and third identification:

```
Matlab

%% Second identification
apa_sweep = {};
for i = 1:length(apa_nums)
    apa_sweep(i) = {load(sprintf('frf_data_%i_sweep.mat', apa_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};
end

%% Third identification
apa_noise_hf = {};
for i = 1:length(apa_nums)
    apa_noise_hf(i) = {load(sprintf('frf_data_%i_noise_hf.mat', apa_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};
end
```

The time is the same for all measurements.

```
Matlab

t = apa_sweep{1}.t - apa_sweep{1}.t(1); % Time vector [s]

%% Sampling
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a "Hanning" windows that will be used for the spectral analysis:

```
win = hanning(ceil(0.5*Fs)); % Hannning Windows
Matlab
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```
% Only used to have the frequency vector "f"
[~, f] = tfestimate(apa_sweep{1}.Va, apa_sweep{1}.de, win, [], [], 1/Ts);
```

3.3.3 FRF Identification - DVF

In this section, the dynamics from excitation voltage V_a to encoder measured displacement d_e is identified.

We compute the coherence for 2nd and 3rd identification:

```
Matlab

coh_sweep = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_sweep{i}.Va, apa_sweep{i}.de, win, [], [], 1/Ts);
    coh_sweep(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_noise_hf{i}.Va, apa_noise_hf{i}.de, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;
end
```

The coherence is shown in Figure 3.20. It is clear that the Sweep sine gives good coherence up to 400Hz and that the high frequency noise excitation signal helps increasing a little bit the coherence at high frequency.

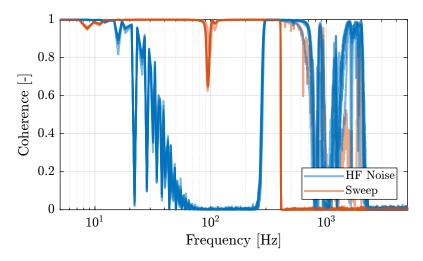


Figure 3.20: Obtained coherence for the plant from V_a to d_e

Then, the transfer function from the DAC output voltage V_a to the measured displacement by the encoders is computed:

```
Matlab

%% Transfer function estimation

dvf_sweep = zeros(length(f), length(apa_nums));

for i = 1:length(apa_nums)

    [frf, ~] = tfestimate(apa_sweep{i}.Va, apa_sweep{i}.de, win, [], [], 1/Ts);

    dvf_sweep(:, i) = frf;
```

```
end

dvf_noise_hf = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [frf, ~] = tfestimate(apa_noise_hf{i}.Va, apa_noise_hf{i}.de, win, [], [], 1/Ts);
    dvf_noise_hf(:, i) = frf;
end
```

The obtained transfer functions are shown in Figure 3.21. They are all superimposed except for the APA7.

Question

Why is the APA7 off? We could think that the APA7 is stiffer, but also the mass line is off. It seems that there is a "gain" problem. The encoder seems fine (it measured the same as the Interferometer). Maybe it could be due to the amplifier?

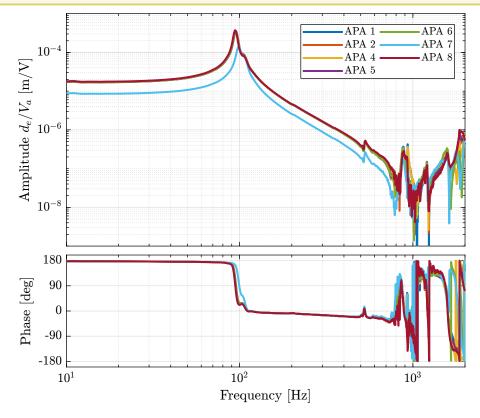


Figure 3.21: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

A zoom on the main resonance is shown in Figure 3.22. It is clear that expect for the APA 7, the response around the resonances are well matching for all the APA.

It is also clear that there is not a single resonance but two resonances, a first one at 95Hz and a second one at 105Hz.

${f Question}$

Why is there a double resonance at around 94Hz?

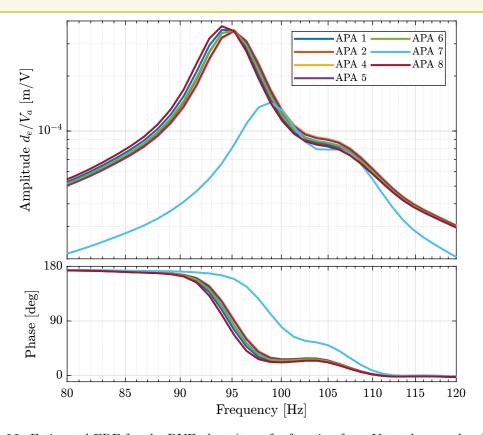


Figure 3.22: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e) - Zoom on the main resonance

3.3.4 FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

First the coherence is computed and shown in Figure 3.23. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

```
Matlab

%% Coherence
coh_sweep = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_sweep{i}.Va, apa_sweep{i}.Vs, win, [], [], 1/Ts);
    coh_sweep(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(apa_nums));
for i = 1:length(apa_nums)
    [coh, ~] = mscohere(apa_noise_hf{i}.Va, apa_noise_hf{i}.Vs, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;
end
```

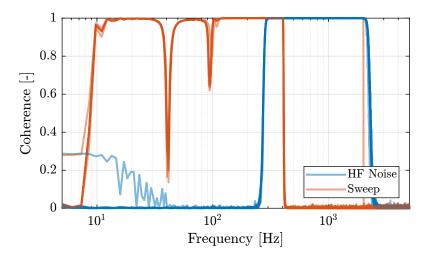


Figure 3.23: Obtained coherence for the IFF plant

Then the FRF are estimated and shown in Figure 3.24

```
Matlab

%% FRF estimation of the transfer function from Va to Vs

iff_sweep = zeros(length(f), length(apa_nums));

for i = 1:length(apa_nums)
    [frf, ~] = tfestimate(apa_sweep{i}.Va, apa_sweep{i}.Vs, win, [], [], 1/Ts);
    iff_sweep(:, i) = frf;
end

iff_noise_hf = zeros(length(f), length(apa_nums));

for i = 1:length(apa_nums)
    [frf, ~] = tfestimate(apa_noise_hf{i}.Va, apa_noise_hf{i}.Vs, win, [], [], 1/Ts);
    iff_noise_hf(:, i) = frf;
end
```

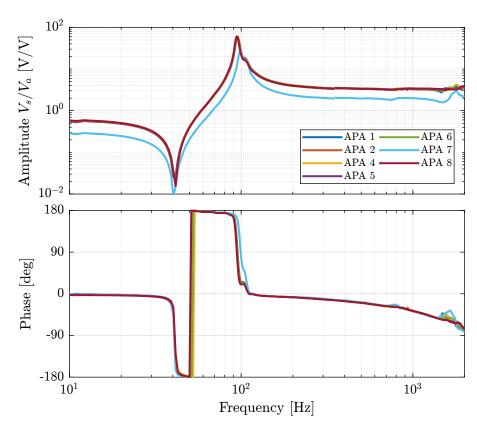


Figure 3.24: Identified IFF Plant

4 Dynamical measurements - Struts

The same bench used in Section 3 is here used with the strut instead of only the APA.

The bench is shown in Figure 4.3. Measurements are performed either when no encoder is fixed to the strut (Figure 4.2) or when one encoder is fixed to the strut (Figure 4.3).



Figure 4.1: Test Bench with Strut - Overview

4.1 Measurement on Strut 1

Measurements are first performed on the strut 1 that contains:

- APA 1
- flex 1 and flex 2

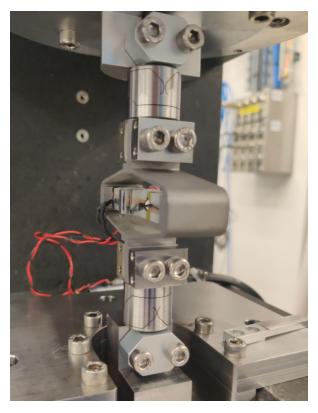


Figure 4.2: Test Bench with Strut - Zoom on the strut

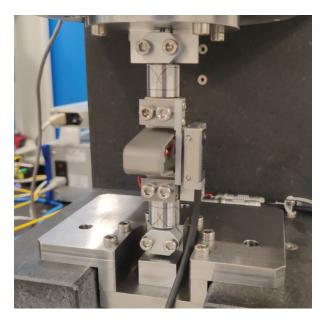


Figure 4.3: Test Bench with Strut - Zoom on the strut with the encoder

4.1.1 Without Encoder

FRF Identification - Setup

The identification is performed in three steps:

- 1. White noise excitation with small amplitude. This is used to determine the main resonance of the system.
- 2. Sweep sine excitation with the amplitude lowered around the resonance. The sweep sine is from $10\mathrm{Hz}$ to $400\mathrm{Hz}$.
- 3. High frequency noise. The noise is band-passed between 300Hz and 2kHz.

Then, the result of the second identification is used between 10Hz and 350Hz and the result of the third identification if used between 350Hz and 2kHz.

```
leg_sweep = load(sprintf('frf_data_leg_%i_sweep.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
leg_noise_hf = load(sprintf('frf_data_leg_%i_noise_hf.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
```

The time is the same for all measurements.

```
Matlab

%% Time vector
t = leg_sweep.t - leg_sweep.t(1); % Time vector [s]

%% Sampling
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a "Hanning" windows that will be used for the spectral analysis:

```
win = hanning(ceil(0.5*Fs)); % Hannning Windows
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```
% Only used to have the frequency vector "f" [~, f] = tfestimate(leg_sweep.Va, leg_sweep.de, win, [], [], 1/Ts);
```

FRF Identification - Displacement

In this section, the dynamics from the excitation voltage V_a to the interferometer d_a is identified.

We compute the coherence for 2nd and 3rd identification:

```
______Matlab [coh_sweep, ~] = mscohere(leg_sweep.Va, leg_sweep.da, win, [], [], 1/Ts); [coh_noise_hf, ~] = mscohere(leg_noise_hf.Va, leg_noise_hf.da, win, [], [], 1/Ts);
```

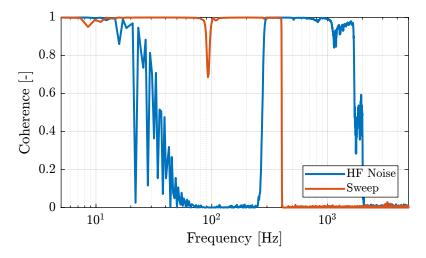


Figure 4.4: Obtained coherence for the plant from V_a to d_a

The transfer function from V_a to the interferometer measured displacement d_a is estimated and shown in Figure 4.5.

```
_____Matlab

[dvf_sweep, ~] = tfestimate(leg_sweep.Va, leg_sweep.da, win, [], [], 1/Ts);

[dvf_noise_hf, ~] = tfestimate(leg_noise_hf.Va, leg_noise_hf.da, win, [], [], 1/Ts);
```

FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

First the coherence is computed and shown in Figure 4.13. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

Then the FRF are estimated and shown in Figure 4.7

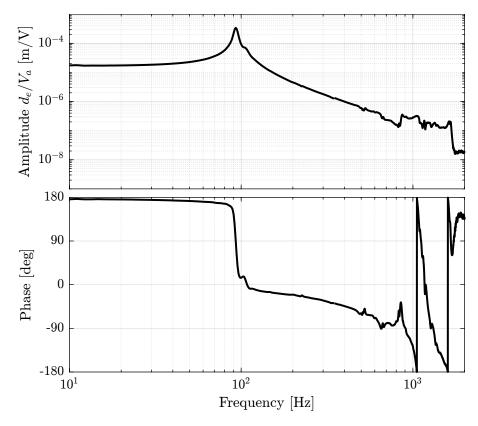


Figure 4.5: Estimated FRF for the DVF plant (transfer function from V_a to the interferometer d_a)

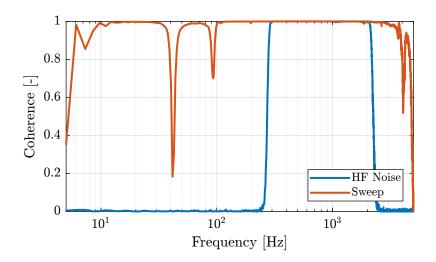


Figure 4.6: Obtained coherence for the IFF plant

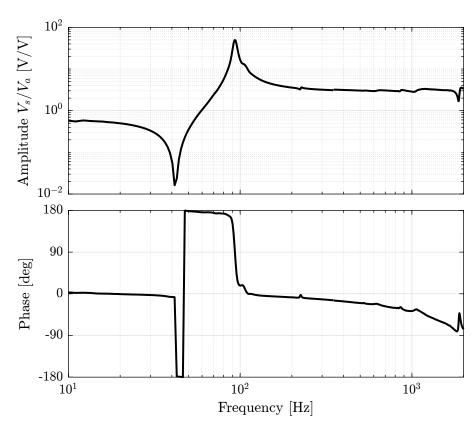


Figure 4.7: Identified IFF Plant for the Strut 1

4.1.2 With Encoder

Measurement Data

FRF Identification - DVF

In this section, the dynamics from V_a to d_e is identified.

We compute the coherence for 2nd and 3rd identification:

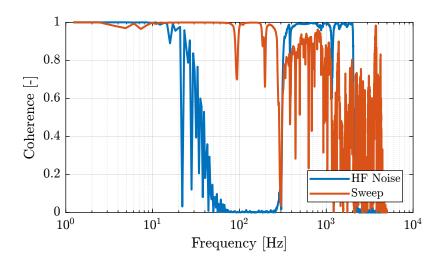


Figure 4.8: Obtained coherence for the plant from V_a to d_e

```
______Matlab
[dvf_enc_sweep, ~] = tfestimate(leg_enc_sweep.Va, leg_enc_sweep.de, win, [], [], 1/Ts);
[dvf_enc_noise_hf, ~] = tfestimate(leg_enc_noise_hf.Va, leg_enc_noise_hf.de, win, [], [], 1/Ts);
```

The obtained transfer functions are shown in Figure 4.9.

They are all superimposed except for the APA7.

Question

Why is the APA7 off? We could think that the APA7 is stiffer, but also the mass line is off. It seems that there is a "gain" problem. The encoder seems fine (it measured the same as the Interferometer). Maybe it could be due to the amplifier?

Question

Why is there a double resonance at around 94Hz?

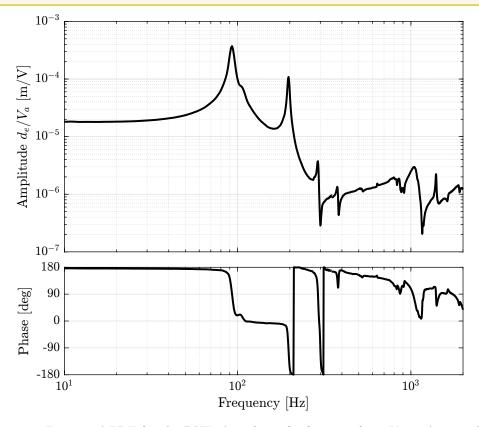


Figure 4.9: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

Comparison of the Encoder and Interferometer

The interferometer could here represent the case where the encoders are fixed to the plates and not the APA.

The dynamics from V_a to d_e and from V_a to d_a are compared in Figure 4.10.

Important

It will clearly be difficult to do something (except some low frequency positioning) with the encoders fixed to the APA.

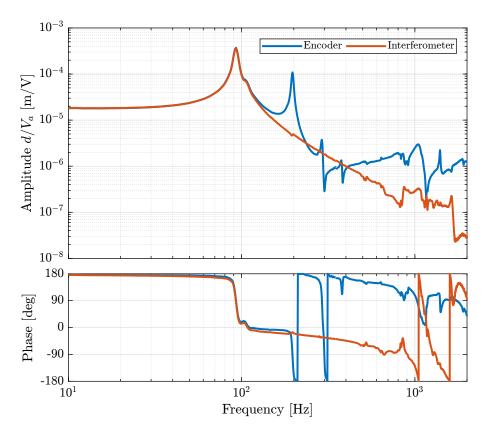


Figure 4.10: Comparison of the transfer functions from excitation voltage V_a to either the encoder d_e or the interferometer d_a

APA Resonances Frequency

As shown in Figure 4.11, we can clearly see three spurious resonances at 197Hz, 290Hz and 376Hz.

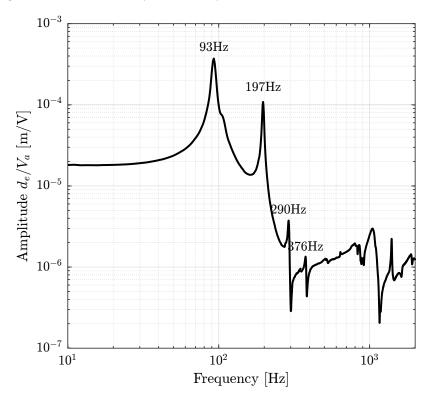


Figure 4.11: Magnitude of the transfer function from excitation voltage V_a to encoder measurement d_e . The frequency of the resonances are noted.

These resonances correspond to parasitic resonances of the APA itself. They are very close to what was estimated using the FEM (Figure 4.12):

- $\bullet\,$ Mode in X-bending at 189Hz
- Mode in Y-bending at 285Hz
- \bullet Mode in Z-torsion at 400Hz

${\bf Important}$

The resonances are indeed due to limited stiffness of the APA.

Estimated Flexible Joint axial stiffness

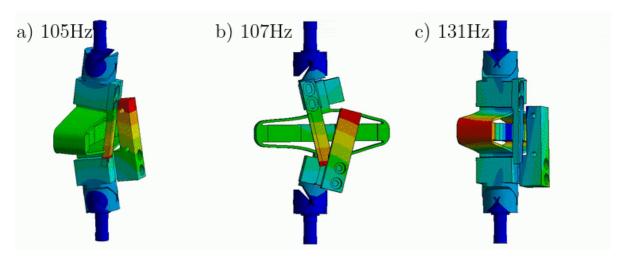


Figure 4.12: Spurious resonances. a) X-bending mode at 189Hz. b) Y-bending mode at 285Hz. c) Z-torsion mode at 400Hz

```
figure;
hold on;
plot(t, de, 'DisplayName', 'Encoder')
plot(t, da, 'DisplayName', 'Interferometer')
hold off;
xlabel('Time [s]'); ylabel('Displacement [m]');
legend('location', 'northeast');
```

```
lin_fit = (added_mass\abs(de_steps-da_steps) - abs(de_steps(1)-da_steps(1)));
```

```
figure;
hold on;
plot(added_mass, abs(de_steps-da_steps) - abs(de_steps(1)-da_steps(1)), 'ok')
plot(added_mass, added_mass*lin_fit, '-r')
hold off;
```

Question

What is strange is that the encoder is measuring a larger displacement than the interferometer. That should be the opposite. Maybe is is caused by the fact that the APA is experiencing some bending and therefore a larger motion is measured on the encoder.

FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

First the coherence is computed and shown in Figure 4.13. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

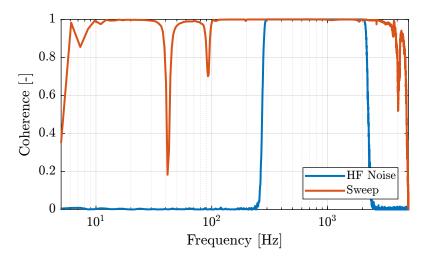


Figure 4.13: Obtained coherence for the IFF plant

Then the FRF are estimated and shown in Figure 4.14

```
iff_enc_sweep, ~] = tfestimate(leg_enc_sweep.Va, leg_enc_sweep.Vs, win, [], [], 1/Ts);
[iff_enc_noise_hf, ~] = tfestimate(leg_enc_noise_hf.Va, leg_enc_noise_hf.Vs, win, [], [], 1/Ts);
```

Let's now compare the IFF plants whether the encoders are fixed to the APA or not (Figure 4.15).

Important

We can see that the IFF does not change whether of not the encoder are fixed to the struts.

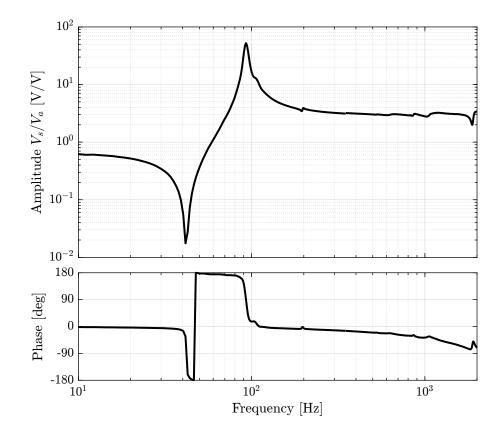


Figure 4.14: Identified IFF Plant

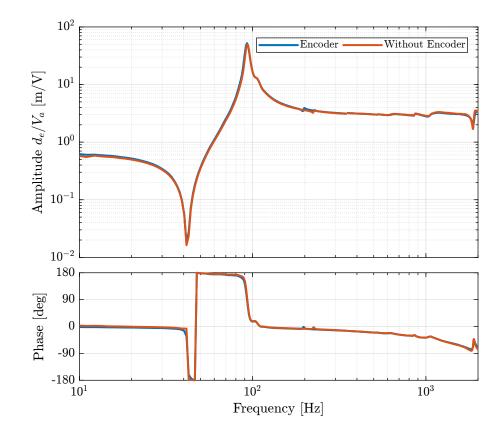


Figure 4.15: Effect of the encoder on the IFF plant

4.2 Comparison of all the Struts

Now all struts are measured using the same procedure and test bench.

4.2.1 FRF Identification - Setup

The identification is performed in two steps:

- 1. White noise excitation with small amplitude. This is used to estimate the low frequency dynamics.
- 2. High frequency noise. The noise is band-passed between 300Hz and 2kHz.

Then, the result of the first identification is used between 10Hz and 350Hz and the result of the second identification if used between 350Hz and 2kHz.

Here are the LEG numbers that have been measured.

```
leg_nums = [1 2 3 4 5];
```

The data are loaded for both the first and second identification:

```
Matlab

%% Second identification
leg_noise = {};
for i = 1:length(leg_nums)
    leg_noise(i) = {load(sprintf('frf_data_leg_coder_%i_noise.mat', leg_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};
end

%% Third identification
leg_noise_hf = {};
for i = 1:length(leg_nums)
    leg_noise_hf(i) = {load(sprintf('frf_data_leg_coder_%i_noise_hf.mat', leg_nums(i)), 't', 'Va', 'Vs', 'de', 'da')};
end
```

The time is the same for all measurements.

```
Matlab

t = leg_noise{1}.t - leg_noise{1}.t(1); % Time vector [s]

%% Sampling
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a "Hanning" windows that will be used for the spectral analysis:

```
win = hanning(ceil(0.5*Fs)); % Hannning Windows
Matlab
```

We get the frequency vector that will be the same for all the frequency domain analysis.

```
% Only used to have the frequency vector "f"
[~, f] = tfestimate(leg_noise{1}.Va, leg_noise{1}.de, win, [], [], 1/Ts);
```

4.2.2 FRF Identification - DVF

In this section, the dynamics from V_a to d_e is identified.

We compute the coherence for 2nd and 3rd identification:

```
Matlab

%% Coherence computation
coh_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
        [coh, ~] = mscohere(leg_noise{i}.Va, leg_noise{i}.de, win, [], [], 1/Ts);
        coh_noise(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
        [coh, ~] = mscohere(leg_noise_hf{i}.Va, leg_noise_hf{i}.de, win, [], [], 1/Ts);
        coh_noise_hf(:, i) = coh;
end
```

The coherence is shown in Figure 4.16. It is clear that the Noise sine gives good coherence up to 400Hz and that the high frequency noise excitation signal helps increasing a little bit the coherence at high frequency.

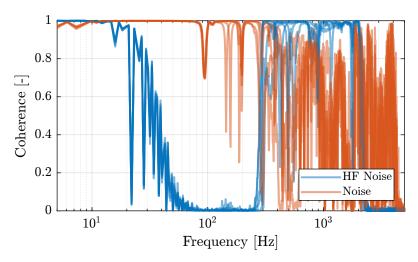


Figure 4.16: Obtained coherence for the plant from V_a to d_e

Then, the transfer function from the DAC output voltage V_a to the measured displacement by the encoders is computed:

```
Matlab

// Transfer function estimation

dvf_noise = zeros(length(f), length(leg_nums));

for i = 1:length(leg_nums)

   [frf, ~] = tfestimate(leg_noise{i}.Va, leg_noise{i}.de, win, [], [], 1/Ts);

   dvf_noise(:, i) = frf;
end
```

```
dvf_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise_hf{i}.Va, leg_noise_hf{i}.de, win, [], [], 1/Ts);
    dvf_noise_hf(:, i) = frf;
end
```

The obtained transfer functions are shown in Figure 4.17.

They are all superimposed except for the LEG7.

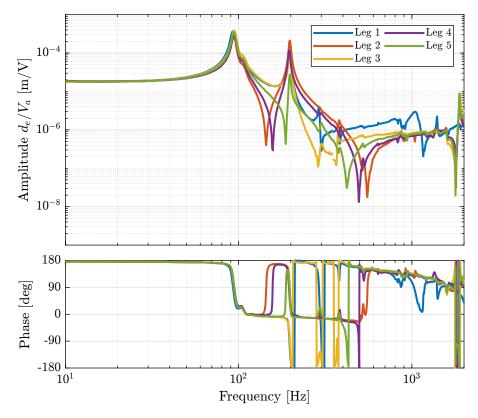


Figure 4.17: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

Important

Depending on how the APA are mounted with the flexible joints, the dynamics can change a lot as shown in Figure 4.17. In the future, a "pin" will be used to better align the APA with the flexible joints. We can expect the amplitude of the spurious resonances to decrease.

4.2.3 FRF Identification - DVF with interferometer

In this section, the dynamics from V_a to d_a is identified.

We compute the coherence.

```
Matlab

**Coherence computation

coh_noise = zeros(length(f), length(leg_nums));

for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise{i}.Va, leg_noise{i}.da, win, [], [], 1/Ts);
    coh_noise(:, i) = coh;

end

coh_noise_hf = zeros(length(f), length(leg_nums));

for i = 1:length(leg_nums)
    [coh, ~] = mscohere(leg_noise_hf{i}.Va, leg_noise_hf{i}.da, win, [], [], 1/Ts);
    coh_noise_hf(:, i) = coh;

end
```

The coherence is shown in Figure 4.18. It is clear that the Noise sine gives good coherence up to 400Hz and that the high frequency noise excitation signal helps increasing a little bit the coherence at high frequency.

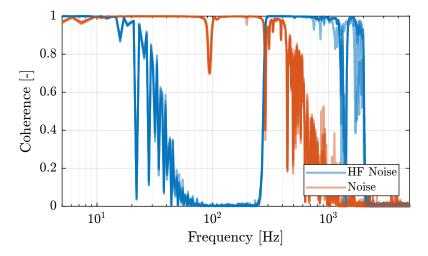


Figure 4.18: Obtained coherence for the plant from V_a to d_e

Then, the transfer function from the DAC output voltage V_a to the measured displacement by the Attocube is computed:

```
Matlab

%% Transfer function estimation

dvf_a_noise = zeros(length(f), length(leg_nums));

for i = 1:length(leg_nums)
        [frf, ~] = tfestimate(leg_noise{i}.Va, leg_noise{i}.da, win, [], [], 1/Ts);
        dvf_a_noise(:, i) = frf;

end

dvf_a_noise_hf = zeros(length(f), length(leg_nums));

for i = 1:length(leg_nums)
        [frf, ~] = tfestimate(leg_noise_hf{i}.Va, leg_noise_hf{i}.da, win, [], [], 1/Ts);
        dvf_a_noise_hf(:, i) = frf;

end
```

The obtained transfer functions are shown in Figure 4.19.

They are all superimposed except for the LEG7.

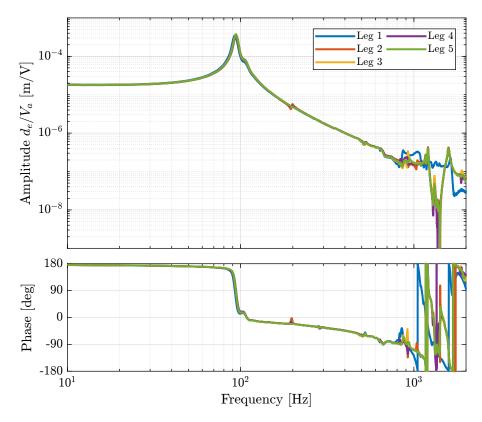


Figure 4.19: Estimated FRF for the DVF plant (transfer function from V_a to the encoder d_e)

4.2.4 FRF Identification - IFF

In this section, the dynamics from V_a to V_s is identified.

First the coherence is computed and shown in Figure 4.20. The coherence is very nice from 10Hz to 2kHz. It is only dropping near a zeros at 40Hz, and near the resonance at 95Hz (the excitation amplitude being lowered).

```
Matlab

%% Coherence
coh_noise = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
        [coh, ~] = mscohere(leg_noise{i}.Va, leg_noise{i}.Vs, win, [], [], 1/Ts);
        coh_noise(:, i) = coh;
end

coh_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
        [coh, ~] = mscohere(leg_noise_hf{i}.Va, leg_noise_hf{i}.Vs, win, [], [], 1/Ts);
        coh_noise_hf(:, i) = coh;
end
```

Then the FRF are estimated and shown in Figure 4.21

```
Matlab

// FRF estimation of the transfer function from Va to Vs

iff_noise = zeros(length(f), length(leg_nums));

for i = 1:length(leg_nums)

[frf, ~] = tfestimate(leg_noise{i}.Va, leg_noise{i}.Vs, win, [], [], 1/Ts);
```

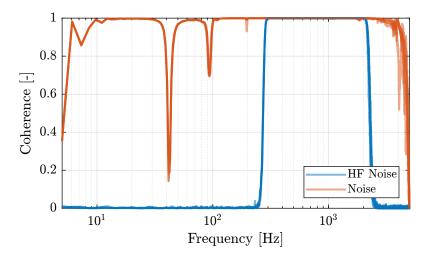


Figure 4.20: Obtained coherence for the IFF plant

```
iff_noise(:, i) = frf;
end

iff_noise_hf = zeros(length(f), length(leg_nums));
for i = 1:length(leg_nums)
    [frf, ~] = tfestimate(leg_noise_hf{i}.Va, leg_noise_hf{i}.Vs, win, [], [], 1/Ts);
    iff_noise_hf(:, i) = frf;
end
```

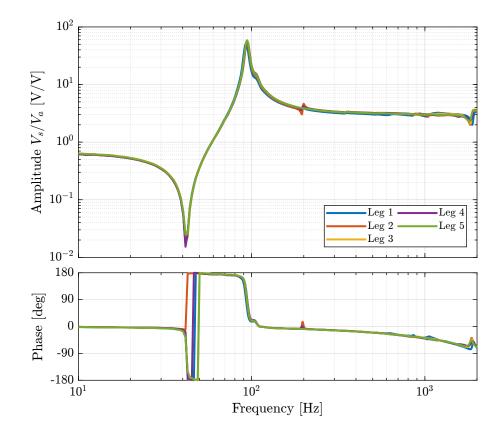


Figure 4.21: Identified IFF Plant

5 Test Bench APA300ML - Simscape Model

5.1 Introduction

A simscape model (Figure 5.1) of the measurement bench is used.

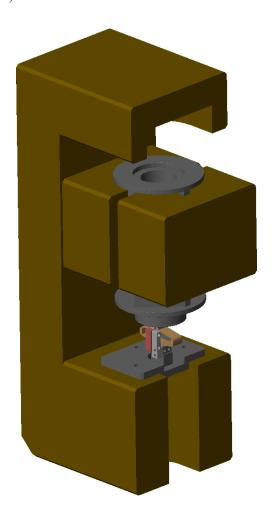


Figure 5.1: Screenshot of the Simscape model

5.2 First Identification

The APA is first initialized with default parameters and the transfer function from excitation voltage V_a (before the amplification of 20 due to the PD200 amplifier) to the sensor stack voltage V_s , encoder d_L and interferometer z is identified.

```
%% Initialize the structure containing the data used in the model
n_hexapod = struct();
n_hexapod.actuator = initializeAPA('type', '2dof');
```

```
Matlab

%% Input/Output definition
clear io; io_i = 1;
io(io_i) = linio([mdl, '/Va'], 1, 'openinput'); io_i = io_i + 1; % Actuator Voltage
io(io_i) = linio([mdl, '/Vs'], 1, 'openoutput'); io_i = io_i + 1; % Sensor Voltage
io(io_i) = linio([mdl, '/dL'], 1, 'openoutput'); io_i = io_i + 1; % Relative Motion Outputs
io(io_i) = linio([mdl, '/z'], 1, 'openoutput'); io_i = io_i + 1; % Vertical Motion

%% Run the linearization
Ga = linearize(mdl, io, 0.0, options);
Ga.InputName = {'Va'};
Ga.OutputName = {'Vs', 'dL', 'z'};
```

The obtain dynamics are shown in Figure 5.2 and 5.3.

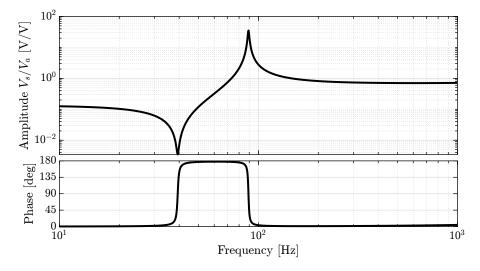


Figure 5.2: Bode plot of the transfer function from V_a to V_s

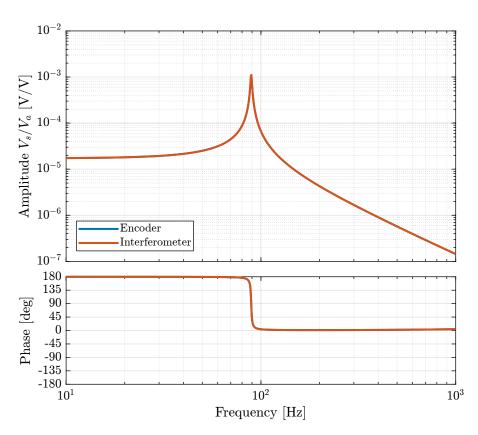


Figure 5.3: Bode plot of the transfer function from V_a to d_L and to z

5.3 Identify Sensor/Actuator constants and compare with measured FRF

5.3.1 How to identify these constants?

Piezoelectric Actuator Constant

Using the measurement test-bench, it is rather easy the determine the static gain between the applied voltage V_a to the induced displacement d.

$$d = g_{d/V_a} \cdot V_a \tag{5.1}$$

Using the Simscape model of the APA, it is possible to determine the static gain between the actuator force F_a to the induced displacement d:

$$d = g_{d/F_a} \cdot F_a \tag{5.2}$$

From the two gains, it is then easy to determine g_a :

$$g_a = \frac{F_a}{V_a} = \frac{F_a}{d} \cdot \frac{d}{V_a} = \frac{g_{d/V_a}}{g_{d/F_a}}$$
 (5.3)

Piezoelectric Sensor Constant

Similarly, it is easy to determine the gain from the excitation voltage V_a to the voltage generated by the sensor stack V_s :

$$V_s = g_{V_s/V_a} V_a \tag{5.4}$$

Note here that there is an high pass filter formed by the piezo capacitor and parallel resistor.

The gain can be computed from the dynamical identification and taking the gain at the wanted frequency (above the first resonance).

Using the simscape model, compute the gain at the same frequency from the actuator force F_a to the strain of the sensor stack dl:

$$dl = g_{dl/F_a} F_a (5.5)$$

Then, the "sensor" constant is:

$$g_s = \frac{V_s}{dl} = \frac{V_s}{V_a} \cdot \frac{V_a}{F_a} \cdot \frac{F_a}{dl} = \frac{g_{V_s/V_a}}{g_a \cdot g_{dl/F_a}}$$

$$(5.6)$$

5.3.2 Identification Data

```
______Matlab

_______Matlab
leg_sweep = load(sprintf('frf_data_%i_sweep.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
leg_noise_hf = load(sprintf('frf_data_%i_noise_hf.mat', 1), 't', 'Va', 'Vs', 'de', 'da');
```

The time is the same for all measurements.

```
Matlab

%% Time vector
t = leg_sweep.t - leg_sweep.t(1); % Time vector [s]

%% Sampling Time / Frequency
Ts = (t(end) - t(1))/(length(t)-1); % Sampling Time [s]
Fs = 1/Ts; % Sampling Frequency [Hz]
```

Then we defined a "Hanning" windows that will be used for the spectral analysis:

```
%% Window used for spectral analysis
win = hanning(ceil(0.5*Fs)); % Hannning Windows
```

We get the frequency vector that will be the same for all the frequency domain analysis.

5.3.3 2DoF APA

2DoF APA

```
Matlab

%% Initialize a 2DoF APA with Ga=Gs=1
n_hexapod.actuator = initializeAPA('type', '2dof', 'ga', 1, 'gs', 1);
```

Identification without actuator or sensor constants

```
Matlab

%% Input/Output definition
clear io; io_i = 1;
io(io_i) = linio([mdl, '/Va'], 1, 'openinput'); io_i = io_i + 1; % Actuator Voltage
io(io_i) = linio([mdl, '/Vs'], 1, 'openoutput'); io_i = io_i + 1; % Sensor Voltage
io(io_i) = linio([mdl, '/dL'], 1, 'openoutput'); io_i = io_i + 1; % Relative Motion Outputs
io(io_i) = linio([mdl, '/z'], 1, 'openoutput'); io_i = io_i + 1; % Vertical Motion
```

```
%% Identification
Gs = linearize(mdl, io, 0.0, options);
Gs.InputName = {'Va'};
Gs.OutputName = {'Vs', 'dL', 'z'};
```

Actuator Constant

Sensor Constant

Comparison

Identify the dynamics with included constants.

```
%% Identify again the dynamics with correct Ga,Gs
Gs = linearize(mdl, io, 0.0, options);
Gs = Gs*exp(-Ts*s);
Gs.InputName = {'Va'};
Gs.OutputName = {'Vs', 'dL', 'z'};
```

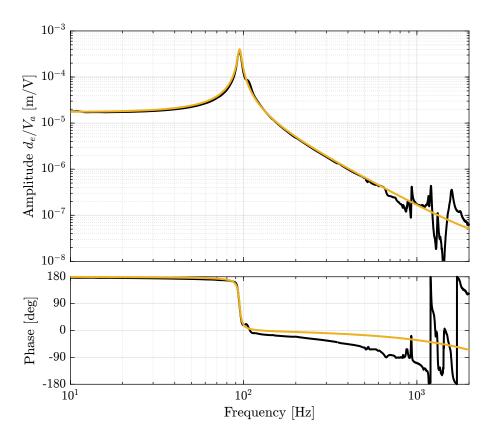


Figure 5.4: Comparison of the experimental data and Simscape model (u to $d\mathcal{L}_m$)

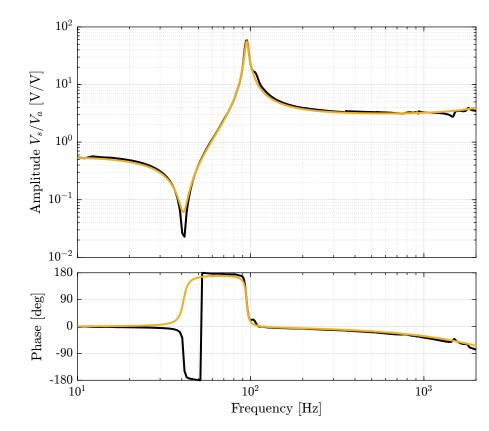


Figure 5.5: Comparison of the experimental data and Simscape model $(V_a \text{ to } V_s)$

5.3.4 Flexible APA

Flexible APA

Identification without actuator or sensor constants

```
Matlab

%% Identify the dynamics
Gs = linearize(mdl, io, 0.0, options);
Gs.InputName = {'Va'};
Gs.OutputName = {'Vs', 'dL', 'z'};
```

Actuator Constant

Sensor Constant

Comparison

```
%% Identify with updated constants
Gs = linearize(mdl, io, 0.0, options);
Gs = Gs*exp(-Ts*s);
Gs.InputName = {'Va'};
Gs.OutputName = {'Vs', 'dL', 'z'};
```

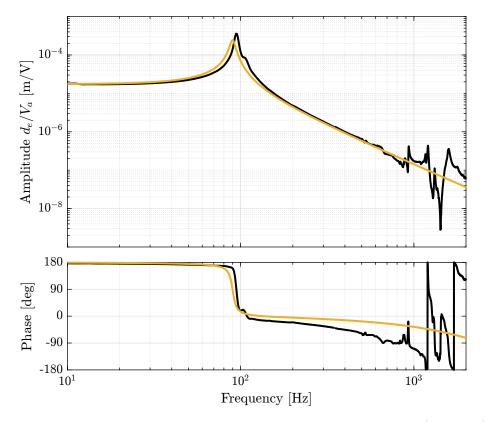


Figure 5.6: Comparison of the experimental data and Simscape model (u to $d\mathcal{L}_m$)

5.4 Optimize 2-DoF model to fit the experimental Data

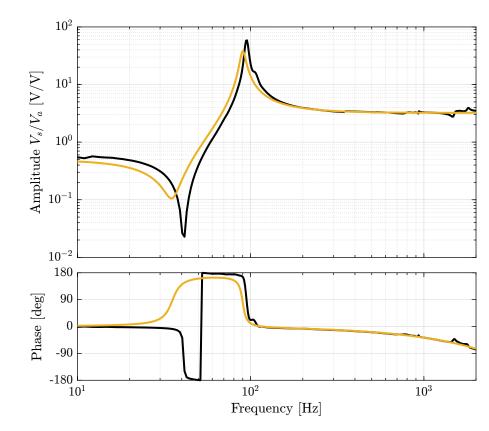


Figure 5.7: Comparison of the experimental data and Simscape model (u to τ_m)

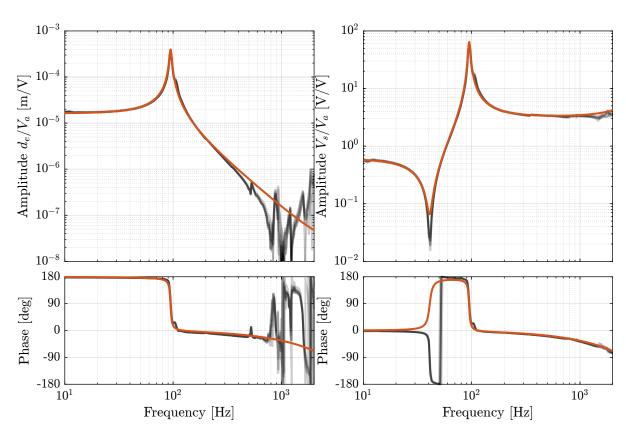


Figure 5.8: Comparison of the measured FRF and the optimized model

6 Test Bench Struts - Simscape Model

6.1 Introduction

6.2 First Identification

The object containing all the data is created.

```
Matlab

%% Initialize structure containing data for the Simscape model
n_hexapod = struct();
n_hexapod.flex_bot = initializeBotFlexibleJoint('type', '2dof');
n_hexapod.flex_top = initializeTopFlexibleJoint('type', '2dof');
n_hexapod.actuator = initializeAPA('type', '2dof');
```

```
Matlab

%% Input/Output definition
clear io; io_i = 1;
io(io_i) = linio([mdl, '/Va'], 1, 'openinput'); io_i = io_i + 1; % Actuator Voltage
io(io_i) = linio([mdl, '/Vs'], 1, 'openoutput'); io_i = io_i + 1; % Sensor Voltage
io(io_i) = linio([mdl, '/dL'], 1, 'openoutput'); io_i = io_i + 1; % Relative Motion Outputs
io(io_i) = linio([mdl, '/z'], 1, 'openoutput'); io_i = io_i + 1; % Vertical Motion
```

6.3 Effect of flexible joints

Perfect

```
Matlab

%% Perfect Joints
n_hexapod.flex_bot = initializeBotFlexibleJoint('type', '2dof');
n_hexapod.flex_top = initializeTopFlexibleJoint('type', '3dof');

Gp = linearize(mdl, io, 0.0, options);
Gp.InputName = {'Va'};
Gp.OutputName = {'Vs', 'dL', 'z'};
```

Top Flexible

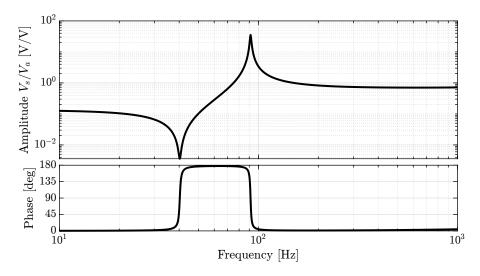


Figure 6.1: Identified transfer function from V_a to V_s

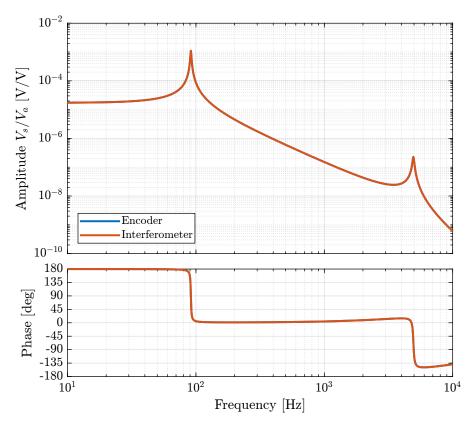


Figure 6.2: Identified transfer function from V_a to d_L

```
Matlab

%% Top joint with Axial stiffness
n_hexapod.flex_bot = initializeBotFlexibleJoint('type', '2dof');
n_hexapod.flex_top = initializeTopFlexibleJoint('type', '4dof');

Gt = linearize(mdl, io, 0.0, options);
Gt.InputName = {'Va'};
Gt.OutputName = {'Vs', 'dL', 'z'};
```

Bottom Flexible

```
Matlab

%% Bottom joint with Axial stiffness
n_hexapod.flex_bot = initializeBotFlexibleJoint('type', '4dof');
n_hexapod.flex_top = initializeTopFlexibleJoint('type', '3dof');

Gb = linearize(mdl, io, 0.0, options);
Gb.InputName = {'Va'};
Gb.OutputName = {'Vs', 'dL', 'z'};
```

Both Flexible

```
Matlab

%% Both joints with Axial stiffness
n_hexapod.flex_bot = initializeBotFlexibleJoint('type', '4dof');
n_hexapod.flex_top = initializeTopFlexibleJoint('type', '4dof');

Gf = linearize(mdl, io, 0.0, options);
Gf.InputName = {'Va'};
Gf.OutputName = {'Vs', 'dL', 'z'};
```

Comparison in Figures 6.3, 6.4 and 6.5.

Important

Imperfection of the flexible joints has the largest impact on the transfer function from V_a to d_L . However, it has relatively small impact on the transfer functions from V_a to V_s and to z.

6.4 Integral Force Feedback

6.4.1 Initialize the system

```
Matlab

%% Initialize typical system
n_hexapod = struct();
n_hexapod.flex_bot = initializeBotFlexibleJoint('type', '2dof');
n_hexapod.flex_top = initializeTopFlexibleJoint('type', '3dof');
n_hexapod.actuator = initializeAPA('type', '2dof');
```

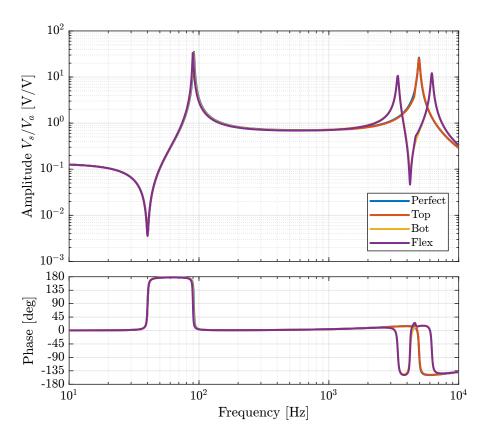


Figure 6.3: Effect of the joint's flexibility on the transfer function from V_a to V_s

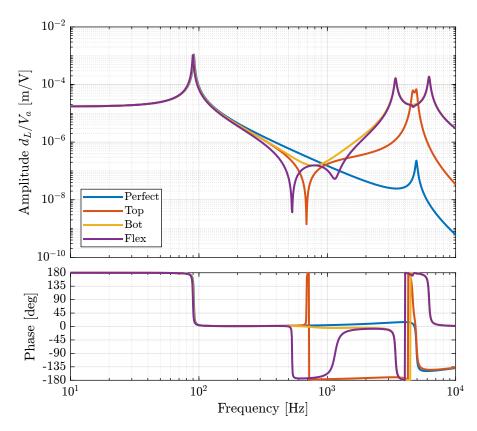


Figure 6.4: Effect of the joint's flexibility on the transfer function from V_a to d_L (encoder)

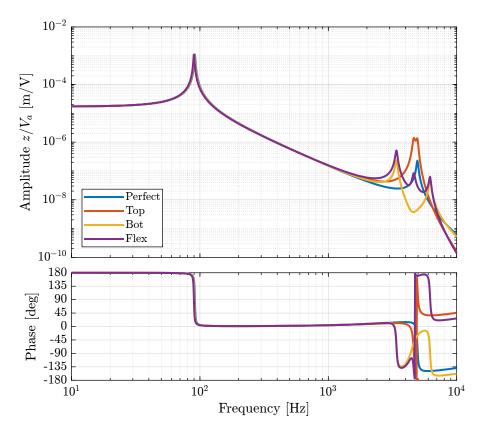


Figure 6.5: Effect of the joint's flexibility on the transfer function from V_a to z (interferometer)

6.4.2 Plant Identification

```
%% Identify th edynamics
G = linearize(mdl, io, 0.0, options);
G.InputName = {'Va'};
G.OutputName = {'Vs', 'dL', 'z'};

Matlab

### Matlab

### IFF Plant (transfer function from u to taum)

Giff = G('Vs', 'Va');
```

6.4.3 Root Locus

```
K_{\rm IFF} = \frac{g}{s + \omega_c} \tag{6.1}
```

6.5 Comparison with the experimental Data

```
Matlab

%% Input/Output definition
clear io; io_i = 1;
io(io_i) = linio([mdl, '/Va'], 1, 'openinput'); io_i = io_i + 1; % Actuator Voltage
io(io_i) = linio([mdl, '/Vs'], 1, 'openoutput'); io_i = io_i + 1; % Sensor Voltage
io(io_i) = linio([mdl, '/z'], 1, 'openoutput'); io_i = io_i + 1; % Relative Motion Outputs

%% Identification
Gs = exp(-s*Ts)*linearize(mdl, io, 0.0, options);
Gs.InputName = {'Va'};
Gs.OutputName = {'Vs', 'z'};
```

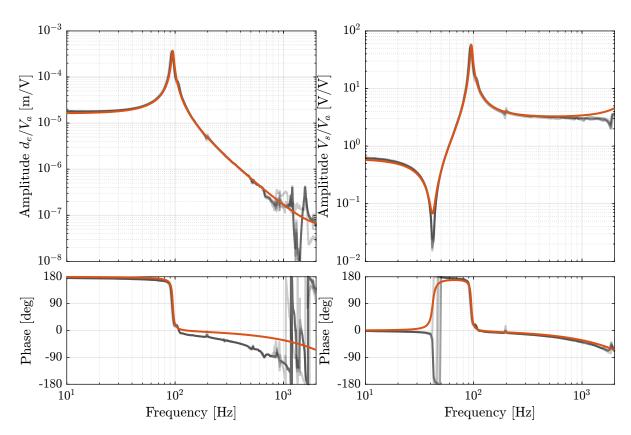


Figure 6.6: Comparison of the measured FRF and the optimized model

7 Function

7.1 initializeBotFlexibleJoint - Initialize Flexible Joint

Function description

```
function [flex_bot] = initializeBotFlexibleJoint(args)
% initializeBotFlexibleJoint -
%
% Syntax: [flex_bot] = initializeBotFlexibleJoint(args)
%
% Inputs:
% - args -
%
% Outputs:
% - flex_bot -
```

Optional Parameters

```
arguments
args.type char {mustBeMember(args.type,{'2dof', '3dof', '4dof'})} = '2dof'

args.kRx (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*5
args.kRz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*5
args.kRz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*260
args.kz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*100

args.cRx (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*0.1
args.cRz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*0.1
args.cRz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*0.1
args.cz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*10.1
args.cz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*10.1
args.cz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*10.1
```

Initialize the structure

```
flex_bot = struct();
Matlab
```

Set the Joint's type

```
switch args.type
case '2dof'
flex_bot.type = 1;
case '3dof'
flex_bot.type = 2;
case '4dof'
flex_bot.type = 3;
end
```

Set parameters

```
flex_bot.kRx = args.kRx;
flex_bot.kRy = args.kRy;
flex_bot.kRz = args.kRz;
flex_bot.kz = args.kz;

flex_bot.cRx = args.cRx;
flex_bot.cRy = args.cRy;
flex_bot.cRz = args.cRy;
flex_bot.cRz = args.cRz;
flex_bot.cRz = args.cRz;
flex_bot.cRz = args.cRz;
flex_bot.cz = args.cz;
```

7.2 initializeTopFlexibleJoint - Initialize Flexible Joint

Function description

```
function [flex_top] = initializeTopFlexibleJoint(args)
% initializeTopFlexibleJoint -
%
% Syntax: [flex_top] = initializeTopFlexibleJoint(args)
%
% Inputs:
% - args -
%
% Outputs:
% - flex_top -
```

Optional Parameters

```
arguments
args.type char {mustBeMember(args.type,{'2dof', '3dof', '4dof'})} = '2dof'
args.kRx (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*5
```

```
args.kRy (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*5
args.kRz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*260
args.kz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*1e8

args.cRx (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*0.1
args.cRy (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*0.1
args.cRz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*0.1
args.cz (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*1e2
end
```

Initialize the structure

```
flex_top = struct();
Matlab
```

Set the Joint's type

```
switch args.type
case '2dof'
flex_top.type = 1;
case '3dof'
flex_top.type = 2;
case '4dof'
flex_top.type = 3;
end
```

Set parameters

flex_top.cz = args.cz;

```
flex_top.kRx = args.kRx;
flex_top.kRy = args.kRy;
flex_top.kRz = args.kRz;
flex_top.kz = args.kz;

flex_top.cRx = args.cRx;
flex_top.cRx = args.cRx;
flex_top.cRy = args.cRy;
flex_top.cRy = args.cRy;
flex_top.cRz = args.cRy;
flex_top.cRz = args.cRz;
```

7.3 initializeAPA - Initialize APA

Function description

```
function [actuator] = initializeAPA(args)
% initializeAPA -
%
% Syntax: [actuator] = initializeAPA(args)
%
% Inputs:
% - args -
%
% Outputs:
% - actuator -
```

Optional Parameters

```
arguments
args.type char {mustBeMember(args.type,{'2dof', 'flexible frame', 'flexible'})} = '2dof'

args.Ga (1,1) double {mustBeNumeric} = 0
args.Gs (1,1) double {mustBeNumeric} = 0

**For 2DoF
args.k (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*0.38e6
args.ke (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*1.75e6
args.ka (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*3e7

args.c (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*3e1
args.ce (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*2e1
args.ca (6,1) double {mustBeNumeric, mustBePositive} = ones(6,1)*2e1
args.Leq (6,1) double {mustBeNumeric} = ones(6,1)*0.056

**Force Flexible APA
args.xi (1,1) double {mustBeNumeric, mustBePositive} = 0.5

**For Flexible Frame
args.ks (1,1) double {mustBeNumeric, mustBePositive} = 235e6
args.cs (1,1) double {mustBeNumeric, mustBePositive} = 1e1
end
```

Initialize Structure

```
actuator = struct();
Matlab
```

Type

```
switch args.type
case '2dof'
actuator.type = 1;
case 'flexible frame'
actuator.type = 2;
case 'flexible'
actuator.type = 3;
end
Matlab
```

Actuator/Sensor Constants

```
if args.Ga == 0
    switch args.type
    case '2dof'
        actuator.Ga = -30.0;
    case 'flexible frame'
        actuator.Ga = 1; % TODO
    case 'flexible'
        actuator.Ga = 23.4;
    end
else
    actuator.Ga = args.Ga; % Actuator gain [N/V]
end
```

```
if args.Gs == 0
    switch args.type
    case '2dof'
    actuator.Gs = 0.098;
    case 'flexible frame'
    actuator.Gs = 1; % TODO
    case 'flexible'
    actuator.Gs = -4674824;
    end
else
    actuator.Gs = args.Gs; % Sensor gain [V/m]
end
```

2DoF parameters

```
actuator.k = args.k; % [N/m]
actuator.ke = args.ke; % [N/m]
actuator.ka = args.ka; % [N/m]

actuator.c = args.c; % [N/(m/s)]
actuator.ce = args.ce; % [N/(m/s)]
actuator.ca = args.ca; % [N/(m/s)]
actuator.Leq = args.Leq; % [m]
```

Flexible frame and fully flexible

```
switch args.type
    case 'flexible frame'
    actuator.K = readmatrix('APA300ML_b_mat_K.CSV'); % Stiffness Matrix
    actuator.M = readmatrix('APA300ML_b_mat_M.CSV'); % Mass Matrix
    actuator.P = extractNodes('APA300ML_b_out_nodes_3D.txt'); % Node coordinates [m]
    case 'flexible'
    actuator.K = readmatrix('full_APA300ML_K.CSV'); % Stiffness Matrix
    actuator.M = readmatrix('full_APA300ML_M.CSV'); % Mass Matrix
    actuator.P = extractNodes('full_APA300ML_out_nodes_3D.txt'); % Node coordinates [m]
end

actuator.xi = args.xi; % Damping ratio

actuator.ks = args.ks; % Stiffness of one stack [N/m]
actuator.cs = args.cs; % Damping of one stack [N/m]
```

7.4 generateSweepExc: Generate sweep sinus excitation

Function description

Optional Parameters

```
arguments
                            (1,1) double {mustBeNumeric, mustBePositive} = 1e-4
    args.Ts
                           (1,1) double {mustBeNumeric, mustBePositive} = 1
(1,1) double {mustBeNumeric, mustBePositive} = 1e3
    args.f_start
    args.f_end
                           (1,1) double {mustBeNumeric} = 0
    args.V_mean
    args.V_exc
    args.t_start
                           (1,1) double {mustBeNumeric, mustBeNonnegative} = 5
    args.exc_duration
                          (1,1) double {mustBeNumeric, mustBePositive} = 10
                                          {mustBeMember(args.sweep_type,{'log', 'lin'})} = 'lin'
    args.sweep_type
                                  logical {mustBeNumericOrLogical} = true
    {\it args.smooth\_ends}
end
```

Sweep Sine part

```
If isnumeric(args.V_exc)

V_sweep = args.V_mean + args.V_exc*V_exc;
elseif isct(args.V_exc)

if strcmp(args.sweep_type, 'log')

V_sweep = args.V_mean + abs(squeeze(freqresp(args.V_exc,

args.f_start*(args.f_end/args.f_start).^(t_sweep/args.exc_duration), 'Hz')))'.*V_exc;
elseif strcmp(args.sweep_type, 'lin')

V_sweep = args.V_mean + abs(squeeze(freqresp(args.V_exc,

elseif strcmp(args.sweep_type, 'lin')

V_sweep = args.V_mean + abs(squeeze(freqresp(args.V_exc,

args.f_start+(args.f_end-args.f_start)/args.exc_duration*t_sweep, 'Hz')))'.*V_exc;
end
end
```

Smooth Ends

```
Matlab
if args.t_start > 0
     t_smooth_start = args.Ts:args.Ts:args.t_start;
    V_smooth_start = zeros(size(t_smooth_start));
V_smooth_end = zeros(size(t_smooth_start));
    if args.smooth_ends
          Vd_max = args.V_mean/(0.7*args.t_start);
          V_d = zeros(size(t_smooth_start));
          V_d(t\_smooth\_start < 0.2*args.t\_start) = t\_smooth\_start(t\_smooth\_start < 0.2*args.t\_start) * Vd\_max/(0.2*args.t\_start); 
          V_d(t_smooth_start > 0.2*args.t_start & t_smooth_start < 0.7*args.t_start) = Vd_max;
V_d(t_smooth_start > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start > 0.7*args.t_start) = Vd_max - (t_smooth_start)

→ 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) - 0.7*args.t_start)*Vd_max/(0.2*args.t_start);
</p>
          V_smooth_start = cumtrapz(V_d)*args.Ts;
          V_smooth_end = args.V_mean - V_smooth_start;
    end
else
     V_smooth_start = [];
    V_smooth_end = [];
end
```

Combine Excitation signals

```
V_exc = [V_smooth_start, V_sweep, V_smooth_end];
t_exc = args.Ts*[0:1:length(V_exc)-1];
```

```
U_exc = [t_exc; V_exc];
```

7.5 generateShapedNoise: Generate Shaped Noise excitation

Function description

Optional Parameters

```
arguments

args.Ts (1,1) double {mustBeNumeric, mustBePositive} = 1e-4

args.V_mean (1,1) double {mustBeNumeric} = 0

args.V_exc = 1

args.t_start (1,1) double {mustBeNumeric, mustBePositive} = 5

args.exc_duration args.smooth_ends end
```

Shaped Noise

```
Matlab

t_noise = 0:args.Ts:args.exc_duration;

Matlab

if isnumeric(args.V_exc)
    V_noise = args.V_mean + args.V_exc*sqrt(1/args.Ts/2)*randn(length(t_noise), 1)';
elseif isct(args.V_exc)
    V_noise = args.V_mean + lsim(args.V_exc, sqrt(1/args.Ts/2)*randn(length(t_noise), 1), t_noise)';
end
```

Smooth Ends

```
t_smooth_start = args.Ts:args.t_start;

V_smooth_start = zeros(size(t_smooth_start));
V_smooth_end = zeros(size(t_smooth_start));

if args.smooth_ends
    Vd_max = args.V_mean/(0.7*args.t_start);

V_d = zeros(size(t_smooth_start));
V_d(t_smooth_start < 0.2*args.t_start) = t_smooth_start(t_smooth_start < 0.2*args.t_start)*Vd_max/(0.2*args.t_start);
V_d(t_smooth_start > 0.2*args.t_start & t_smooth_start < 0.7*args.t_start) = Vd_max;
V_d(t_smooth_start > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start) > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start) > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max/(0.2*args.t_start);
V_smooth_start = cumtrapz(V_d)*args.Ts;
V_smooth_end = args.V_mean - V_smooth_start;
end</pre>
```

Combine Excitation signals

```
V_exc = [V_smooth_start, V_noise, V_smooth_end];
t_exc = args.Ts*[0:1:length(V_exc)-1];

Matlab

U_exc = [t_exc; V_exc];
```

7.6 generateSinIncreasingAmpl: Generate Sinus with increasing amplitude

Function description

Optional Parameters

```
arguments

args.Ts (1,1) double {mustBeNumeric, mustBePositive} = 1e-4

args.V_mean (1,1) double {mustBeNumeric} = 0

args.sin_ampls double {mustBeNumeric, mustBePositive} = [0.1, 0.2, 0.3]

args.sin_period (1,1) double {mustBeNumeric, mustBePositive} = 1

args.sin_num (1,1) double {mustBeNumeric, mustBePositive, mustBeInteger} = 3

args.t_start (1,1) double {mustBeNumeric, mustBePositive} = 5

args.smooth_ends logical {mustBeNumericOrLogical} = true

end
```

Sinus excitation

Smooth Ends

```
t_smooth_start = args.Ts:args.Ts:args.t_start;

V_smooth_start = zeros(size(t_smooth_start));
V_smooth_end = zeros(size(t_smooth_start));

if args.smooth_ends
    Vd_max = args.V_mean/(0.7*args.t_start);

V_d = zeros(size(t_smooth_start));
    V_d(t_smooth_start < 0.2*args.t_start) = t_smooth_start(t_smooth_start < 0.2*args.t_start)*Vd_max/(0.2*args.t_start);
    V_d(t_smooth_start > 0.2*args.t_start & t_smooth_start < 0.7*args.t_start) = Vd_max;
    V_d(t_smooth_start > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start > 0.7*args.t_start) & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start > 0.7*args.t_start) & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start(t_smooth_start) > 0.7*args.t_start & t_smooth_start < 0.9*args.t_start) = Vd_max - (t_smooth_start);
    V_smooth_start = cumtrapz(V_d)*args.Ts;
    V_smooth_end = args.V_mean - V_smooth_start;
end</pre>
```

Combine Excitation signals

```
V_exc = [V_smooth_start, sin_exc, V_smooth_end];
t_exc = args.Ts*[0:1:length(V_exc)-1];
```

	Matlab
U_exc = [t_exc; V_exc];	

Bibliography

[1] Gerrit Wijnand van der Poel. "An Exploration of Active Hard Mount Vibration Isolation for Precision Equipment". PhD thesis. University of Twente, 2010. ISBN: 978-90-365-3016-3. DOI: 10.3990/1.9789036530163. URL: https://doi.org/10.3990/1.9789036530163.