

# **Nano Hexapod - Optimal Geometry**

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# Contents

- 1 Review of Stewart platforms** **4**
  
- 2 Effect of geometry on Stewart platform properties** **7**
  - 2.1 Stiffness . . . . . 7
  - 2.2 Mobility and required joint and actuator stroke . . . . . 7
  
- 3 The Cubic Architecture** **8**
  - 3.1 The Cubic Architecture . . . . . 8
  - 3.2 Static Properties . . . . . 8
  - 3.3 Dynamical Properties? . . . . . 8
  
- 4 Conclusion** **10**
  
- Bibliography** **11**

- In the conceptual design phase, the geometry of the Stewart platform was not optimized
- In the detail design phase, we want to see if the geometry can be optimized to improve the overall performances
- Optimization criteria: mobility, stiffness, dynamical decoupling, more performance / bandwidth

Outline:

- Review of Stewart platform: Section 1 Geometry, Actuators, Sensors, Joints
- Effect of geometry on the Stewart platform characteristics 2
- Cubic configuration: benefits? 3

# 1 Review of Stewart platforms

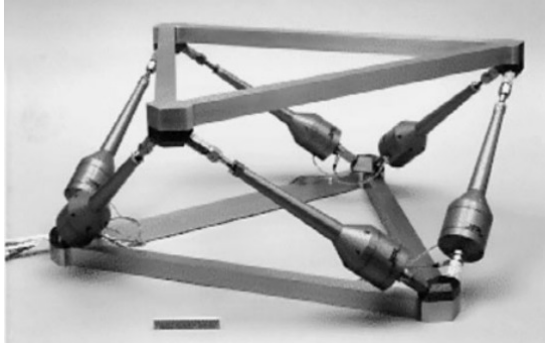
- as was explained in the conceptual phase, Stewart platform have the following key elements:
  - two plates
  - flexible joints
  - actuators
  - sensors
- the geometry
- This results in various designs as shown in Table 1.1
- The focus is here made on Stewart platforms for nano-positioning of vibration control. Not on long stroke stewart platforms.
- All presented Stewart platforms are using flexible joints, as it is a prerequisites for nano-positioning capabilities.
- Most of stewart platforms are using voice coil actuators or piezoelectric actuators. The actuators used for the Stewart platform will be chosen in the next section.
- Depending on the application, various sensors are integrated in the struts or on the plates. The choice of sensor for the nano-hexapod will be described in the next section.

Only keep integrated sensor and not external metrology

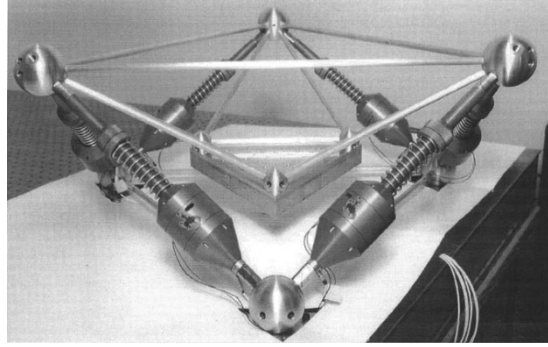
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<https://research.tdehaeze.xyz/stewart-simscape/docs/bibliography.html>

Joints and actuators are optimized in the next section



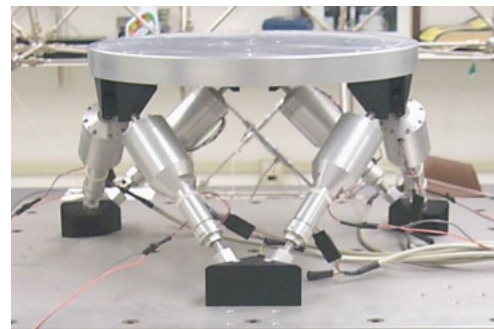
(a) California Institute of Technology - USA



(b) University of Wyoming - USA



(c) ULB - Belgium

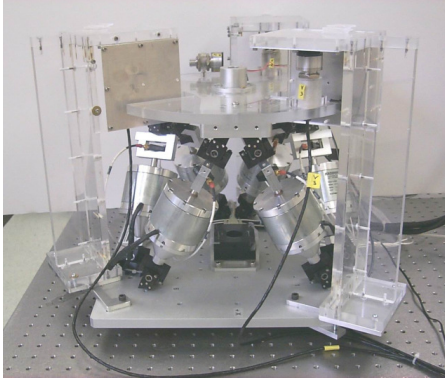


(d) Naval Postgraduate School - USA

**Figure 1.1:** Some examples of developed Stewart platform with Cubic geometry. (a), (b), (c), (d)

**Table 1.1:** Examples of Stewart platform developed. When not specifically indicated, sensors are included in the struts. All presented Stewart platforms are using flexible joints. The table is sorted by “date”

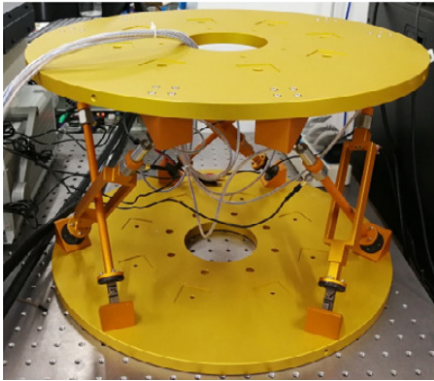
|             | Geometry          | Actuators                           | Sensors                            | Reference  |
|-------------|-------------------|-------------------------------------|------------------------------------|------------|
| Figure 1.1a | Cubic (6-UPU)     | Magnetostrictive                    | Force (collocated), Accelerometers | [20]–[22]  |
|             | Cubic             | Voice Coil (0.5 mm)                 | Force (collocated)                 | [1], [2]   |
|             | Cubic             | Voice Coil (10 mm)                  | Force, LVDT, Geophones             | [3]–[5]    |
| Figure 1.1b | Cubic (CoM=CoK)   | Voice Coil                          | Force                              | [6]–[10]   |
|             | Cubic             | Piezoelectric ( $25 \mu\text{m}$ )  | Piezo force sensors                | [33]       |
| Figure 1.1c | Cubic             | APA ( $50 \mu\text{m}$ )            | Force sensor                       | [24]       |
| Figure 1.2a | Non-Cubic         | Voice Coil                          | Accelerometers                     | [14]       |
|             | Cubic             | Voice Coil                          | Force                              | [11], [12] |
| Figure 1.1d | Cubic             | Piezoelectric ( $50 \mu\text{m}$ )  | Geophone aligned with the strut    | [25]       |
|             | Non-Cubic         | Piezoelectric ( $16 \mu\text{m}$ )  | Eddy Current                       | [30]       |
|             | Cubic             | Piezoelectric ( $120 \mu\text{m}$ ) | External capacitive                | [26], [27] |
| Figure 1.2b | Non-Cubic         | Piezoelectric ( $160 \mu\text{m}$ ) | External capacitive (LION)         | [28]       |
|             | Non-cubic         | Magnetostrictive                    | Inertial                           | [23]       |
|             | 6-SPS (Optimized) | Piezoelectric                       | Strain Gauge                       | [29]       |
|             | Cubic             | Voice Coil                          | Accelerometer in each leg          | [15]–[17]  |
|             | Cubic             | Piezoelectric                       | Force Sensor + Accelerometer       | [32]       |
| Figure 1.2c | Almost cubic      | Voice Coil                          | Force Sensor + Accelerometer       | [18], [19] |
| Figure 1.2d | 6-UPS (Cubic?)    | Piezoelectric                       | Force, Position                    | [31]       |
|             | Non-Cubic         | 3-phase rotary motor                | Rotary Encoders                    | [35], [36] |



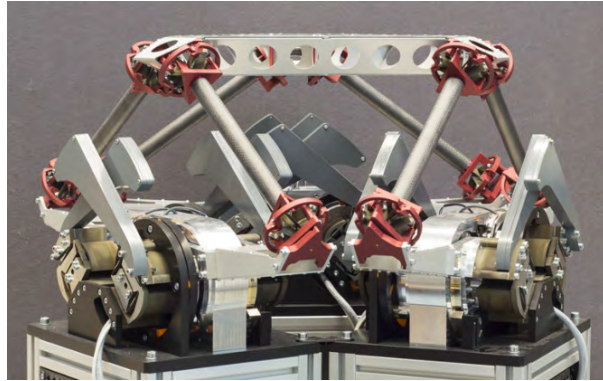
(a) Naval Postgraduate School - USA



(b) Beihang University - China



(c) Nanjing University - China



(d) University of Twente - Netherlands

**Figure 1.2:** Some examples of developed Stewart platform with non-cubic geometry. (a), (b), (c), (d)

## 2 Effect of geometry on Stewart platform properties

- Remind that the choice of frames (independently of the physical geometry) impacts the obtained stiffness matrix (as it is defined as forces/motion evaluated at the chosen frame)
- Important:  $b_i$  (joint position w.r.t top platform) and  $s_i$  (orientation of struts)

For the nano-hexapod:

- Size requirements: Maximum height, maximum radius

### 2.1 Stiffness

- Give some examples:
  - struts further apart: higher angular stiffness, same linear stiffness
  - orientation: more vertical  $\Rightarrow$  increase vertical stiffness, decrease horizontal stiffness

### 2.2 Mobility and required joint and actuator stroke

- Comparison of the XYZ mobility (fixed orientation) for two geometry (or maybe only in the XY or YZ plane to see more clearly the differences)

[Estimated required actuator stroke from specified platform mobility](#) Will be useful to choose the actuators

[Estimation of the Joint required Stroke](#) Will be useful to design the flexible joints

## Conclusion

Table that summarize the findings [Optimal Nano-Hexapod Geometry](#)

# 3 The Cubic Architecture

Cubic configuration <file:///home/thomas/Cloud/work-projects/ID31-NASS/matlab/stewart-simscape/org/cubic-configuration.org>

## 3.1 The Cubic Architecture

From [21], 7 properties of cubic configuration:

1. Uniformity in control capability in all directions
  2. Uniformity in stiffness in all directions
  3. Minimum cross coupling force effect among actuators
  4. Facilitate collocated sensor-actuator control system design
  5. Simple kinematics relationships
  6. Simple dynamic analysis
  7. Simple mechanical design
- Principle
  - Examples of Stewart platform with Cubic architecture
  - Different options? Center of the cube above the top platform? Where to mention that ? With examples

## 3.2 Static Properties

Explain that we get diagonal K matrix =, static decoupling in the cartesian frame. Uniform mobility in X,Y,Z directions

## 3.3 Dynamical Properties?

[8]



**afzali-far16'vibrat'dynam'isotr'hexap'analy'studies:**

- proposes an architecture where the CoM can be above the top platform
- “**Dynamic isotropy**, leading to equal eigenfrequencies, is a powerful optimization measure.”
- Show examples where the dynamics can indeed be decoupled in the cartesian frame (i.e. decoupled K and M matrices)
- Better decoupling between the struts? not sure... Compute the coupling between the struts for a cubic and non-cubic architecture
- Same resonance frequencies for suspension modes? Maybe in one case: sphere at the CoM? Could be nice to show that. Say that this can be nice for optimal damping for instance (link to paper explaining that)

## 4 Conclusion

Inertia used for experiments will be very broad =, difficult to optimize the dynamics Specific geometry is not found to have a huge impact on performances. Practical implementation is important.

Geometry impacts the static and dynamical characteristics of the Stewart platform. Considering the design constrains, the slight change of geometry will not significantly impact the obtained results.

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