

# Mechatronics Approach for the Development of a Nano-Active-Stabilization-System

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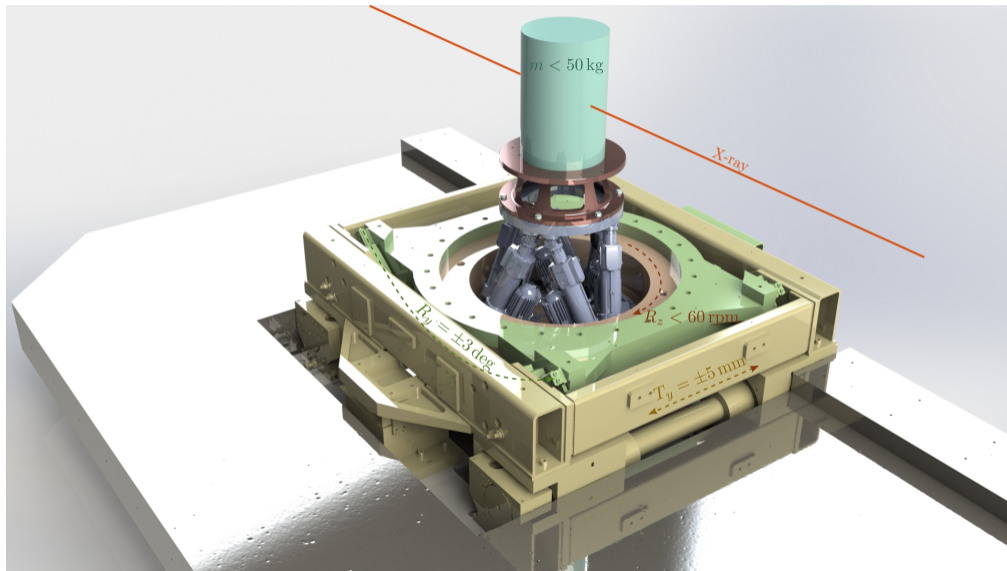
<sup>3</sup>BEAMS Department, Free University of Brussels, Belgium



**Precision  
Mechatronics  
Laboratory**



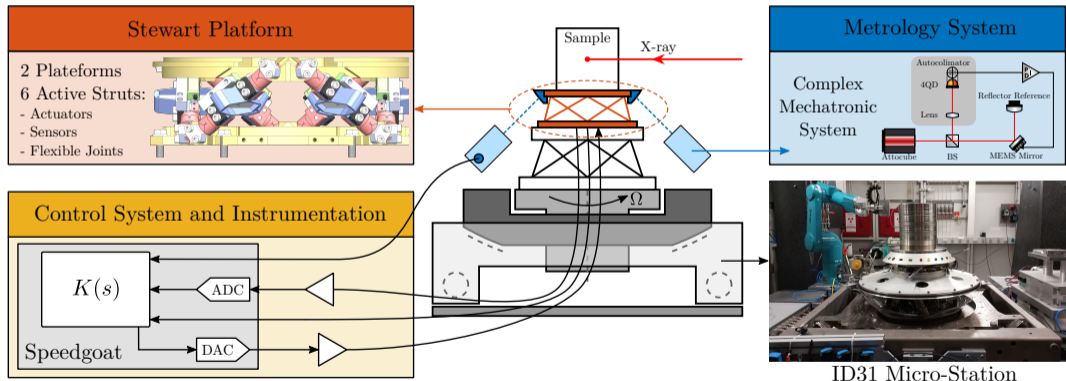
# The ID31 Micro Station



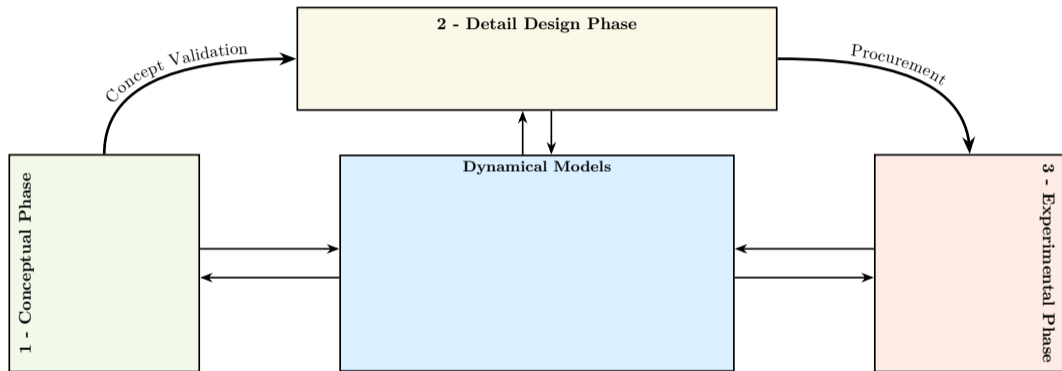
# Introduction - The Nano Active Stabilization System

**Objective:** Improve the position accuracy from  $\approx 10 \mu m$  down to  $\approx 10 nm$

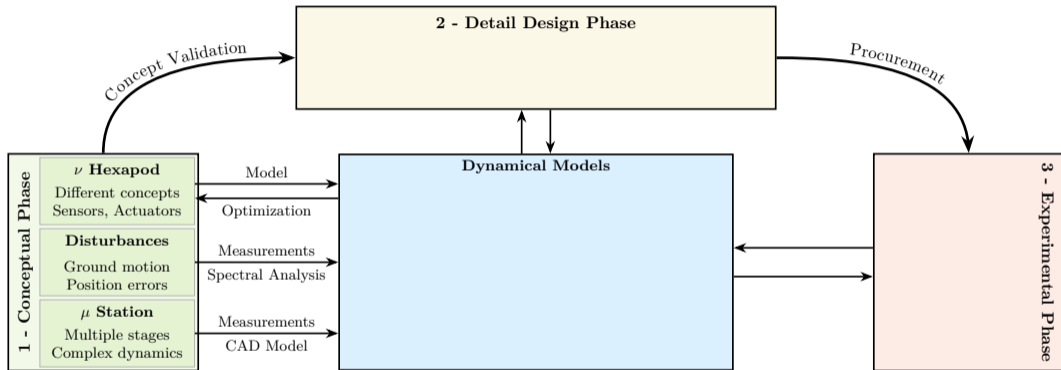
**Design approach:** “Model based design” / “Predictive Design”



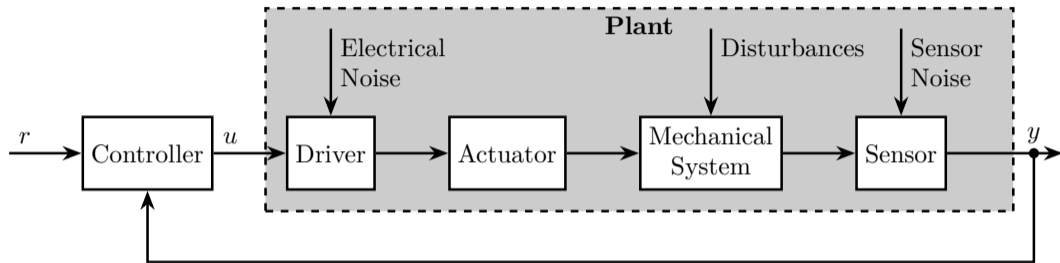
# Overview of the Mechatronic Approach - Model Based Design



# Outline - Conceptual Phase



# Feedback Control - The Control Loop



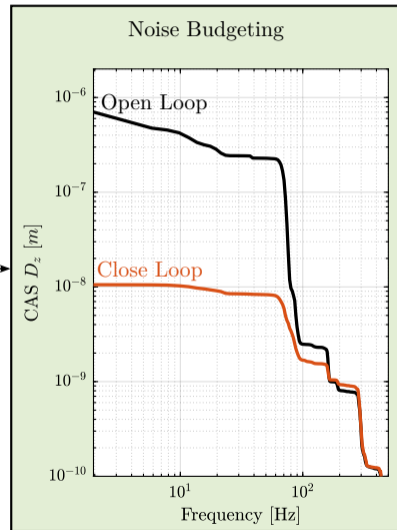
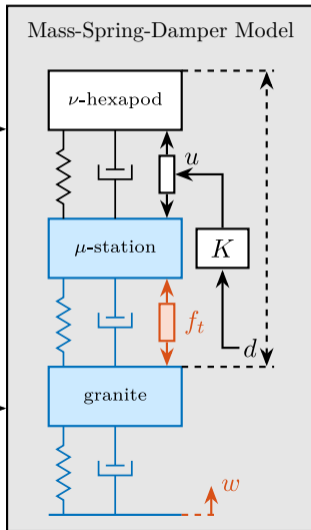
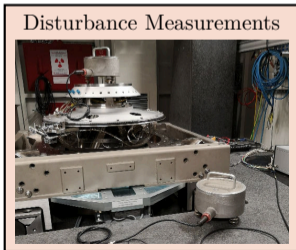
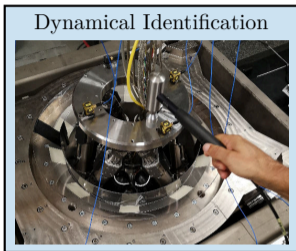
## Why Feedback?

- Model uncertainties
- Unknown disturbances

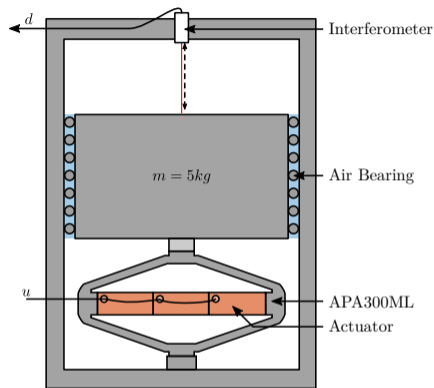
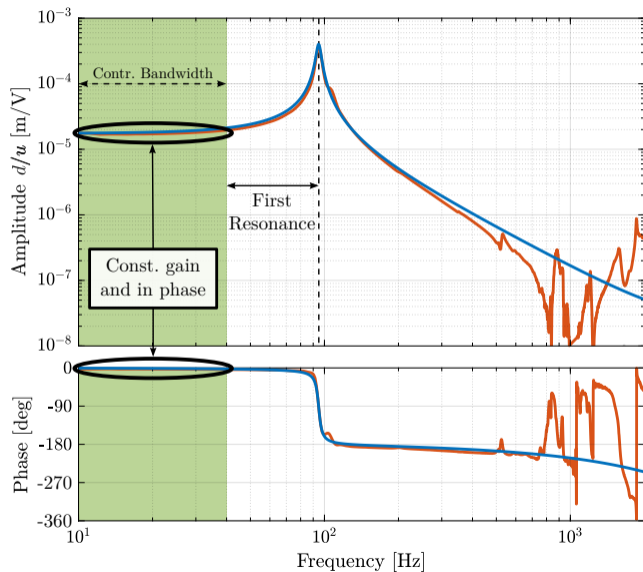
## Every elements can limit the performances

- Drivers, Actuators, Sensors
- Mechanical System
- Controller

# Noise Budgeting and Required Control Bandwidth



# Limitation of the Controller Bandwidth?

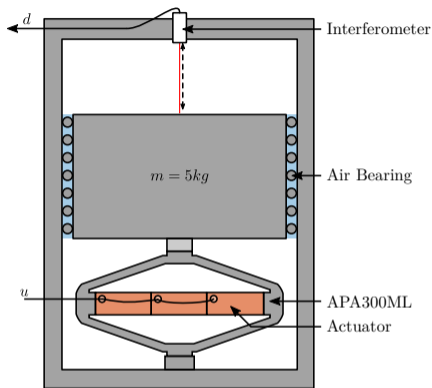
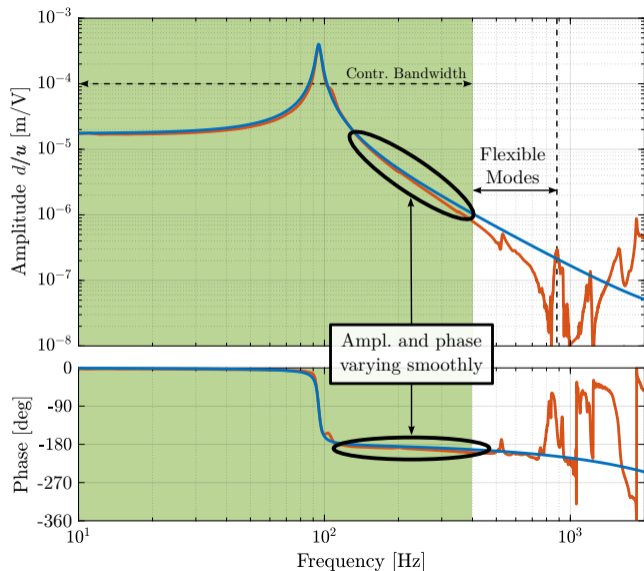


## Typical Approach

"As stiff as possible"  
Simple controller (e.g. PID)



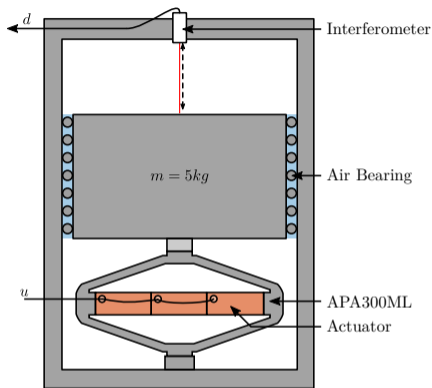
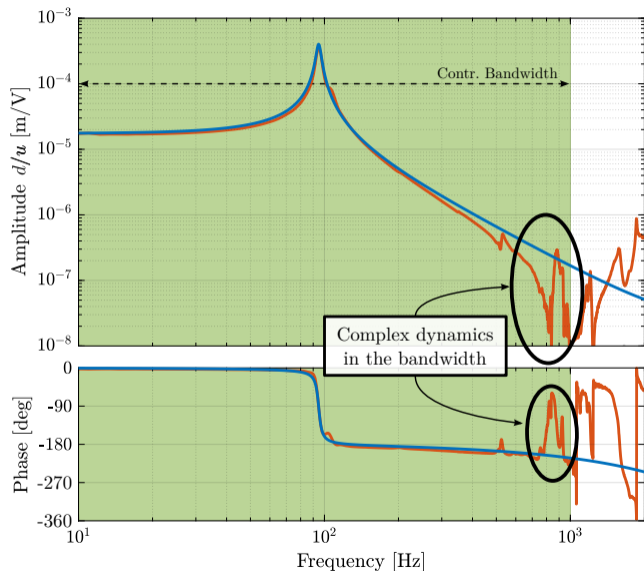
# Limitation of the Controller Bandwidth?



## Alternative Approach

Limited by complex dynamics  
Model based controller

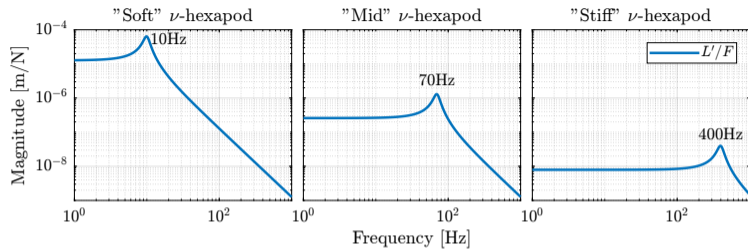
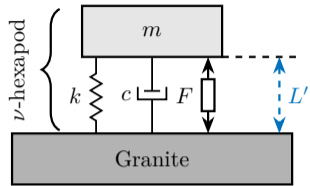
# Limitation of the Controller Bandwidth?



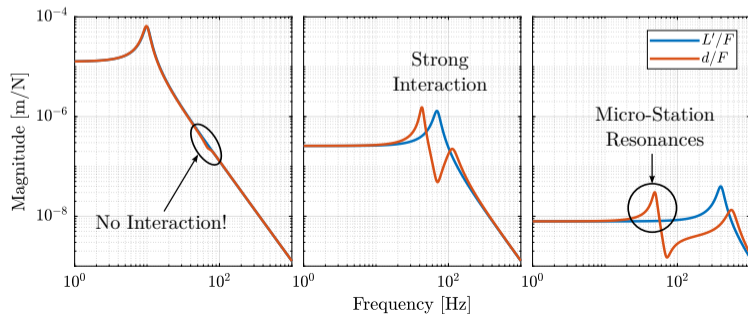
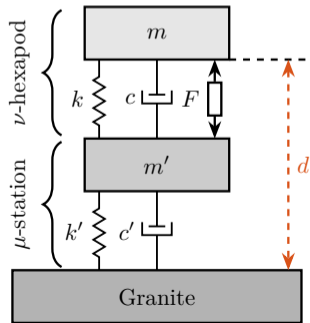
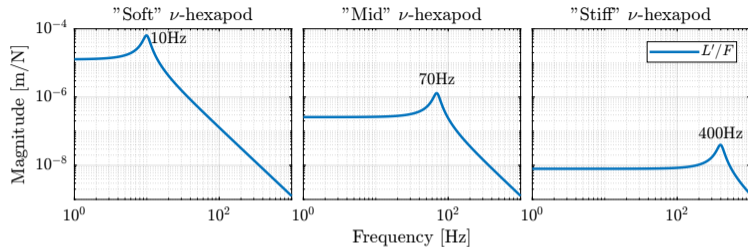
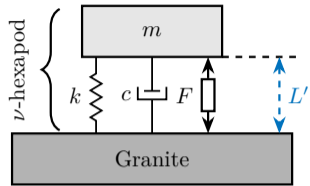
Next-Gen Systems

Active research topic  
Complex controllers

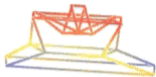
# Soft or Stiff $\nu$ -hexapod ? Interaction with the $\mu$ -station



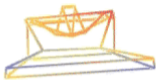
# Soft or Stiff $\nu$ -hexapod ? Interaction with the $\mu$ -station



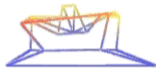
# Complexity of the Micro-Station Dynamics (Model Analysis)



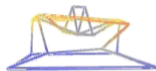
a) 11Hz



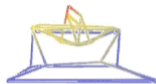
b) 18Hz



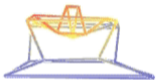
c) 37Hz



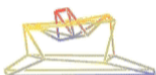
d) 40Hz



e) 54Hz



f) 56Hz



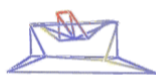
g) 70Hz



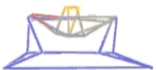
h) 72Hz



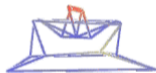
i) 72Hz



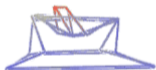
j) 85Hz



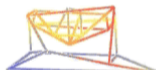
k) 90Hz



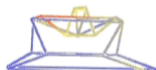
l) 91Hz



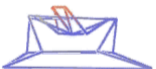
m) 96Hz



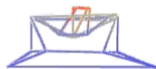
n) 105Hz



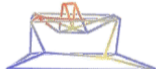
o) 107Hz



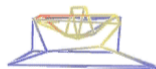
p) 113Hz



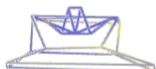
q) 117Hz



r) 124Hz

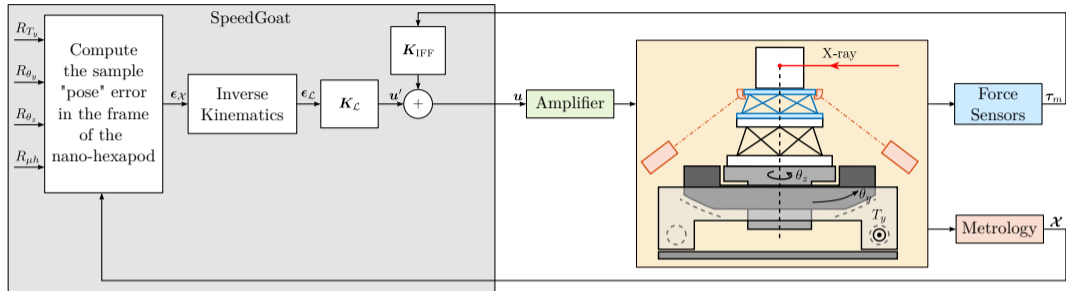


s) 145Hz



t) 150Hz

# Control Strategy: HAC-LAC



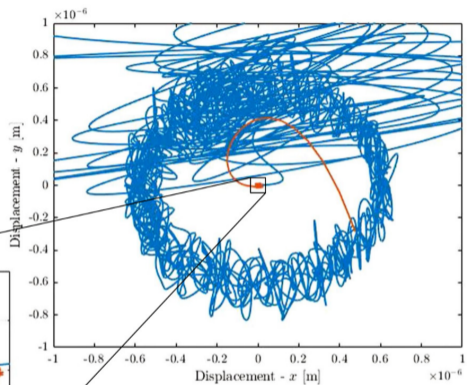
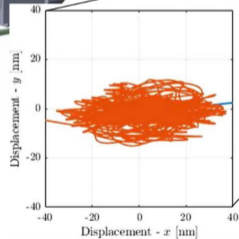
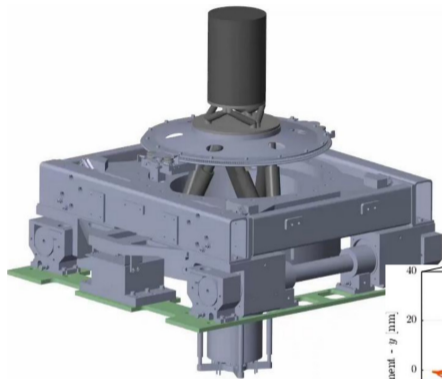
## Low Authority Control

- Collocated sensors/actuators
- Guaranteed Stability
- Adds damping
- ↘ vibration near resonances

## High Authority Control

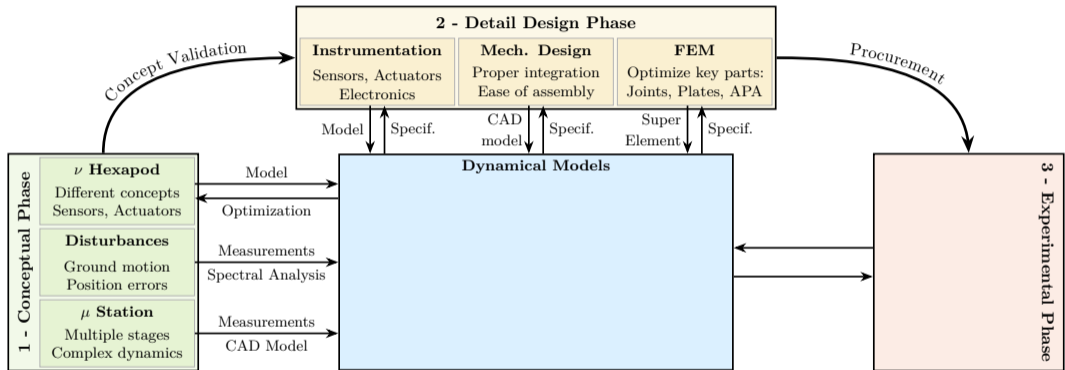
- Position sensors
- Complex dynamics
- ↘ vibration in the bandwidth
- Use transformation matrices

# Multi-Body Models - Simulations



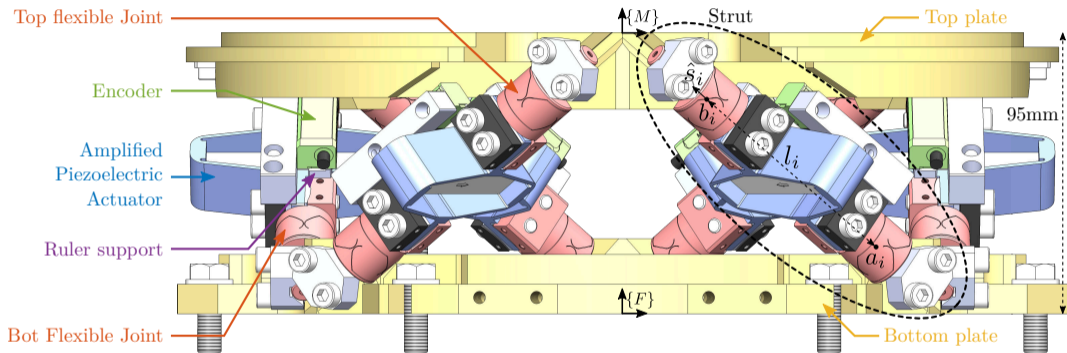
Validation of the concept

# Outline - Detail Design Phase

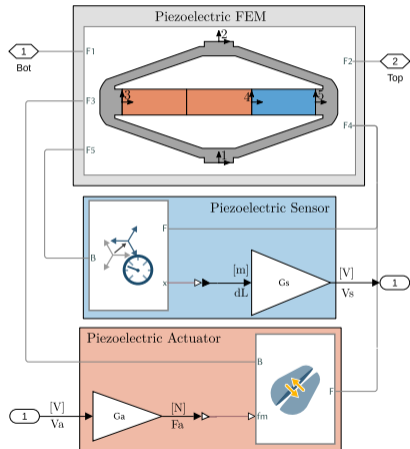
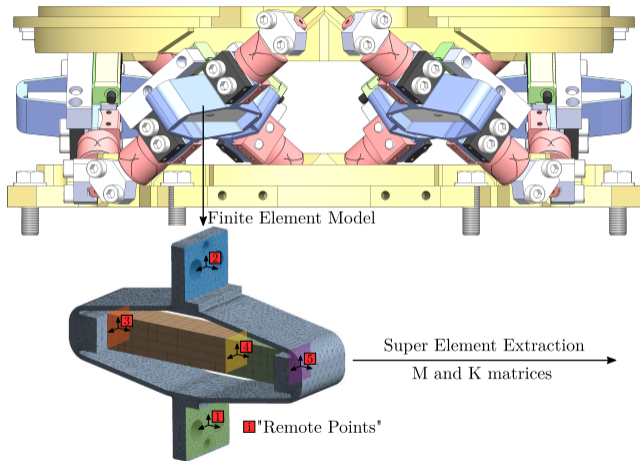




# Nano-Hexapod Overview - Key elements



# Include Flexible Elements in a Multi-Body model



# Choice of Actuator - Amplifier Piezoelectric Actuator

Characteristic	Specs	Doc.
Axial Stiff.	$\approx 1 \text{ N}/\mu\text{m}$	$1.8 \text{ N}/\mu\text{m}$
Sufficient Stroke	$>100 \mu\text{m}$	$368 \mu\text{m}$
Height	$<50 \text{ mm}$	$30 \text{ mm}$
High Resolution	$<5 \text{ nm}$	$3 \text{ nm}$

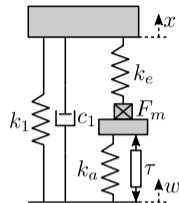


Fig.: 2-DoF Model

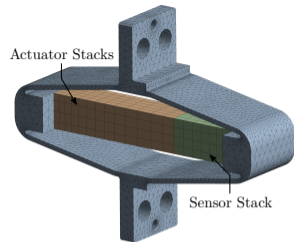


Fig.: APA Finite Element Model

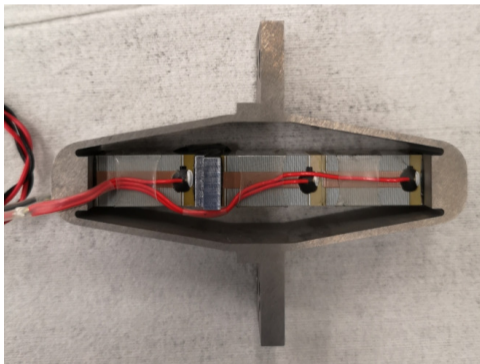


Fig.: Picture of the APA300ML

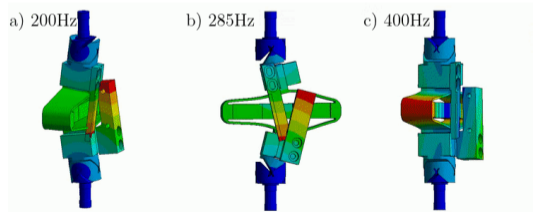


Fig.: Flexible Modes due to limited APA stiffness

# Flexible Joints - Specifications and Optimization (link)

Goal	Stiffness	Specs	FEM	Measured
High DVF Damping	Axial	$>100 \text{ N}/\mu\text{m}$	94	
Low Coupling	Bending	$<100 \text{ Nm}/\text{rad}$	5	3.8
Low Coupling	Torsion	$<500 \text{ Nm}/\text{rad}$	260	
Sufficient Stroke	Bending Stroke	$>1 \text{ mrad}$	20	18

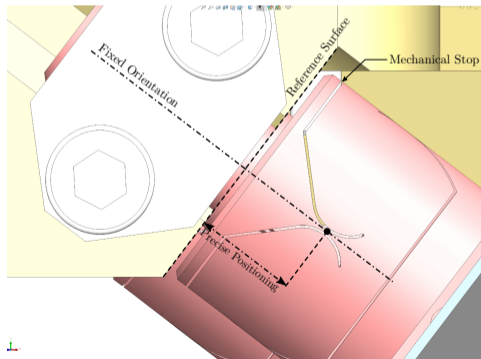
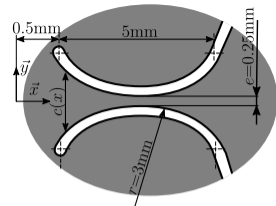


Fig.: Positioning of the top joint

Fig.: Dimensions after optimization

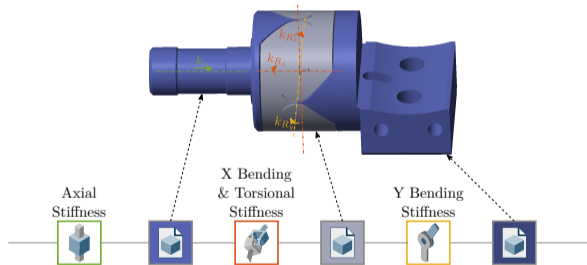
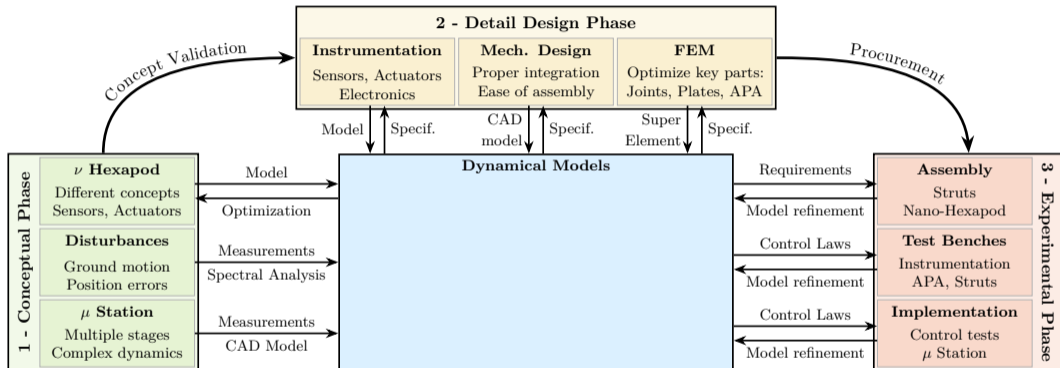


Fig.: Simscape Model

## Instrumentation

- PD200 amplifier
- Encoders
- Speedgoat, DAC, ADC
- PEPU
- Attocube

# Outline - Experimental Phase



# Flexible Joints - Measurements

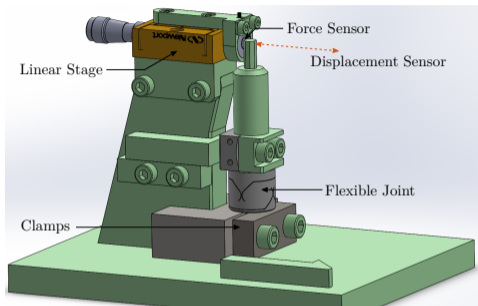


Fig.: Measurement bench

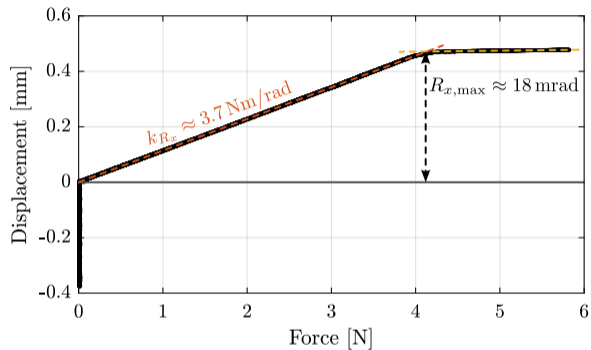
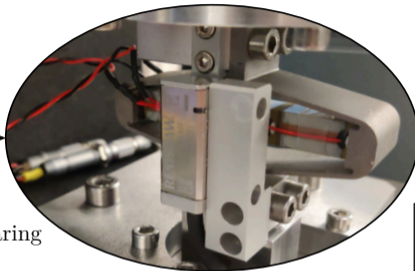
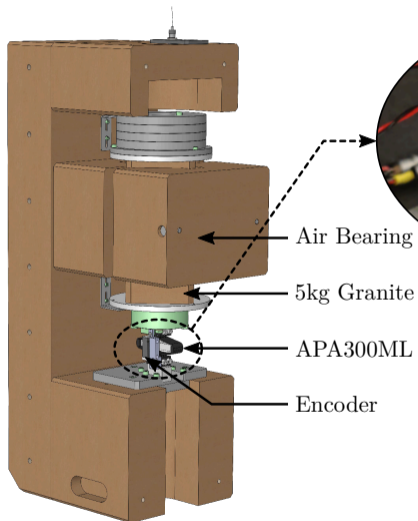


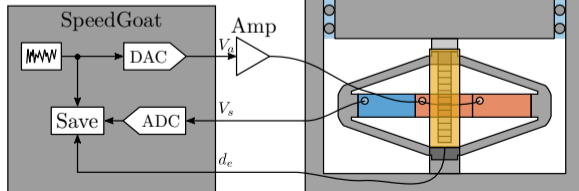
Fig.: Measured displacement and force

# Amplified Piezoelectric Actuator - Test Bench



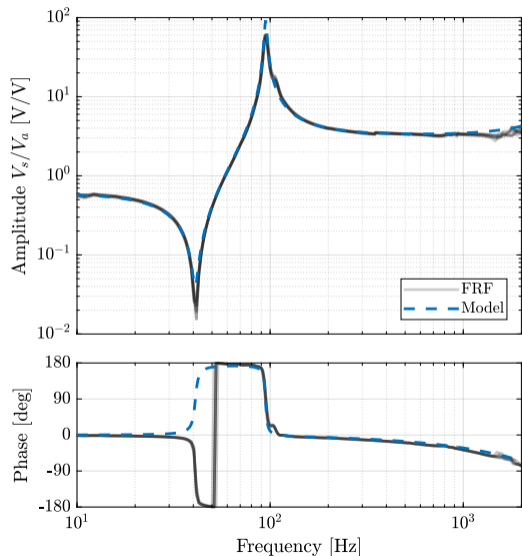
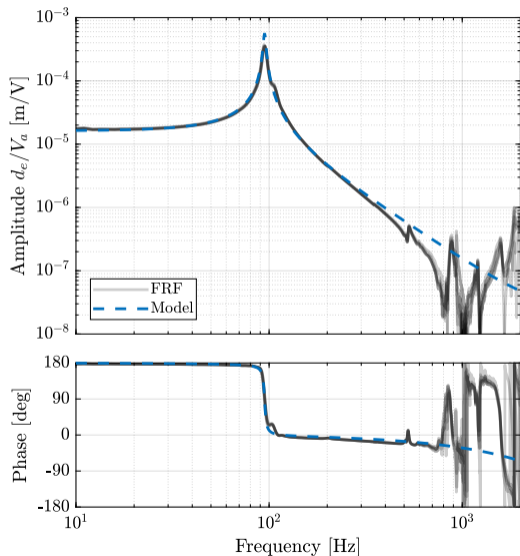
## Goals

- Identify Dynamics
- Tune APA Model
- Test IFF

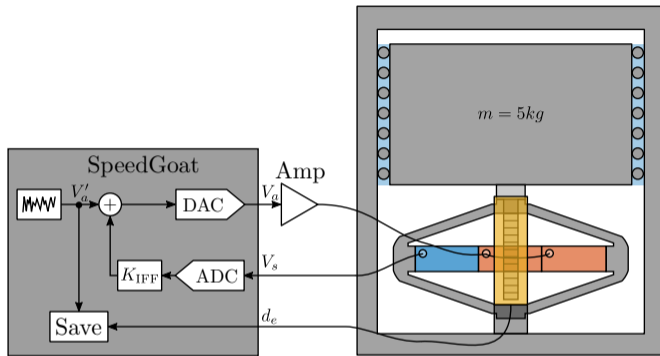




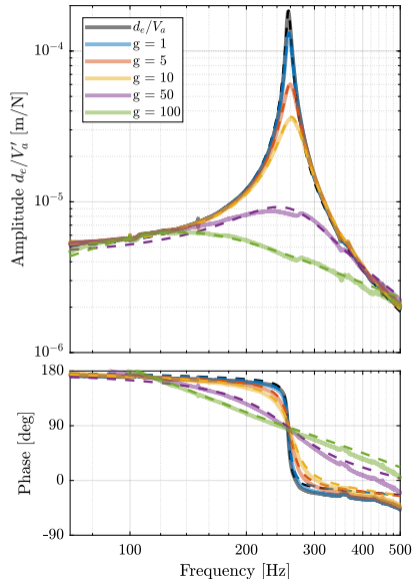
# Amplified Piezoelectric Actuator - Extracted Model



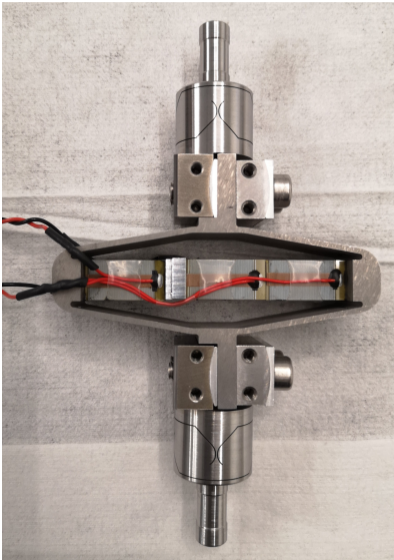
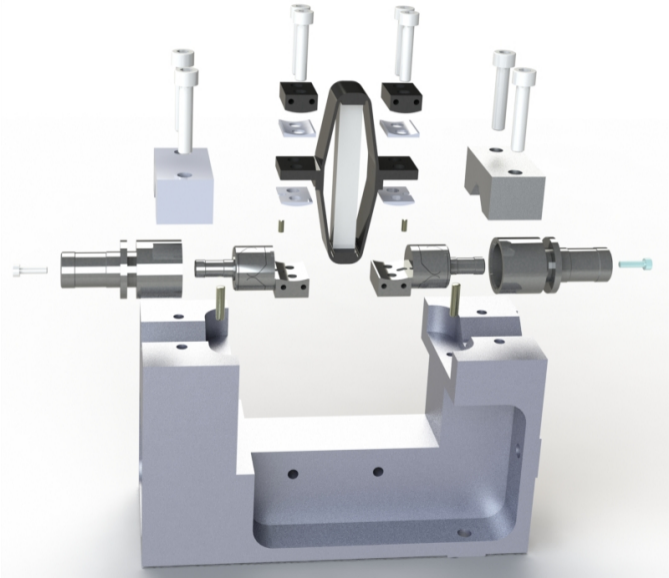
# Amplified Piezoelectric Actuator - Integral Force Feedback



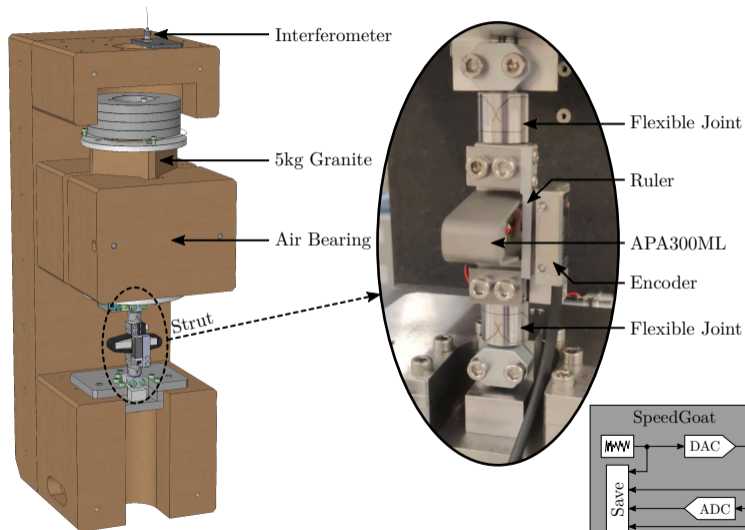
$$K_{1FF}(s) = \frac{g}{s}$$



# Strut - Mounting Tool

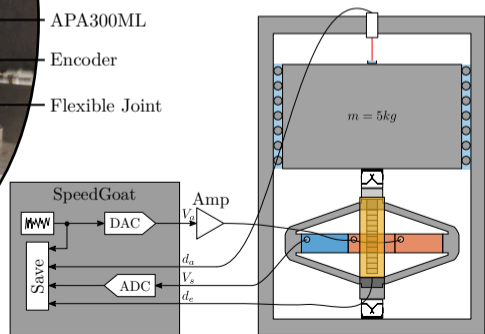


# Strut - Dynamical Measurements

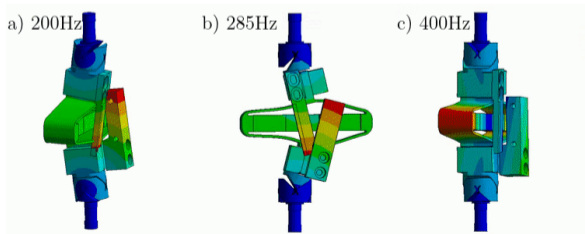
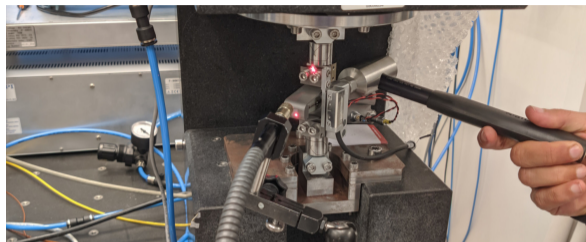
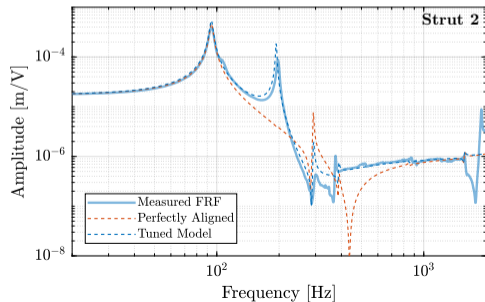
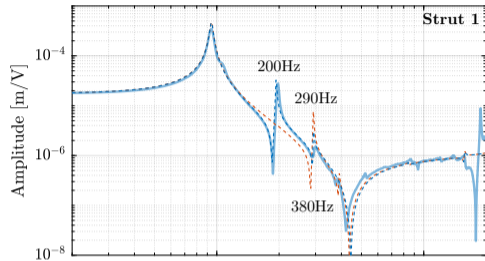


## Goals

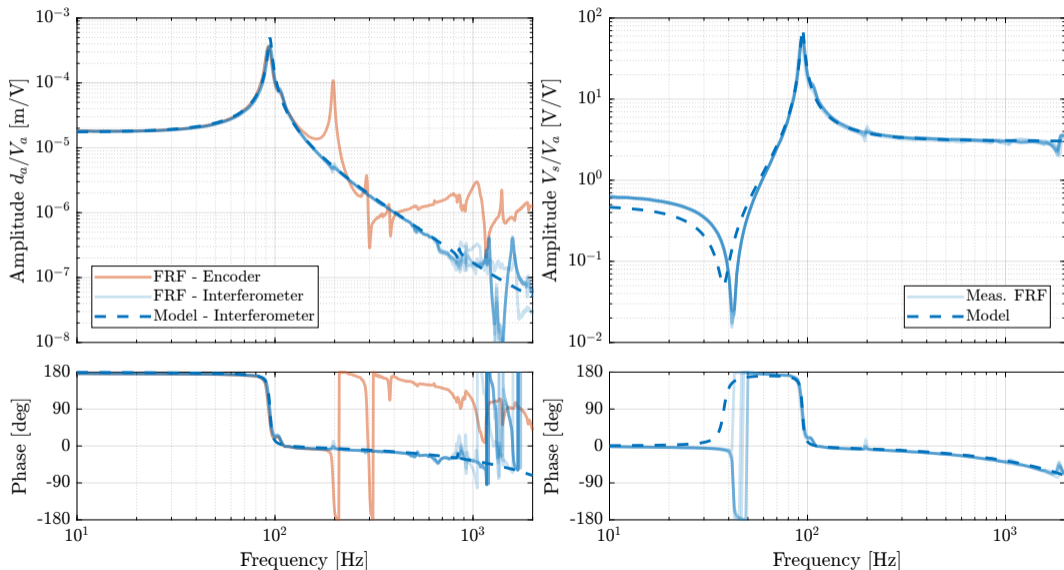
- Identify Dynamics
- Tune Model
- Flexible joints effects
- Encoder effect



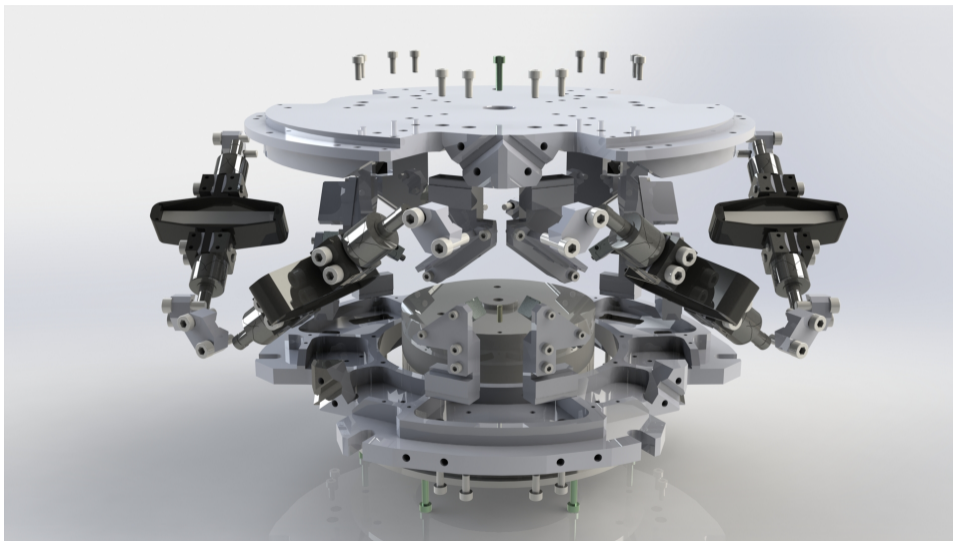
# Strut - Encoders Output and Spurious Modes



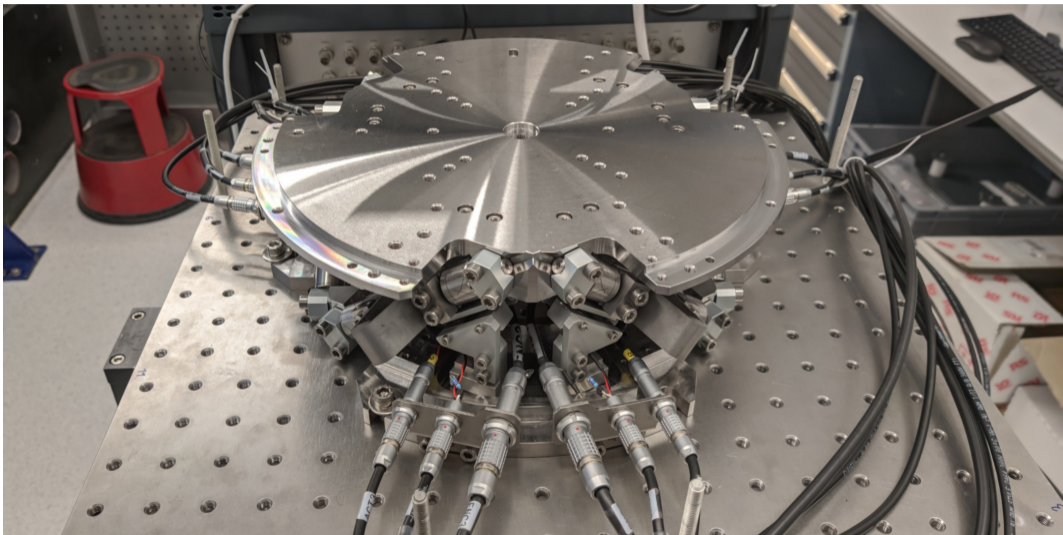
# Strut - Extracted Model



# Nano-Hexapod Mounting Tool



## Mounted Nano-Hexapod





## Nano-Hexapod - Identified Dynamics

Diagonal + off-diagonal transfer function from  $V_a$  to  $D_e$  (comp with model)

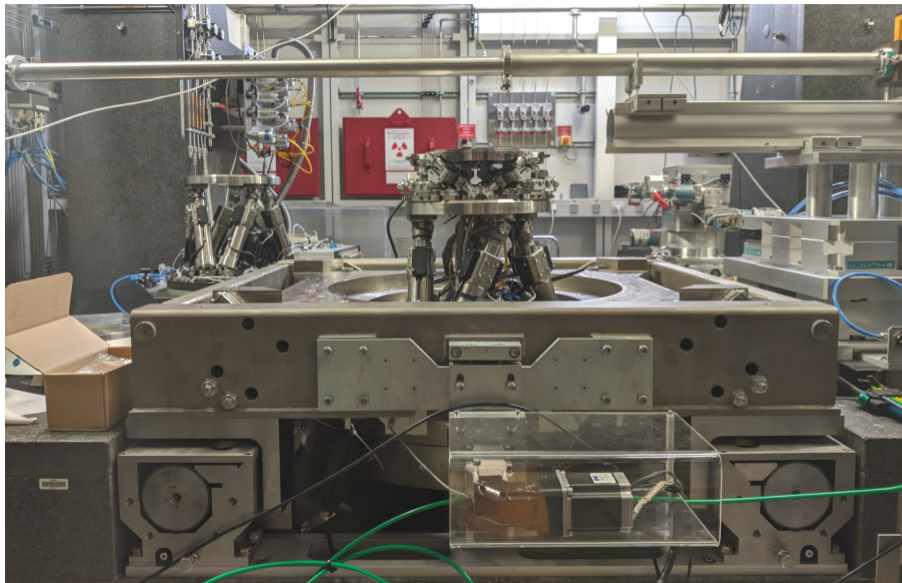
## Nano-Hexapod - Force Sensors

Diagonal + off-diagonal transfer function from  $V_a$  to  $V_s$

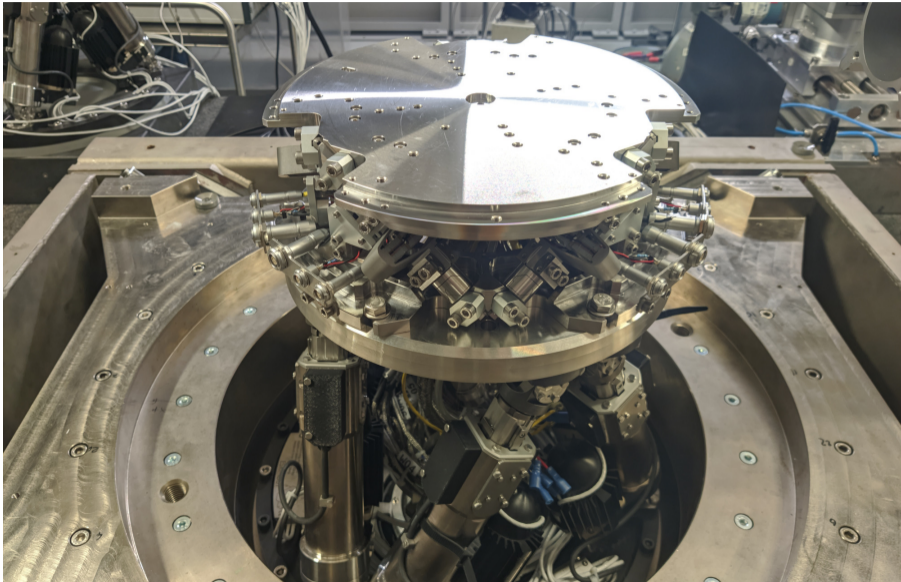
## Nano-Hexapod - Damped Dynamics

Damped and Undamped, Diagonal + off-diagonal transfer function from  $V_a$  to  $D_e$

## The Nano-Hexapod on top of the Micro-Station



## The Nano-Hexapod on top of the Micro-Station



# Conclusion